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PUMP CHARACTERISTICS FOR A  
COMPUTERIZED PUMP TRANSIENT ANALYSIS

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A Thesis submitted to the Faculty and the Board of Trustees  
of the Colorado School of Mines in partial fulfillment of the  
requirements for the degree of Master of Science (Mathematics).

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## ABSTRACT

A method is presented for approximating a centrifugal pump's characteristic by an interpolation procedure when its true characteristic is not known. The characteristic is then represented by 12 sixth degree polynomials for use in a transient analysis computer program. Data used for approximating a centrifugal pump's characteristic is shown in tabular form in Tables 1 through 12. The data was obtained from the known characteristics of 7 different specific speed pumps.

The transient analysis of a water conveyance system containing centrifugal pumps requires knowledge of each pump's characteristic and because a pump's characteristic is not generally known, it must be approximated. The characteristic can be critical in the transient analysis because changing the characteristic used to analyze the system will change the resulting head and discharge history of the system during a transient. The characteristic used may thus play a major role in determining such things as pipe thickness, surge tank size, air chamber size and valve closure times. Choosing a characteristic which approximated that of each pump in the system as closely as possible can be critical in determining the most economical design of the system.

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## INTRODUCTION

The design of water conveyance systems whose main components are pipelines and centrifugal pumps and whose size and discharge can vary over a wide range requires knowledge of all the systems' components. The design of a water conveyance system must be analyzed for both steady and transient flow. In analyzing a system that contains a centrifugal pump, one of the difficulties is defining the characteristic to describe the pump's operation. Each pump has a unique characteristic which, when known along with the head across the pump ( $h$ ), the speed of the pump ( $\alpha$ ) and the torque on the pump impeller ( $\beta$ ), will enable the calculation of the discharge ( $q$ ). A pump characteristic thus relates the four variables involved in pump operation ( $h, \alpha, \beta, q$ ). The characteristic of a pump is determined by testing the actual pump, or approximated by testing a model of the pump.

This thesis presents a method for approximating a given pump's characteristic when it is not known. Also, a procedure for storing the characteristic (known or approximated) in the form of 12 sixth degree polynomial equations is suggested. The 12 equations can then be used in a computer program to determine transient conditions in a water conveyance system containing a known pump. A pump's characteristic is broken down into the following 3 areas of pump operation for ease of representation.

1. The normal pumping zone, where speed and pump discharge are both positive.

2. The energy dissipation zone, where discharge has reversed (negative), but the pump is still rotating in a normal direction.
3. The turbine zone, where the discharge is negative, and the pump is rotating in the reverse direction.

When power failure of a pumping plant is a possibility, the consequences must be analyzed. To do so requires knowledge of the pump's  $WR^2$  (rotating weight or inertia) and the characteristics of the pump. In the event of a power failure, there is no more energy supplied to keep the water moving, and the downstream head is higher than the upstream head. This situation is just the opposite of when power is being supplied to the pump. Such a condition causes the water in the pipeline to slow down and finally reverse. The reverse flow will continue in the line and in the pump until a valve is shut, stopping the flow. The slowing down and reversing of the water in the pipeline is of interest because there is an immediate downsurge in the line followed by an upsurge or headrise. The magnitude of these changes depends on the pump's characteristics and  $WR^2$ . In general, the larger the value  $WR^2$  is of a given pump, the smaller the downsurge and subsequent upsurge, because the increased inertia of the pump supplies the water flowing in the pipeline with energy so that changes in velocity are slower, resulting in less variation in head (pressure). If the transient analysis of a pumping system is not done properly, the result can vary from the extreme of broken lines and damaged pumping plants to the extreme of a highly overdesigned and expensive

system. Although this thesis is directed to closed water conveyance systems containing centrifugal pumps, the design of oil and gas pipelines containing pumps may be dependent on the pump characteristic used in their transient analysis.

#### A PRESENT METHOD OF ANALYSIS

When a pump has not been tested, its characteristic has not been defined, and it is usually approximated by using the known characteristic of another pump. The known characteristic can be chosen from one of the 7 for which data is tabulated in Tables 1 through 12. The method of picking which of the 7 known characteristics to use is usually based on the assumption that any two geometrically similar pumps of the same specific speed ( $N_s$ , a type number derived from the rated head, discharge and speed of a pump) will have similar performance characteristics.

Thus, when analyzing a conveyance system containing a centrifugal pump whose characteristic is not known, its specific speed is determined, and a known characteristic from a pump of similar specific speed is used in the analysis. A problem with this procedure is that there are not enough known pump characteristics to adequately cover the range of specific speeds encountered in centrifugal pumps. Thus a known characteristic from a pump which has a specific speed closest to that of the actual pump in the system is used in the transient analysis for the system. When using a computer to analyze a pump power failure, the known characteristic is usually represented by tabulated values of  $h$ ,  $q$ ,  $\alpha$ , and  $\beta$  taken from the actual test of the pump. The computer program will use the tabulated values as a

basis for interpolating specific values to be used during each time increment of the analysis.

#### PROPOSED METHOD

If the additional assumption that specific speed varies uniformly and continuously with pump geometry is made(Ref. 1), there is a uniformly continuous relation between specific speed and the corresponding pump's characteristic. Using the 7 known characteristics of varying specific speed shown in this paper, and fitting a 2nd or 3rd degree polynomial to them using the method of least squares, the characteristic for any specific speed pump can be approximated. The use of a 2nd or 3rd degree polynomial is arbitrary, and the one chosen for each case was done after observing a plot of the portion of the characteristic in question against specific speed. A function other than a 2nd or 3rd degree polynomial could be used if it represented the plotted data. This procedure thus uses a relation determined by using all 7 characteristics to approximate that of the pump in the system instead of picking the one known characteristic that has its specific speed closest to the specific speed of the pump actually in the system.

Once data representing the approximate characteristic is determined, it can be put in the form of 12 sixth degree polynomials which are then used in a pump transient analysis computer program to represent the pump operation.

#### PUMP TRANSIENT ANALYSIS

The method commonly used in the computerized analysis of pipeline transients incorporates the method of characteristics to

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represent the pipeline and a table containing values of the pump's characteristics (Ref. 2). The table of the pump's characteristics contains values of  $h$ ,  $q$ ,  $\alpha$ , and  $\beta$  which represent the actual pump operation. In the transient analysis, values for all pump conditions are interpolated from the table so that at each instant of time during the transient, the head and discharge anywhere in the system are known, as well as the pump speed and torque.

Actual pump characteristic data can be put in the form of curves (Ref. 3 and 10) using the following homologous relations:

$$\begin{array}{ll} h/q^2 \text{ vs } \alpha/q & h/\alpha^2 \text{ vs } q/\alpha \\ \beta/q^2 \text{ vs } \alpha/q & \beta/\alpha^2 \text{ vs } q/\alpha \end{array}$$

The four relations above can be graphed to represent each of the three pump zones: normal, energy dissipation, and turbine. The result is 12 curves. A sixth degree polynomial equation was chosen to represent each of the curves by taking 7 points from each curve and solving for the coefficients of the equations.

Data points representing the pump characteristic for any given specific speed are first found by a least squares fit procedure on data from 7 known characteristics of different specific speed pumps. The data from the 7 known characteristics and the 2nd or 3rd degree polynomial resulting from the least squares fitting procedure are shown in Tables 1 through 12. The portion of the pump characteristic in each table is shown on the next page.

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Table 1	Normal pump zone, ( $0 \leq \alpha/q \leq 1$ ), $h/q^2$ vs $\alpha/q$
Table 2	Normal pump zone, ( $0 \leq \alpha/q \leq 1$ ), $B/q^2$ vs $\alpha/q$
Table 3	Normal pump zone, ( $0 \leq q/\alpha \leq 1$ ), $h/\alpha^2$ vs $q/\alpha$
Table 4	Normal pump zone, ( $0 \leq q/\alpha \leq 1$ ), $B/\alpha^2$ vs $q/\alpha$
Table 5	Energy dissipation zone, ( $-1 \leq \alpha/q \leq 0$ ), $h/q^2$ vs $\alpha/q$
Table 6	Energy dissipation zone, ( $-1 \leq \alpha/q \leq 0$ ), $B/q^2$ vs $\alpha/q$
Table 7	Energy dissipation zone, ( $-1 \leq q/\alpha \leq 0$ ), $h/\alpha^2$ vs $q/\alpha$
Table 8	Energy dissipation zone, ( $-1 \leq q/\alpha \leq 0$ ), $B/\alpha^2$ vs $q/\alpha$
Table 9	Turbine zone, ( $0 \leq \alpha/q \leq 1$ ), $h/q^2$ vs $\alpha/q$
Table 10	Turbine zone, ( $0 \leq \alpha/q \leq 1$ ), $B/q^2$ vs $\alpha/q$
Table 11	Turbine zone, ( $0 \leq q/\alpha \leq 1$ ), $h/\alpha^2$ vs $q/\alpha$
Table 12	Turbine zone, ( $0 \leq q/\alpha \leq 1$ ), $B/\alpha^2$ vs $q/\alpha$

Data was taken from the following 7 known characteristics:

Pump Specific Speed	Reference
1276	3
1700	4
1940	5
3700	6
4600	7
7600	3
13,500	3

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Some data is missing from the tables because the original tests were incomplete. The sixth degree polynomial, when used to approximate an actual characteristic, has the value of the ordinate within 0.01 of the actual curve representing the test of a pump. This is an adequate representation, because actual testing does not give accuracy with magnitude 0.01.

To determine the characteristic of any specific speed pump, a relation is derived using a least squares fitting procedure for each of  $h/\alpha^2$ ,  $h/q^2$ ,  $\beta/\alpha^2$ , versus  $N_s$  at the seven values of  $\alpha/q$  and  $q/\alpha$ . For instance, the data from Table 3 at  $q/\alpha = 0$  are plotted in Figure 1, and a second degree polynomial is fit through the points. One point on a characteristic is defined in Figure 1 (i.e.,  $q/\alpha = 0$ ,  $h/\alpha^2 = f(N_s)$ ). The point in this instance is determined by identifying the specific speed of the actual pump in the system. The use of a 2nd degree polynomial as the function to describe the variation of  $h/\alpha^2$  is arbitrary, and could be any type of function which will adequately represent the relation. Second or third degree polynomials were used here, and the choice was made on the basis of observation and experience. There are 7 equations for each table. Each equation along with the corresponding value of  $q/\alpha$  or  $\alpha/q$  represents a point on the characteristic whose specific speed is  $N_s$ . As other known characteristics are added to the data, these relations should be reviewed.

To approximate the unknown characteristic of a centrifugal pump whose specific speed is known, one must first solve the 7 equations under each table. This gives 7 points for each of the 12 areas into which the characteristic is subdivided. Because intermediate values are required during a pump transient analysis, a 6th degree polynomial is derived from the 7 points in each table. These 12 sixth degree

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polynomials then describe an approximate characteristic of a pump given the specific speed. They are stored in the computer program in the following form:

Pump zone ( $0 \leq \alpha/q \leq 1$ )

$$h/q^2 = A_1(\alpha/q)^6 + B_1(\alpha/q)^5 + C_1(\alpha/q)^4 + D_1(\alpha/q)^3 + E_1(\alpha/q)^2 + F_1(\alpha/q) + G_1$$

$$B/q^2 = A_2(\alpha/q)^6 + B_2(\alpha/q)^5 + C_2(\alpha/q)^4 + D_2(\alpha/q)^3 + E_2(\alpha/q)^2 + F_2(\alpha/q) + G_2$$

$$(0 \leq q/\alpha \leq 1)$$

$$h/\alpha^2 = A_3(q/\alpha)^6 + B_3(q/\alpha)^5 + C_3(q/\alpha)^4 + D_3(q/\alpha)^3 + E_3(q/\alpha)^2 + F_3(q/\alpha) + G_3$$

$$B/\alpha^2 = A_4(q/\alpha)^6 + B_4(q/\alpha)^5 + C_4(q/\alpha)^4 + D_4(q/\alpha)^3 + E_4(q/\alpha)^2 + F_4(q/\alpha) + G_4$$

Energy dissipation zone ( $-1 \leq \alpha/q \leq 0$ )

$$h/q^2 = A_5(\alpha/q)^6 + B_5(\alpha/q)^5 + C_5(\alpha/q)^4 + D_5(\alpha/q)^3 + E_5(\alpha/q)^2 + F_5(\alpha/q) + G_5$$

$$B/q^2 = A_6(\alpha/q)^6 + B_6(\alpha/q)^5 + C_6(\alpha/q)^4 + D_6(\alpha/q)^3 + E_6(\alpha/q)^2 + F_6(\alpha/q) + G_6$$

$$(-1 \leq q/\alpha \leq 0)$$

$$h/\alpha^2 = A_7(q/\alpha)^6 + B_7(q/\alpha)^5 + C_7(q/\alpha)^4 + D_7(q/\alpha)^3 + E_7(q/\alpha)^2 + F_7(q/\alpha) + G_7$$

$$B/\alpha^2 = A_8(q/\alpha)^6 + B_8(q/\alpha)^5 + C_8(q/\alpha)^4 + D_8(q/\alpha)^3 + E_8(q/\alpha)^2 + F_8(q/\alpha) + G_8$$

Turbine zone ( $0 \leq \alpha/q \leq 1$ )

$$h/q^2 = A_9(\alpha/q)^6 + B_9(\alpha/q)^5 + C_9(\alpha/q)^4 + D_9(\alpha/q)^3 + E_9(\alpha/q)^2 + F_9(\alpha/q) + G_9$$

$$B/q^2 = A_{10}(\alpha/q)^6 + B_{10}(\alpha/q)^5 + C_{10}(\alpha/q)^4 + D_{10}(\alpha/q)^3 + E_{10}(\alpha/q)^2 + F_{10}(\alpha/q) + G_{10}$$

$$(0 \leq q/\alpha \leq 1)$$

$$h/\alpha^2 = A_{11}(q/\alpha)^6 + B_{11}(q/\alpha)^5 + C_{11}(q/\alpha)^4 + D_{11}(q/\alpha)^3 + E_{11}(q/\alpha)^2 + F_{11}(q/\alpha) + G_{11}$$

$$B/\alpha^2 = A_{12}(q/\alpha)^6 + B_{12}(q/\alpha)^5 + C_{12}(q/\alpha)^4 + D_{12}(q/\alpha)^3 + E_{12}(q/\alpha)^2 + F_{12}(q/\alpha) + G_{12}$$

The coefficients for each 6th degree equation can be determined in the program by solving seven equations in seven unknowns.

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## EXAMPLE

For purposes of illustration, consider the pump zone, ( $0 \leq q/\alpha \leq 1$ ) and the relation  $h/\alpha^2 = f(q/\alpha)$ . Assume a pump in the system has a rated head, discharge, and speed such that its  $N_s = 6000$ . The value of  $N_s$  would be determined in the computer program from the input of the pump's rated head, discharge, and speed. The seven equations in Table 3 would then be solved with  $N_s = 6000$  and would result in the following seven points: (Ref. Table 3)

<u><math>q/\alpha</math></u>	<u><math>h/\alpha^2</math></u>
0	1.768
0.17	1.598
0.33	1.375
0.50	1.190
0.67	1.104
0.83	1.052
1.00	1.000

A sixth degree polynomial is then fit through these seven points, resulting in the following equation:

$$h/\alpha^2 = 11.886(q/\alpha)^6 - 32.051(q/\alpha)^5 + 28.001(q/\alpha)^4 - 6.466(q/\alpha)^3 - 1.461(q/\alpha)^2 - 0.677(q/\alpha) + 1.768$$

The other 11 equations representing the rest of the pump's characteristics are developed in a similar way.

### SUMMARY OF METHOD

The computer program for pump transients will contain 7 equations for each of the 12 areas of pump operation. The equations are shown under each of the 12 tables. For the pump being analyzed, the data read into the program allows the calculation of specific speed  $N_s = f(Q_R, H_R, N_R)$ . Specific speed is the independent variable of the 7 equations shown under each table. When the equations are solved in the program, 7 points for each of the 12 areas of pump operation are determined. A 6th degree polynomial is then determined for each of these 12 areas using the 7 points from each area, and these equations are used in the transient analysis of the system containing the pump.

### RESULTS AND CONCLUSIONS

As more characteristics become available the equations under each table should be re-evaluated to incorporate the additional data points. A least squares procedure was used to fit an equation to the points because there was some scatter (Fig. 1). The scatter arises because given a specific speed, a limited range of pumps can be designed (Ref. 8). If, however, when designing a pump, the criteria were used that given the  $N_s$ , the most efficient pump be designed, the relation between the pump's characteristic and  $N_s$  as shown in Figure 1 would have no scatter of points (Ref. 9). Because criteria other than designing the most efficient pump must be taken into consideration, there will be scatter. The method proposed here, as well as any other method of approximating a pump's characteristic, should be used only when the actual or model characteristics are not available.

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#### ACKNOWLEDGEMENTS

This paper is based on Reference 10 with additional data added to make it more usable for the engineer involved in determining pump transients.

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Table 1

Normal Pump Zone

 $q \geq 0, \alpha \geq 0 \quad (0 \leq \alpha/q \leq 1)$ 

		$\alpha/q$						
		0	0.17	0.33	0.50	0.67	0.83	1.00
$h/q^2$	$N_s$							
	1276	-0.53	-0.42	-0.22	-0.04	0.23	0.56	1.00
	1700	-	-	-	-0.05	0.26	0.63	1.00
	1940	-	-	-	-	0.27	0.60	1.00
	3700	-	-	-	-	0.09	0.57	1.00
	4600	-	-	-	-	0.19	0.60	1.00
	7600	-1.56	-1.13	-0.69	-0.35	0.05	0.52	1.00
	13500	-1.00	-0.90	-0.76	-0.50	-0.11	0.37	1.00

$\alpha/q$	Equation, $h/q^2 = f(N_s)$
0	$h/q^2 = 2.1089 \times 10^{-8} N_s^2 - 3.5005 \times 10^{-4} N_s - 0.1177$
0.17	$h/q^2 = 1.2374 \times 10^{-8} N_s^2 - 2.2210 \times 10^{-4} N_s - 0.1567$
0.33	$h/q^2 = 0.5109 \times 10^{-8} N_s^2 - 1.1967 \times 10^{-4} N_s - 0.0756$
0.50	$h/q^2 = 0.1995 \times 10^{-8} N_s^2 - 0.6776 \times 10^{-4} N_s + 0.0590$
0.67	$h/q^2 = 0.0532 \times 10^{-8} N_s^2 - 0.3772 \times 10^{-4} N_s + 0.3034$
0.83	$h/q^2 = -0.1492 \times 10^{-8} N_s^2 + 0.0338 \times 10^{-4} N_s + 0.5938$
1.00	$h/q^2 = 1.0000$

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Table 2  
 Normal Pump Zone  
 $q \geq 0, \alpha \geq 0$  ( $0 \leq \alpha/q \leq 1$ )

		$\alpha/q$						
Ns		0	0.17	0.33	0.50	0.67	0.83	1.00
$B/q^2$	1276	-0.30	-0.16	0.01	0.20	0.43	0.70	1.00
	1700	-	-	-	-	0.60	0.77	1.00
	1940	-	-	-	-	-	-	1.00
	3700	-	-	-	-	0.36	0.70	1.00
	4600	-	-	-	-	0.32	0.72	1.00
	7600	-1.56	-0.93	-0.47	-0.11	0.29	0.67	1.00
	13500	-0.56	-0.62	-0.60	-0.39	0.03	0.51	1.00

$\alpha/q$	Equation, $B/q^2 = f(Ns)$
0	$B/q^2 = 3.0165 \times 10^{-8} Ns^2 - 4.6698 \times 10^{-4} Ns + 0.2468$
0.17	$B/q^2 = 1.4259 \times 10^{-8} Ns^2 - 2.4332 \times 10^{-4} Ns + 0.1336$
0.33	$B/q^2 = 0.4407 \times 10^{-8} Ns^2 - 1.1502 \times 10^{-4} Ns + 0.1496$
0.50	$B/q^2 = 0.0128 \times 10^{-8} Ns^2 - 0.5015 \times 10^{-4} Ns + 0.2638$
0.67	$B/q^2 = 0.0395 \times 10^{-8} Ns^2 - 0.4378 \times 10^{-4} Ns + 0.5562$
0.83	$B/q^2 = -0.1422 \times 10^{-8} Ns^2 + 0.0304 \times 10^{-4} Ns + 0.7281$
1.00	$B/q^2 = 1.0000$

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Table 3  
Normal Pump Zone  
 $q \geq 0, \alpha \geq 0$  ( $0 \leq q/\alpha \leq 1$ )

		$q/\alpha$						
		0	0.17	0.33	0.50	0.67	0.83	1.00
$h/\alpha^2$	Ns							
	1276	1.29	1.29	1.27	1.23	1.18	1.10	1.00
	1700	1.08	1.08	1.09	1.09	1.10	1.06	1.00
	1940	1.41	1.35	1.30	1.25	1.19	1.11	1.00
	3700	1.40	1.36	1.26	1.09	1.04	1.05	1.00
	4600	1.68	1.56	1.42	1.27	1.17	1.07	1.00
	7600	1.96	1.71	1.40	1.21	1.12	1.06	1.00
	13500	2.73	2.33	2.02	1.76	1.57	1.36	1.00

$q/\alpha$	Equation, $h/\alpha^2 = f(Ns)$
0	$h/\alpha^2 = 0.0870 \times 10^{-8} Ns^2 + 1.1138 \times 10^{-4} Ns + 1.0688$
0.17	$h/\alpha^2 = 0.1324 \times 10^{-8} Ns^2 + 0.7126 \times 10^{-4} Ns + 1.1229$
0.33	$h/\alpha^2 = 0.4333 \times 10^{-8} Ns^2 + 0.0001 \times 10^{-4} Ns + 1.2190$
0.50	$h/\alpha^2 = 0.6344 \times 10^{-8} Ns^2 - 0.4847 \times 10^{-4} Ns + 1.2524$
0.67	$h/\alpha^2 = 0.6186 \times 10^{-8} Ns^2 - 0.5387 \times 10^{-4} Ns + 1.2341$
0.83	$h/\alpha^2 = 0.4189 \times 10^{-8} Ns^2 - 0.4103 \times 10^{-4} Ns + 1.1474$
1.00	$h/\alpha^2 = 1.0000$

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Table 4  
 Normal Pump Zone  
 $q \geq 0, \alpha \geq 0 \quad (0 \leq q/\alpha \leq 1)$

		q/α						
Ns		0	0.17	0.33	0.50	0.67	0.83	1.00
B/α <sup>2</sup>	1276	0.44	0.54	0.66	0.77	0.86	0.94	1.00
	1700	0.48	0.53	0.61	0.69	0.79	0.90	1.00
	1940	-	-	-	-	-	-	1.00
	3700	0.79	0.81	0.82	0.78	0.82	0.92	1.00
	4600	1.07	0.97	0.91	0.88	0.89	0.94	1.00
	7600	1.48	1.18	0.98	0.91	0.94	0.98	1.00
	13500	1.95	1.67	1.35	1.13	1.07	1.05	1.00

q/α	Equation, $B/\alpha^2 = f(Ns)$
0	$B/\alpha^2 = -0.6799 \times 10^{-8} Ns^2 + 2.2739 \times 10^{-4} Ns + 0.1246$
0.17	$B/\alpha^2 = -0.2820 \times 10^{-8} Ns^2 + 1.3533 \times 10^{-4} Ns + 0.3510$
0.33	$B/\alpha^2 = -0.0789 \times 10^{-8} Ns^2 + 0.6909 \times 10^{-4} Ns + 0.5523$
0.50	$B/\alpha^2 = 0.0218 \times 10^{-8} Ns^2 + 0.2942 \times 10^{-4} Ns + 0.6911$
0.67	$B/\alpha^2 = 0.0581 \times 10^{-8} Ns^2 + 0.1225 \times 10^{-4} Ns + 0.8019$
0.83	$B/\alpha^2 = 0.0463 \times 10^{-8} Ns^2 + 0.0425 \times 10^{-4} Ns + 0.9105$
1.00	$B/\alpha^2 = 1.0000$

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Table 5  
 Energy Dissipation Zone  
 $q \leq 0, \alpha \geq 0$  ( $-1 \leq \alpha/q \leq 0$ )

		$\alpha/q$						
Ns		0	-0.17	-0.33	-0.50	-0.67	-0.83	-1.00
$h/q^2$	1276	0.69	0.79	0.90	1.08	1.30	1.62	1.99
	1700	0.77	0.87	0.99	1.14	1.35	1.58	1.89
	1940	-	-	-	-	-	-	-
	3700	1.09	1.27	1.48	1.66	1.94	2.26	2.68
	4600	-	-	-	-	-	-	-
	7600	2.17	2.70	3.20	3.72	4.20	4.73	5.27
	13,500	1.08	1.48	2.02	2.70	3.60	4.82	6.29

$\alpha/q$	Equation, $h/q^2 = f(Ns)$
0	$h/q^2 = -5.3028 \times 10^{-12} Ns^3 + 8.4756 \times 10^{-8} Ns^2 - 1.5591 \times 10^{-4} Ns + 0.7851$
-0.17	$h/q^2 = -6.9699 \times 10^{-12} Ns^3 + 11.4708 \times 10^{-8} Ns^2 - 2.3874 \times 10^{-4} Ns + 0.9463$
-0.33	$h/q^2 = -7.9623 \times 10^{-12} Ns^3 + 13.2437 \times 10^{-8} Ns^2 - 2.6603 \times 10^{-4} Ns + 1.0652$
-0.50	$h/q^2 = -9.5911 \times 10^{-12} Ns^3 + 16.6633 \times 10^{-8} Ns^2 - 4.0247 \times 10^{-4} Ns + 1.3625$
-0.67	$h/q^2 = -9.7667 \times 10^{-12} Ns^3 + 17.2933 \times 10^{-8} Ns^2 - 4.0384 \times 10^{-4} Ns + 1.5671$
-0.83	$h/q^2 = -9.9354 \times 10^{-12} Ns^3 + 18.2964 \times 10^{-8} Ns^2 - 4.4174 \times 10^{-4} Ns + 1.8828$
-1.00	$h/q^2 = -9.0057 \times 10^{-12} Ns^3 + 17.2356 \times 10^{-8} Ns^2 - 3.7935 \times 10^{-4} Ns + 2.1562$

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Table 6  
 Energy Dissipation Zone  
 $q \leq 0, \alpha \geq 0$  ( $-1 \leq \alpha/q \leq 0$ )

		$\alpha/q$						
		0	-0.17	-0.33	-0.50	-0.67	-0.83	-1.00
$B/q^2$	Ns							
	1276	0.86	0.91	0.95	0.98	1.00	1.02	1.04
	1700	1.13	1.18	1.25	1.32	1.37	1.41	1.44
	1940	-	-	-	-	-	-	-
	3700	1.20	1.38	1.55	1.72	1.96	2.19	2.42
	4600	-	-	-	-	-	-	-
	7600	2.10	2.57	3.05	3.40	3.68	3.95	4.22
13.500	0.67	1.07	1.60	2.25	3.16	4.30	5.52	

$\beta/q$	Equation, $B/q^2 = f(Ns)$
0	$B/q^2 = -5.1269 \times 10^{-12} Ns^3 + 8.0581 \times 10^{-8} Ns^2 - 1.8418 \times 10^{-4} Ns + 1.0857$
-0.17	$B/q^2 = -6.1144 \times 10^{-12} Ns^3 + 9.6169 \times 10^{-8} Ns^2 - 1.8674 \times 10^{-4} Ns + 1.1089$
-0.33	$B/q^2 = -7.0421 \times 10^{-12} Ns^3 + 11.2035 \times 10^{-8} Ns^2 - 1.9509 \times 10^{-4} Ns + 1.1426$
-0.50	$B/q^2 = -6.8531 \times 10^{-12} Ns^3 + 10.7919 \times 10^{-8} Ns^2 - 1.2201 \times 10^{-4} Ns + 1.0914$
-0.67	$B/q^2 = -5.0053 \times 10^{-12} Ns^3 + 7.1744 \times 10^{-8} Ns^2 + 1.1473 \times 10^{-4} Ns + 0.8516$
-0.83	$B/q^2 = -2.8710 \times 10^{-12} Ns^3 + 3.2554 \times 10^{-8} Ns^2 + 3.5758 \times 10^{-4} Ns + 0.6043$
-1.00	$B/q^2 = -0.6232 \times 10^{-12} Ns^3 - 0.8140 \times 10^{-8} Ns^2 + 6.0675 \times 10^{-4} Ns + 0.3463$

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Table 7  
 Energy Dissipation Zone  
 $q \leq 0, \alpha \geq 0$  ( $-1 \leq q/\alpha \leq 0$ )

		$q/\alpha$						
		0	-0.17	-0.33	-0.50	-0.67	-0.83	-1.00
$h/q^2$	$N_s$	0	-0.17	-0.33	-0.50	-0.67	-0.83	-1.00
	1276	1.29	1.31	1.35	1.43	1.56	1.73	1.99
	1700	1.08	1.09	1.12	1.21	1.40	1.64	1.89
	1940	1.41	-	-	-	-	-	-
	3700	1.40	1.45	1.55	1.66	1.86	2.20	2.68
	4600	1.68	-	-	-	-	-	-
	7600	1.96	2.16	2.43	2.83	3.41	4.19	5.27
	13,500	2.73	3.27	3.86	4.47	5.07	5.81	6.29

$q/\alpha$	Equation, $h/\alpha^2 = f(N_s)$
0	$h/\alpha^2 = 0.0870 \times 10^{-8} N_s^2 + 1.1138 \times 10^{-4} N_s + 1.0688$
-0.17	$h/\alpha^2 = 0.3941 \times 10^{-8} N_s^2 + 1.1449 \times 10^{-4} N_s + 1.0164$
-0.33	$h/\alpha^2 = 0.5391 \times 10^{-8} N_s^2 + 1.3913 \times 10^{-4} N_s + 1.0107$
-0.50	$h/\alpha^2 = 0.5076 \times 10^{-8} N_s^2 + 1.8965 \times 10^{-4} N_s + 1.0043$
-0.67	$h/\alpha^2 = 0.1151 \times 10^{-8} N_s^2 + 2.8847 \times 10^{-4} N_s + 0.9981$
-0.83	$h/\alpha^2 = -0.5840 \times 10^{-8} N_s^2 + 4.4086 \times 10^{-4} N_s + 0.9666$
-1.00	$h/\alpha^2 = -9.0057 \times 10^{-12} N_s^3 + 17.2356 \times 10^{-8} N_s^2 - 3.7935 \times 10^{-4} N_s + 2.1562$

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Table 8  
 Energy Dissipation Zone  
 $q \neq 0, \alpha \geq 0$  ( $-1 \leq q/\alpha \leq 0$ )

Ns	$q/\alpha$						
	0	-0.17	-0.33	-0.50	-0.67	-0.83	-1.00
1276	0.44	0.38	0.38	0.43	0.53	0.74	1.04
1700	0.48	0.44	0.43	0.49	0.72	1.03	1.44
1940	-	-	-	-	-	-	-
$B/\alpha^2$ 3700	0.79	0.83	0.96	1.11	1.35	1.88	2.42
4600	1.07	-	-	-	-	-	-
7600	1.48	1.58	1.72	2.03	2.47	3.23	4.22
13,500	1.95	2.37	2.94	3.67	4.38	4.99	5.52

$q/\alpha$	Equation, $B/\alpha^2 = f(Ns)$
0	$B/\alpha^2 = -0.6799 \times 10^{-8} Ns^2 + 2.2739 \times 10^{-4} Ns + 0.1246$
-0.17	$B/\alpha^2 = -0.4275 \times 10^{-8} Ns^2 + 2.2813 \times 10^{-4} Ns + 0.0735$
-0.33	$B/\alpha^2 = -0.1311 \times 10^{-8} Ns^2 + 2.2909 \times 10^{-4} Ns + 0.0811$
-0.50	$B/\alpha^2 = 0.0910 \times 10^{-8} Ns^2 + 2.5149 \times 10^{-4} Ns + 0.1019$
-0.67	$B/\alpha^2 = 0.0959 \times 10^{-8} Ns^2 + 2.9678 \times 10^{-4} Ns + 0.1918$
-0.83	$B/\alpha^2 = -0.8172 \times 10^{-8} Ns^2 + 4.6254 \times 10^{-4} Ns + 0.2265$
-1.00	$B/\alpha^2 = -0.6232 \times 10^{-12} Ns^3 - 0.8140 \times 10^{-8} Ns^2 + 6.0675 \times 10^{-4} Ns + 0.3463$

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Table 9  
Turbine Zone

$q/\leq 0, \alpha \leq 0 (0 \leq \alpha/q \leq 1)$

		$\alpha/q$						
		0	0.17	0.33	0.50	0.67	0.83	1.00
Ns								
$h/q^2$	1276	0.69	0.64	0.63	0.66	0.73	0.85	1.01
	1700	0.77	0.71	0.70	0.70	0.73	0.83	0.94
	1940	-	-	-	-	-	0.72	0.83
	3700	1.09	1.01	0.92	0.87	0.81	0.79	0.79
	4600	-	-	-	-	-	-	-
	7600	2.17	1.60	1.10	0.77	0.60	0.47	0.38
	13,500	1.08	0.81	0.78	0.77	0.67	0.44	0.13

$\alpha/q$	Equation, $h/q^2 = f(Ns)$
0	$h/q^2 = -5.3028 \times 10^{-12} Ns^3 + 8.4756 \times 10^{-8} Ns^2 - 1.5591 \times 10^{-4} Ns + 0.7851$
0.17	$h/q^2 = -2.3634 \times 10^{-12} Ns^3 + 2.9630 \times 10^{-8} Ns^2 + 0.5078 \times 10^{-4} Ns + 0.5395$
0.33	$h/q^2 = -1.0533 \times 10^{-8} Ns^2 + 1.6727 \times 10^{-4} Ns + 0.4408$
0.50	$h/q^2 = -0.2722 \times 10^{-8} Ns^2 + 0.4523 \times 10^{-4} Ns + 0.6433$
0.67	$h/q^2 = 0.1252 \times 10^{-8} Ns^2 - 0.2692 \times 10^{-4} Ns + 0.7929$
0.83	$h/q^2 = 0.3005 \times 10^{-8} Ns^2 - 0.7827 \times 10^{-4} Ns + 0.9398$
1.00	$h/q^2 = 0.3575 \times 10^{-8} Ns^2 - 1.2284 \times 10^{-4} Ns + 1.1323$

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Table 10  
Turbine Zone  
 $q \leq 0, \alpha \leq 0$  ( $0 \leq \alpha/q \leq 1$ )

		$\alpha/q$						
	$N_s$	0	0.17	0.33	0.50	0.67	0.83	1.00
$B/q^2$	1276	0.86	0.81	0.76	0.69	0.62	0.54	0.46
	1700	1.13	1.03	0.95	0.88	0.83	0.79	0.73
	1940	-	-	-	-	-	-	-
	3700	1.20	1.09	0.96	0.84	0.70	0.56	0.45
	4600	-	-	-	-	-	-	-
	7600	2.10	1.57	1.15	0.83	0.56	0.31	0
	13,500	0.67	0.43	0.46	0.63	0.54	0.18	-0.15

$\alpha/q$	Equation, $B/q^2 = f(N_s)$
0	$B/q^2 = -5.1269 \times 10^{-12} N_s^3 + 8.0581 \times 10^{-8} N_s^2 - 1.8418 \times 10^{-4} N_s + 1.0857$
0.17	$B/q^2 = -2.7431 \times 10^{-12} N_s^3 + 3.7035 \times 10^{-8} N_s^2 - 0.3342 \times 10^{-4} N_s + 0.8813$
0.33	$B/q^2 = -1.2870 \times 10^{-8} N_s^2 + 1.6269 \times 10^{-4} N_s + 0.6169$
0.50	$B/q^2 = -0.3919 \times 10^{-8} N_s^2 + 0.4535 \times 10^{-4} N_s + 0.7216$
0.67	$B/q^2 = -0.0954 \times 10^{-8} N_s^2 - 0.3050 \times 10^{-4} N_s + 0.7701$
0.83	$B/q^2 = 0.1887 \times 10^{-8} N_s^2 - 0.6932 \times 10^{-4} N_s + 0.7637$
1.00	$B/q^2 = 0.4065 \times 10^{-8} N_s^2 - 1.2508 \times 10^{-4} N_s + 0.7829$

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Table 11

Turbine Zone

 $q \leq 0, \alpha \leq 0$  ( $0 \leq q/\alpha \leq 1$ )

Ns	q/α						
	0	0.17	0.33	0.50	0.67	0.83	1.00
1276	0.63	0.66	0.69	0.73	0.79	0.88	1.01
1700	-	-	0.63	0.67	0.73	0.82	0.94
1940	-	-	-	0.65	0.68	0.73	0.83
h/α <sup>2</sup> 3700	-	-	-	-	0.56	0.67	0.79
4600	-	-	-	-	-	-	-
7600	-0.67	-0.46	-0.28	-0.04	0.06	0.19	0.38
13,500	-2.20	-1.55	-1.14	-0.86	-0.60	-0.28	0.13

q/α	Equation, $h/\alpha^2 = f(Ns)$
0	$h/\alpha^2 = -0.4398 \times 10^{-8} Ns^2 - 1.6653 \times 10^{-4} Ns + 0.8497$
0.17	$h/\alpha^2 = -0.0625 \times 10^{-8} Ns^2 - 1.7155 \times 10^{-4} Ns + 0.8799$
0.33	$h/\alpha^2 = 0.0651 \times 10^{-8} Ns^2 - 1.5960 \times 10^{-4} Ns + 0.8958$
0.50	$h/\alpha^2 = -0.1458 \times 10^{-8} Ns^2 - 1.0806 \times 10^{-4} Ns + 0.8646$
0.67	$h/\alpha^2 = -0.0427 \times 10^{-8} Ns^2 - 1.0702 \times 10^{-4} Ns + 0.9187$
0.83	$h/\alpha^2 = 0.1323 \times 10^{-8} Ns^2 - 1.1355 \times 10^{-4} Ns + 1.0063$
1.00	$h/\alpha^2 = 0.3575 \times 10^{-8} Ns^2 - 1.2284 \times 10^{-4} Ns + 1.1323$

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Table 12

Turbine Zone

 $q \leq 0, \alpha \leq 0$  ( $0 \leq q/\alpha \leq 1$ )

		q/ $\alpha$						
Ns		0	0.17	0.33	0.50	0.67	0.83	1.00
B/ $\alpha^2$	1276	-0.67	-0.37	-0.16	-0.03	0.10	0.24	0.46
	1700	-	-	-0.10	0.05	0.23	0.48	0.73
	1940	-	-	-	-	-	-	-
	3700	-	-	-	-	-0.10	0.18	0.45
	4600	-	-	-	-	-	-	-
	7600	-1.50	-1.09	-0.79	-0.63	-0.46	-0.29	0
	13,500	-2.33	-1.58	-1.30	-1.10	-0.84	-0.55	-0.15

q/ $\alpha$	Equation, B/ $\alpha^2 = f(Ns)$		
0	B/ $\alpha^2 = -0.0772 \times 10^{-8} Ns^2$	$-1.2440 \times 10^{-4} Ns$	-0.5100
0.17	B/ $\alpha^2 = 0.2520 \times 10^{-8} Ns^2$	$-1.3622 \times 10^{-4} Ns$	-0.2003
0.33	B/ $\alpha^2 = 0.1513 \times 10^{-8} Ns^2$	$-1.1999 \times 10^{-4} Ns$	+0.0423
0.50	B/ $\alpha^2 = 0.1755 \times 10^{-8} Ns^2$	$-1.1862 \times 10^{-4} Ns$	+0.1792
0.67	B/ $\alpha^2 = 0.2856 \times 10^{-8} Ns^2$	$-1.2598 \times 10^{-4} Ns$	+0.3384
0.83	B/ $\alpha^2 = 0.3540 \times 10^{-8} Ns^2$	$-1.3053 \times 10^{-4} Ns$	+0.5543
1.00	B/ $\alpha^2 = 0.4065 \times 10^{-8} Ns^2$	$-1.2508 \times 10^{-4} Ns$	+0.7829

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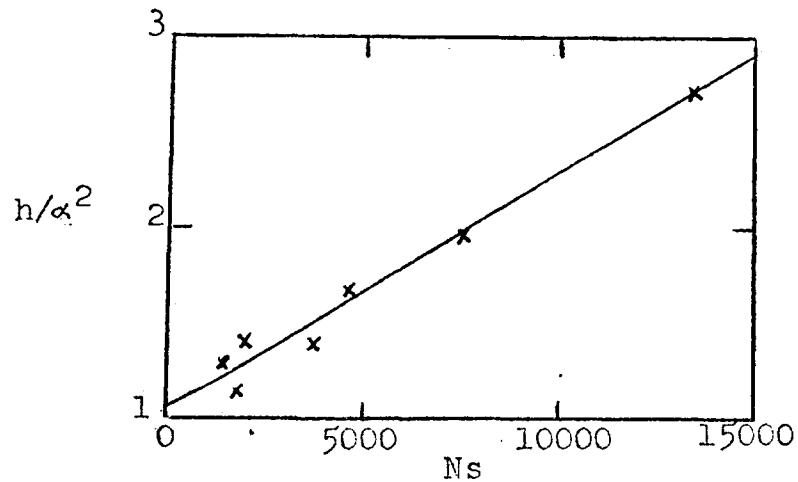


FIGURE 1

Data from TABLE 3 at  $q/\alpha = 0$ 

$Ns$	1276	1700	1940	3700	4600	7600	13500
$h/\alpha^2$	1.29	1.08	1.41	1.40	1.68	1.96	2.73

has a 2nd degree polynomial fit through it resulting in the following equation:

$$h/\alpha^2 = 0.0870 \times 10^{-8} Ns^2 + 1.1138 \times 10^{-4} Ns + 1.0688$$

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APPENDIX

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Nomenclature

H	Pressure head
$H_R$	Rated pressure head of a pump
h	Dimensionless pressure head $H/H_R$
N	Rotational speed of a pump
$N_R$	Rated rotational speed of a pump
$N_S$	Specific pump speed $\frac{N_R \sqrt{Q_R}}{(H_R)^{3/4}}$
Q	Discharge
$Q_R$	Rated discharge of a pump
q	Dimensionless discharge $Q/Q_R$
T	Torque on a pump
$T_R$	Rated torque of a pump
$\alpha$	Dimensionless speed ratio $N/N_R$
$\beta$	Dimensionless torque ratio $T/T_R$

$A_n$   
 $B_n$   
 $C_n$   
 $D_n$   
 $E_n$   
 $F_n$   
 $G_n$

Constant coefficients in the 6th degree polynomial equations representing a pump's characteristics

Note: "Rated" refers to the point of best efficiency at normal speed.

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## DESCRIPTION OF PROGRAM

The main program uses the method of characteristics as described in Reference 2 to compute the transients in the pipes. This involves using two equations; one from the downstream direction and the other from the upstream direction, as well as a continuity equation for head, and another equation for discharge. These four equations are solved simultaneously for head and discharge at the point of interest. When a pipe boundary is a pump, the head and discharge at the pump are a function of the head and discharge from the upstream and downstream pipes, the  $\alpha, \beta$  relation (function of  $WR^2$ ), and the pump's characteristic relation. Thus four relations are used to solve for  $\alpha, \beta, q$ , and  $h$  of the pump at any specific time. At each time increment the pump characteristic relation uses two of the twelve equations determined by the method shown in this thesis. The two equations to be used at any given time are determined by the zone of operation in which the pump happens to be.

Two subroutines in the program use the results of this thesis. They are the PUMPSB subroutine and the COEFP subroutine. Following is a description and listing of each.

## PUMPSB

The pump subroutine (PUMPSB(I), I representing the pump in question) has two main functions; (1) to determine the initial operating condition of the pump, and (2) to determine the

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operating condition at any time after a power failure. The discussion here will be concerned with determining the operating condition of the pump after power failure. The upstream pipe characteristic has the form:

$$QP(K,J) + CF2 \times HP(K,J) = CF1.$$

The downstream pipe characteristic has the form:

$QP(K,J) + CB2 \times HP(K,J) = CB1$ , where (K) represents the pipe of interest and (J) represents the segment in pipe (k) that is closest to the pump. At statement 20100 the pump speed is estimated using the speed at the previous time increment (ALPHAP(I), I is the pump under consideration) and an estimated change in speed (DALPP(I)). The discharge is then estimated (QPPI) using a similar procedure. This determines the value of  $\alpha/q$  used in the thesis so that the zone of operation and the appropriate 6th degree polynomial coefficients can be stored in A1, B1, C1, D1, E1, F1, and G1. U and D are then substituted for  $\alpha$  and q (ALI and QPPI).

Statement 20300 computes the head across the pump (HPP(I)), and the next statement computes the torque on the pump (BETAI) (the h and  $\beta$  used in the paper).

The downstream pipe head (HP(JDPM,(1))) and discharge (QP(JDPM, 1)) are then calculated using an equation derived from the characteristic equations and the continuity equations. This head and discharge are compared with the discharge and speed assumed at the beginning of the subroutine. If the error is too large, the procedure is repeated, and continues iterating until

the error is acceptable. Values of  $\alpha$ ,  $\beta$ ,  $q$ , and  $h$  for the pump, and the head and discharge of the downstream and upstream pipe are then established for the specified time (TIME). The subroutine is used once each time increment so a history of the transient is known.

#### COEFP

The subroutine (COEFP(I) I represents the pump in question) computes the coefficients of the 12 sixth degree equations which represent a pump's characteristics. The specific speed (SNS(I)) of the pump is determined in the main program from the values read in representing the pump, i.e., rated head, discharge, and speed.  $X_1, X_2, \dots, X_6$  are the values of the abscissa in the paper, and the  $Y_0, Y_1, \dots, Y_6$  are determined from the appropriate equation for each table. From statement 19200 on, the Gaussian elimination method is used to determine the coefficients of a 6th degree polynomial which will go through the 7 points (X,Y) of the characteristic. The coefficients have the form AHQP(I,J), BHQP(I,J), CHQP(I,J), DHQP(I,J), and so forth. These are common variables to the program so they can be used in the subroutine that computes the pump transient.

The variable NSNS(I) can have values of 0, 1276, 7600, or 13,500. If the value of 0 is stored, the subroutine will produce characteristics as outlined in the paper. If any one of the other three are stored, the corresponding original characteristic is

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reproduced. Thus the program has the option of using a specific known characteristic or determining a characteristic using the procedure outlined in the thesis.

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15.4I

SUBROUTINE PUMPSB(I, JDPN, JUPN, JUPNN)

C

C

SUBROUTINE TO COMPUTE H [ Q AT A PUMP

C

```

COMMON/GSUB1/ Q(20,200), H(20,200), QP(20,200), HP(20,200)
COMMON/GSUB2/ PIPDIA(20), PFRICT(20), NSEG(20), DT, TIME
COMMON/CDT/ PLENGH(20), PCELET(20), NPIPES
COMMON/PPP/QRATEP(10), HRATEP(10), SRATEP(10),
1         WR2RP(10), AK1P(10), ALPHAP(10), BETAP(10),
2         QPP(10), HPP(10), DALPP(10), DQPP(10),
3         IDPNP(10), IUPNP(10)

```

C

```

COMMON/PPPC/AHQP(10,3), BHQP(10,3), CHQP(10,3),
1         DHQP(10,3), EHQP(10,3), FHQP(10,3), GHQP(10,3),
2         AHAP(10,3), BHAP(10,3), CHAP(10,3),
3         DHAP(10,3), EHAP(10,3), FHAP(10,3), GHAP(10,3),
4         ABQP(10,3), BBQP(10,3), CBQP(10,3),
5         DBQP(10,3), EBQP(10,3), FBQP(10,3), GBQP(10,3),
6         ABAP(10,3), BBAP(10,3), CBAP(10,3),
7         DBAP(10,3), EBAP(10,3), FBAP(10,3), GBAP(10,3),
8         SNS(10), NSNS(10)

```

C

C

AT I=3 ASSUME ALPHA=1- GET STEADY STATE PUMPING

C

C

C

C

C

C

C

C

C

C

C

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C

C

CF2=.7854\*PIPDIA(JUPN)\*\*2\*32.2/PCELET(JUPN)

CF1=Q(JUPN, JUPNN-1)+CF2\*H(JUPN, JUPNN-1)

1 -DT\*PFRICT(JUPN)\*ABS(Q(JUPN, JUPNN-1))\*Q(JUPN, JUPNN-1)

2 /(2.\*PIPDIA(JUPN)\*\*3\*.7854)

CB2=-.7854\*PIPDIA(JDPN)\*\*2\*32.2/PCELET(JDPN)

CB1=Q(JDPN, 2)+CB2\*H(JDPN, 2)

1 -DT\*PFRICT(JDPN)\*ABS(Q(JDPN, 2))\*Q(JDPN, 2)

2 /(2.\*PIPDIA(JDPN)\*\*3\*.7854)

IF(TIME-.0001)20010,20010,20100

20010 QPP1=Q(JDPN, 1)/QRATEP(I)

20040 IF(QPP1-1)20060,20050,20050

20050 A1=AHQP(I, 1)

B1=BHQP(I, 1)

C1=CHQP(I, 1)

D1=DHQP(I, 1)

E1=EHQP(I, 1)

F1=FHQP(I, 1)

G1=GHQP(I, 1)

A2=ABQP(I, 1)

B2=BBQP(I, 1)

C2=CBQP(I, 1)

D2=DBQP(I, 1)

E2=EBQP(I, 1)

F2=FBQP(I, 1)

G2=GBQP(I, 1)

QA=QPP1

GO TO 20070

20060 A1=AHAP(I, 1)

B1=BHAP(I, 1)

C1=CHAP(I, 1)

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```

15.4I      PUMPSB

          D1=DHAP(I,1)
          E1=EHAP(I,1)
          F1=FHAP(I,1)
          G1=GHAP(I,1)
          A2=ABAP(I,1)
          B2=BBAP(I,1)
          C2=CBAP(I,1)
          D2=DBAP(I,1)
          E2=EBAP(I,1)
          F2=FBAP(I,1)
          G2=GBAP(I,1)
          QA=1./QPP1
          GO TO 20071
20070      HPP(I)=A1/  QA**4+B1/  QA**3+C1/  QA**2+D1/  QA+
           1      E1      +F1*QA      +G1*QA**2
          GO TO 20072
20071      HPP(I)=A1/QA**6+B1/QA**5+C1/QA**4+D1/QA**3+E1/QA**2
           1      +F1/QA+G1
20072      HP(JDPN,1)=((CF1-CB1)/CF2+HRATEP(I)*HPP(I))/(1.-CB2/CF2)
          QPP2=(CB1-CB2*HP(JDPN,1))/QRATEP(I)
          WRITE(61,2J075)QPP1,QPP2,HPP(I),HP(JDPN,1)
20075      FORMAT(4E12.5)
          IF(ABS(QPP1-QPP2)-.0004)20090,20090,20080
20080      QPP1=QPP2
          GO TO 20040
20090      HP(JUPN,JUPNN)=(CF1-QPP2*QRATEP(I))/CF2
          QP(JUPN,JUPNN)=QPP2*QRATEP(I)
          ALPHAP(I)=1.
          IF(QPP1-1.)20095,20095,20097
20095      BETAP(I)=A2/QA**6+B2/QA**5+C2/QA**4+D2/QA**3+E2/QA**2
           1      +F2/QA+G2
          GO TO 20098
20097      BETAP(I)=A2/QA**4+B2/QA**3+C2/QA**2+D2/QA+
           1      E2+F2*QA+G2*QA**2
20098      DALPP(I)=-.003
          DQPP(I)=-.002
          QP(JDPN,1)  =QPP2*QRATEP(I)
          QPP(I)=QPP2
          RETURN
C
C
C  DO TRANSIENT PUMP CALCULATIONS
C
20100      AL1=ALPHAP(I)+DALPP(I)
          QPP1=QPP(I)+DQPP(I)
20110      IF(AL1/QPP1)20160,20120,20120
20120      IF(AL1)20190,20130,20130
20130      IF(AL1/QPP1-1.)20140,20140,20150
20140      A1=AHQP(I,1)
          B1=BHQP(I,1)
          C1=CHQP(I,1)
          D1=DHQP(I,1)
          E1=EHQP(I,1)
          F1=FHQP(I,1)
          G1=GHQP(I,1)

```

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15.4I

PUMPSB

A2=A9QP(I,1)  
 B2=BBQP(I,1)  
 C2=CBQP(I,1)  
 D2=DBQP(I,1)  
 E2=EBQP(I,1)  
 F2=FBQP(I,1)  
 G2=GBQP(I,1)  
 U=AL1  
 D=QPP1

GO TO 20300

C

C PUMP\_ZONE Q/A LESS 1

C

20150

A1=AHAP(I,1)  
 B1=BHAP(I,1)  
 C1=CHAP(I,1)  
 D1=DHAP(I,1)  
 E1=EHAP(I,1)  
 F1=FHAP(I,1)  
 G1=GHAP(I,1)  
 A2=ABAP(I,1)  
 B2=BBAP(I,1)  
 C2=CBAP(I,1)  
 D2=DBAP(I,1)  
 E2=EBAP(I,1)  
 F2=FBAP(I,1)  
 G2=GBAP(I,1)  
 U=QPP1  
 D=AL1

GO TO 20300

C

C ENERGY DISS\_ZONE A/Q LESS 1

C

20160 IF(AL1/QPP1+1.)20180,20180,20170

20170

A1=AHQP(I,2)  
 B1=BHQP(I,2)  
 C1=CHQP(I,2)  
 D1=DHQP(I,2)  
 E1=EHQP(I,2)  
 F1=FHQP(I,2)  
 G1=GHQP(I,2)  
 A2=ABQP(I,2)  
 B2=BBQP(I,2)  
 C2=CBQP(I,2)  
 D2=DBQP(I,2)  
 E2=EBQP(I,2)  
 F2=FBQP(I,2)  
 G2=GBQP(I,2)  
 U=AL1  
 D=QPP1

GO TO 20300

C

C ENERGY DISS\_ZONE Q/A LESS 1

C

20180

A1=AHAP(I,2)

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15.41

PJMP5B

B1=BHAP(I,2)  
 C1=CHAP(I,2)  
 D1=DHAP(I,2)  
 E1=EHAP(I,2)  
 F1=FHAP(I,2)  
 G1=GHAP(I,2)  
 A2=ABAP(I,2)  
 B2=BBAP(I,2)  
 C2=CBAP(I,2)  
 D2=DBAP(I,2)  
 E2=EBAP(I,2)  
 F2=FBAP(I,2)  
 G2=GBAP(I,2)  
 U=QPP1  
 D=AL1

GO TO 20300

C

C TURBINE ZONE A/Q LESS 1

C

20190 IF (AL1/QPP1-1.) 20200, 20200, 20210

20200

A1=AHQP(I,3)  
 B1=BHQP(I,3)  
 C1=CHQP(I,3)  
 D1=DHQP(I,3)  
 E1=EHQP(I,3)  
 F1=FHQP(I,3)  
 G1=GHQP(I,3)  
 A2=ABQP(I,3)  
 B2=BBQP(I,3)  
 C2=CBQP(I,3)  
 D2=DBQP(I,3)  
 E2=EBQP(I,3)  
 F2=FBQP(I,3)  
 U=AL1  
 D=QPP1

GO TO 20300

C

C TURBINE ZONE Q/A LESS 1

C

20210

A1=AHAP(I,3)  
 B1=BHAP(I,3)  
 C1=CHAP(I,3)  
 D1=DHAP(I,3)  
 E1=EHAP(I,3)  
 F1=FHAP(I,3)  
 G1=GHAP(I,3)  
 A2=ABAP(I,3)  
 B2=BBAP(I,3)  
 C2=CBAP(I,3)  
 D2=DBAP(I,3)  
 E2=EBAP(I,3)  
 F2=FBAP(I,3)  
 G2=GBAP(I,3)  
 U=QPP1  
 D=AL1

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15.4I PUMPSB

C

C

CALC. HEAD ACROSS PUMP

C

```

20300      HPP(I)=A1*U**6/D**4 +B1*U**5/D**3 + C1*U**4/D**2
           +D1*U**3/D +E1*U*U +F1*U*D +G1*D*D
           1
           BETA1= A2*U**6/D**4 +B2*U**5/D**3 + C2*U**4/D**2
           1
           +D2*U**3/D +E2*U*U +F2*U*D +G2*D*D
           HP(JDPN,1)=((CF1-CB1)/CF2+HRATEP(I)*HPP(I))/(1.-CB2/CF2)
           QP(JDPN,1)=CB1-CB2*HP(JDPN,1)
           IF (ABS(QP(JDPN,1)-QPP1*QRATEP(I))-.0050) 20320,20320,20310
20310      QPP1=QP(JDPN,1)/QRATEP(I)
           AL1=-AK1P(I)*(BETA1+BETAP(I))*DT+ALPHAP(I)
           GO TO 20110
20320      QP(JUPN,JUPNN)=QP(JDPN,1)
           HP(JUPN,JUPNN)=HP(JDPN,1)-HPP(I)*HRATEP(I)
           DALPP(I)=AL1-ALPHAP(I)
           ALPHAP(I)=AL1
           BETAP(I)=BETA1
           DQPP(I)=QPP1-QPP(I)
           QPP(I)=QP(JDPN,1)/QRATEP(I)
           RETURN
           END

```

MULTI-BANK COMPILATION.



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```

4I      COEFP
19005  Y0=0.000E-12*SNS(I)**3+2.109E-8*SNS(I)**2-3.501E-4*SNS(I)-0.118
        Y1=0.000E-12*SNS(I)**3+1.237E-8*SNS(I)**2-2.221E-4*SNS(I)-0.157
        Y2=0.000E-12*SNS(I)**3+0.511E-8*SNS(I)**2-1.197E-4*SNS(I)-0.076
        Y3=0.000E-12*SNS(I)**3+0.200E-8*SNS(I)**2-0.678E-4*SNS(I)+0.059
        Y4=0.000E-12*SNS(I)**3+0.053E-8*SNS(I)**2-0.377E-4*SNS(I)+0.303
        Y5=0.000E-12*SNS(I)**3-0.149E-8*SNS(I)**2+0.034E-4*SNS(I)+0.594
        Y6=0.000E-12*SNS(I)**3+0.000E-8*SNS(I)**2+0.000E-4*SNS(I)+1.000

```

```

19006      J=1
           K=1

```

```

           GO TO 19200

```

```

C

```

```

C           H/A2 PUMP Q/A      F 3

```

```

C

```

```

19010      IF(NSNS(I))19015,19015,19011

```

```

19011      IF(NSNS(I)-7600)19012,19013,19014

```

```

C

```

```

19012      Y0=1.29

```

```

           Y1=1.29

```

```

           Y2=1.27

```

```

           Y3=1.23

```

```

           Y4=1.18

```

```

           Y5=1.10

```

```

           Y6=1.00

```

```

           GOTO 19015

```

```

19013      Y0=1.96

```

```

           Y1=1.71

```

```

           Y2=1.40

```

```

           Y3=1.21

```

```

           Y4=1.12

```

```

           Y5=1.06

```

```

           Y6=1.00

```

```

           GOTO 19015

```

```

19014      Y0=2.73

```

```

           Y1=2.33

```

```

           Y2=2.02

```

```

           Y3=1.76

```

```

           Y4=1.57

```

```

           Y5=1.36

```

```

           Y6=1.00

```

```

           GOTO 19016

```

```

19015  Y0=0.000E-12*SNS(I)**3+0.087E-8*SNS(I)**2+1.114E-4*SNS(I)+1.069

```

```

        Y1=0.000E-12*SNS(I)**3+0.132E-8*SNS(I)**2+0.713E-4*SNS(I)+1.123

```

```

        Y2=0.000E-12*SNS(I)**3+0.433E-8*SNS(I)**2+0.000E-4*SNS(I)+1.219

```

```

        Y3=0.000E-12*SNS(I)**3+0.634E-8*SNS(I)**2-0.485E-4*SNS(I)+1.252

```

```

        Y4=0.000E-12*SNS(I)**3+0.619E-8*SNS(I)**2-0.589E-4*SNS(I)+1.234

```

```

        Y5=0.000E-12*SNS(I)**3+0.419E-8*SNS(I)**2-0.410E-4*SNS(I)+1.147

```

```

        Y6=0.000E-12*SNS(I)**3+0.000E-8*SNS(I)**2+0.000E-4*SNS(I)+1.000

```

```

19016      J=1

```

```

           K=2

```

```

           GO TO 19200

```

```

C

```

```

C           B/Q2 PUMP      A/Q      F 2

```

```

C

```

```

19020      IF(NSNS(I))19025,19025,19021

```

```

19021      IF(NSNS(I))19022,19023,19024

```

T-1566

4I COEFP

C

19022 Y0=-0.30  
 Y1=-0.16  
 Y2= 0.01  
 Y3= 0.20  
 Y4= 0.43  
 Y5= 0.70  
 Y6= 1.00

GOTO 19026

19023 Y0=-1.55  
 Y1=-0.93  
 Y2=-0.47  
 Y3=-0.11  
 Y4= 0.29  
 Y5= 0.67  
 Y6= 1.00

GOTO 19026

19024 Y0=-0.55  
 Y1=-0.62  
 Y2=-0.60  
 Y3=-0.39  
 Y4= 0.03  
 Y5= 0.51  
 Y6= 1.00

GOTO 19026

19025 Y0=0.0000E-12\*SNS(I)\*\*3+3.017E-8\*SNS(I)\*\*2-4.670E-4\*SNS(I)+0.247  
 Y1=0.0000E-12\*SNS(I)\*\*3+1.426E-8\*SNS(I)\*\*2-2.483E-4\*SNS(I)+0.134  
 Y2=0.0000E-12\*SNS(I)\*\*3+0.441E-8\*SNS(I)\*\*2-1.150E-4\*SNS(I)+0.150  
 Y3=0.0000E-12\*SNS(I)\*\*3+0.013E-8\*SNS(I)\*\*2-0.502E-4\*SNS(I)+0.264  
 Y4=0.0000E-12\*SNS(I)\*\*3+0.040E-8\*SNS(I)\*\*2-0.438E-4\*SNS(I)+0.556  
 Y5=0.0000E-12\*SNS(I)\*\*3-0.142E-8\*SNS(I)\*\*2+0.030E-4\*SNS(I)+0.728  
 Y6=0.0000E-12\*SNS(I)\*\*3+0.000E-8\*SNS(I)\*\*2+0.000E-4\*SNS(I)+1.000

19026 J=1  
 K=3

GO TO 19200

C

C B/A2 PUMP Q/A F 4

C

19030 IF(NSNS(I))19035,19035,19031  
 19031 IF(NSNS(I)-7600)19032,19033,19034

C

19032 Y0=0.44  
 Y1=0.54  
 Y2=0.56  
 Y3=0.77  
 Y4=0.86  
 Y5=0.94  
 Y6=1.00

GOTO 19035

19033 Y0=1.48  
 Y1=1.18  
 Y2=0.98  
 Y3=0.91  
 Y4=0.94  
 Y5=0.98

T-1566

4I COEFP

```

      Y6=1.00
      GOTO 19036
19034  Y0=1.95
      Y1=1.67
      Y2=1.35
      Y3=1.13
      Y4=1.07
      Y5=1.05
      Y6=1.00
      GOTO 19035
19035  Y0=0.000E-12*SNS(I)**3-0.680E-8*SNS(I)**2+2.274E-4*SNS(I)+0.125
      Y1=0.000E-12*SNS(I)**3-0.282E-8*SNS(I)**2+1.353E-4*SNS(I)+0.351
      Y2=0.000E-12*SNS(I)**3-0.079E-8*SNS(I)**2+0.691E-4*SNS(I)+0.552
      Y3=0.000E-12*SNS(I)**3+0.022E-8*SNS(I)**2+0.294E-4*SNS(I)+0.691
      Y4=0.000E-12*SNS(I)**3+0.058E-8*SNS(I)**2+0.123E-4*SNS(I)+0.802
      Y5=0.000E-12*SNS(I)**3+0.046E-8*SNS(I)**2+0.043E-4*SNS(I)+0.911
      Y6=0.000E-12*SNS(I)**3+0.000E-8*SNS(I)**2+0.000E-4*SNS(I)+1.000
19036  J=1
      K=4
      GO TO 19200

```

C

C

C DO ENERGY DISSIPATION ZONE

C

C

H/Q2 ENERGY DISS A/Q F 5

C

```

19040  X1=-.17
      X2=-.33
      X3=-.50
      X4=-.67
      X5=-.83
      X6=-1.00

```

IF(NSNS(I))19045,19045,19041

19041 IF(NSNS(I)-7600)19042,19043,19044

C

```

19042  Y0=0.59
      Y1=0.79
      Y2=0.90
      Y3=1.08
      Y4=1.30
      Y5=1.52
      Y6=1.99

```

GOTO 19046

```

19043  Y0=2.17
      Y1=2.70
      Y2=3.20
      Y3=3.72
      Y4=4.20
      Y5=4.73
      Y6=5.27

```

GOTO 19045

```

19044  Y0=1.08
      Y1=1.48
      Y2=2.02
      Y3=2.70

```

T-1566

.4I COEFP

Y4=3.60

Y5=4.82

Y6=6.29

GOTO 19045

19045 Y0=-5.303E-12\*SNS(I)\*\*3+08.476E-8\*SNS(I)\*\*2-1.560E-4\*SNS(I)+0.785  
 Y1=-6.970E-12\*SNS(I)\*\*3+11.471E-8\*SNS(I)\*\*2-2.387E-4\*SNS(I)+0.946  
 Y2=-7.962E-12\*SNS(I)\*\*3+13.244E-8\*SNS(I)\*\*2-2.660E-4\*SNS(I)+1.065  
 Y3=-9.591E-12\*SNS(I)\*\*3+16.663E-8\*SNS(I)\*\*2-4.025E-4\*SNS(I)+1.363  
 Y4=-9.767E-12\*SNS(I)\*\*3+17.293E-8\*SNS(I)\*\*2-4.038E-4\*SNS(I)+1.567  
 Y5=-9.935E-12\*SNS(I)\*\*3+18.296E-8\*SNS(I)\*\*2-4.417E-4\*SNS(I)+1.883  
 Y6=-9.006E-12\*SNS(I)\*\*3+17.236E-8\*SNS(I)\*\*2-3.794E-4\*SNS(I)+2.156

19046 J=2

K=1

GO TO 19200

C

C

H/A2 ENERGY DISS Q/A F7

C

19050 IF(NSNS(I))19055,19055,19051

19051 IF(NSNS(I)-7600)19052,19053,19054

C

19052 Y0=1.29

Y1=1.31

Y2=1.35

Y3=1.43

Y4=1.56

Y5=1.73

Y6=1.99

GOTO 19053

19053 Y0=1.96

Y1=2.16

Y2=2.43

Y3=2.83

Y4=3.41

Y5=4.19

Y6=5.27

GOTO 19056

19054 Y0=2.73

Y1=3.27

Y2=3.86

Y3=4.47

Y4=5.07

Y5=5.81

Y6=6.29

GOTO 19056

19055 Y0=+0.000E-12\*SNS(I)\*\*3+00.087E-8\*SNS(I)\*\*2+1.114E-4\*SNS(I)+1.069

Y1=+0.000E-12\*SNS(I)\*\*3+00.394E-8\*SNS(I)\*\*2+1.145E-4\*SNS(I)+1.016

Y2=+0.000E-12\*SNS(I)\*\*3+00.539E-8\*SNS(I)\*\*2+1.391E-4\*SNS(I)+1.011

Y3=+0.000E-12\*SNS(I)\*\*3+00.508E-8\*SNS(I)\*\*2+1.897E-4\*SNS(I)+1.004

Y4=+0.000E-12\*SNS(I)\*\*3+00.115E-8\*SNS(I)\*\*2+2.885E-4\*SNS(I)+0.998

Y5=+0.000E-12\*SNS(I)\*\*3-00.584E-8\*SNS(I)\*\*2+4.409E-4\*SNS(I)+0.967

Y6=-9.006E-12\*SNS(I)\*\*3+17.236E-8\*SNS(I)\*\*2-3.794E-4\*SNS(I)+2.156

19056 J=2

K=2

GO TO 19200

C

T-1566

```

,4I          COEFP
C
C          B/Q2 ENERGY DISS A/Q          F 6
19060      IF(NSNS(I))19065,19065,19061
19061      IF(NSNS(I)-7600)19062,19063,19064
C
19062      Y0=0.86
           Y1=0.91
           Y2=0.95
           Y3=0.98
           Y4=1.00
           Y5=1.02
           Y6=1.04
           GOTO 19066
19063      Y0=2.10
           Y1=2.57
           Y2=3.05
           Y3=3.40
           Y4=3.58
           Y5=3.95
           Y6=4.22
           GOTO 19066
19064      Y0=0.57
           Y1=1.07
           Y2=1.60
           Y3=2.25
           Y4=3.16
           Y5=4.30
           Y6=5.52
           GOTO 19066
19065      Y0=-5.127E-12*SNS(I)**3+08.058E-8*SNS(I)**2-1.842E-4*SNS(I)+1.086
           Y1=-6.114E-12*SNS(I)**3+09.617E-8*SNS(I)**2-1.867E-4*SNS(I)+1.109
           Y2=-7.042E-12*SNS(I)**3+11.204E-8*SNS(I)**2-1.951E-4*SNS(I)+1.143
           Y3=-6.853E-12*SNS(I)**3+10.792E-8*SNS(I)**2-1.220E-4*SNS(I)+1.091
           Y4=-5.005E-12*SNS(I)**3+07.174E-8*SNS(I)**2+1.147E-4*SNS(I)+0.852
           Y5=-2.871E-12*SNS(I)**3+03.255E-8*SNS(I)**2+3.576E-4*SNS(I)+0.604
           Y6=-0.623E-12*SNS(I)**3-00.814E-8*SNS(I)**2+6.068E-4*SNS(I)+0.346
19066      J=2
           K=3
           GO TO 19200
C
C          B/A2 ENERGY DISS Q/A          F8
C
19070      IF(NSNS(I))19075,19075,19071
19071      IF(NSNS(I)-7600)19072,19073,19074
C
19072      Y0=0.44
           Y1=0.38
           Y2=0.38
           Y3=0.43
           Y4=0.53
           Y5=0.74
           Y6=1.04
           GOTO 19075
19073      Y0=1.48
           Y1=1.58

```

T-1566

```

.4I      COEFP
          Y2=1.72
          Y3=2.03
          Y4=2.47
          Y5=3.23
          Y6=4.22
19074    GOTO 19075
          Y0=1.95
          Y1=2.37
          Y2=2.94
          Y3=3.67
          Y4=4.38
          Y5=4.99
          Y6=5.52
19075    GOTO 19075
          Y0=+0.000E-12*SNS(I)**3-00.680E-8*SNS(I)**2+2.274E-4*SNS(I)+0.125
          Y1=+0.000E-12*SNS(I)**3-00.428E-8*SNS(I)**2+2.281E-4*SNS(I)+0.074
          Y2=+0.000E-12*SNS(I)**3-00.131E-8*SNS(I)**2+2.291E-4*SNS(I)+0.081
          Y3=+0.000E-12*SNS(I)**3+00.091E-8*SNS(I)**2+2.515E-4*SNS(I)+0.102
          Y4=+0.000E-12*SNS(I)**3+00.096E-8*SNS(I)**2+2.968E-4*SNS(I)+0.192
          Y5=+0.000E-12*SNS(I)**3-00.817E-8*SNS(I)**2+4.625E-4*SNS(I)+0.227
          Y6=-0.623E-12*SNS(I)**3-00.814E-8*SNS(I)**2+6.068E-4*SNS(I)+0.346
19076    J=2
          K=4
          GO TO 19200
C
C
C  DO TURBINE ZONE
C
C          H/Q2 TURB A/Q   F9
C
19080    X1=.17
          X2=.33
          X3=.50
          X4=.67
          X5=.83
          X6=1.00
          IF(NSNS(I))19085,19085,19081
19081    IF(NSNS(I)-7500)19082,19083,19084
C
19082    Y0=0.59
          Y1=0.64
          Y2=0.63
          Y3=0.66
          Y4=0.73
          Y5=0.85
          Y6=1.01
          GOTO 19085
19083    Y0=2.17
          Y1=1.60
          Y2=1.10
          Y3=0.77
          Y4=0.60
          Y5=0.47
          Y6=0.38
          GOTO 19086

```

T-1566

```

.4I      COEFP
19084      Y0=1.08
           Y1=0.81
           Y2=0.78
           Y3=0.77
           Y4=0.57
           Y5=0.44
           Y6=0.13
           GOTO 19085
19085      Y0=-5.303E-12*SNS(I)**3+08.476E-8*SNS(I)**2-1.560E-4*SNS(I)+0.785
           Y1=-2.363E-12*SNS(I)**3+02.963E-8*SNS(I)**2+0.508E-4*SNS(I)+0.540
           Y2=+0.000E-12*SNS(I)**3-01.053E-8*SNS(I)**2+1.673E-4*SNS(I)+0.441
           Y3=+0.000E-12*SNS(I)**3-00.272E-8*SNS(I)**2+0.452E-4*SNS(I)+0.643
           Y4=+0.000E-12*SNS(I)**3+00.125E-8*SNS(I)**2-0.269E-4*SNS(I)+0.793
           Y5=+0.000E-12*SNS(I)**3+00.301E-8*SNS(I)**2-0.783E-4*SNS(I)+0.940
           Y6=+0.000E-12*SNS(I)**3+00.358E-8*SNS(I)**2-1.228E-4*SNS(I)+1.132
19086      J=3
           K=1
           GO TO 19210
C
C          H/A2 TURB Q/A   F 11
C
19090      IF(NSNS(I))19095,19095,19091
19091      IF(NSNS(I)-7600)19092,19093,19094
C
19092      Y0=0.63
           Y1=0.66
           Y2=0.69
           Y3=0.73
           Y4=0.79
           Y5=0.88
           Y6=1.01
           GOTO 19096
19093      Y0=-0.67
           Y1=-0.46
           Y2=-0.28
           Y3=-0.04
           Y4= 0.06
           Y5= 0.19
           Y6= 0.38
           GOTO 19096
19094      Y0=-2.20
           Y1=-1.55
           Y2=-1.14
           Y3=-0.86
           Y4=-0.60
           Y5=-0.28
           Y6= 0.13
           GOTO 19096
19095      Y0=+0.000E-12*SNS(I)**3-00.440E-8*SNS(I)**2-1.665E-4*SNS(I)+0.850
           Y1=+0.000E-12*SNS(I)**3-00.063E-8*SNS(I)**2-1.716E-4*SNS(I)+0.880
           Y2=+0.000E-12*SNS(I)**3+00.065E-8*SNS(I)**2-1.596E-4*SNS(I)+0.896
           Y3=+0.000E-12*SNS(I)**3-00.146E-8*SNS(I)**2-1.081E-4*SNS(I)+0.865
           Y4=+0.000E-12*SNS(I)**3-00.043E-8*SNS(I)**2-1.070E-4*SNS(I)+0.919
           Y5=+0.000E-12*SNS(I)**3+00.132E-8*SNS(I)**2-1.136E-4*SNS(I)+1.006
           Y6=+0.000E-12*SNS(I)**3+00.358E-8*SNS(I)**2-1.228E-4*SNS(I)+1.132

```

T-1566

```

.4I      COEFP
19096      J=3
           K=2
           GO TO 19200
C
C          B/Q2 TURB A/Q   F 10
C
19100      IF(NSNS(I))19105,19105,19101
19101      IF(NSNS(I)-7600)19102,19103,19104
C
19102      Y0=0.86
           Y1=0.81
           Y2=0.76
           Y3=0.59
           Y4=0.62
           Y5=0.54
           Y6=0.46
           GOTO 19106
19103      Y0=2.10
           Y1=1.57
           Y2=1.15
           Y3=0.83
           Y4=0.56
           Y5=0.31
           Y6=0.00
           GOTO 19106
19104      Y0=0.67
           Y1=0.43
           Y2=0.46
           Y3=0.63
           Y4=0.54
           Y5=0.18
           Y6=-0.15
           GOTO 19106
19105      Y0=-5.127E-12*SNS(I)**3+08.058E-8*SNS(I)**2-1.842E-4*SNS(I)+1.086
           Y1=-2.743E-12*SNS(I)**3+03.704E-8*SNS(I)**2-0.334E-4*SNS(I)+0.881
           Y2=+0.000E-12*SNS(I)**3-01.287E-8*SNS(I)**2+1.627E-4*SNS(I)+0.617
           Y3=+0.000E-12*SNS(I)**3-00.392E-8*SNS(I)**2+0.459E-4*SNS(I)+0.722
           Y4=+0.000E-12*SNS(I)**3-00.095E-8*SNS(I)**2-0.305E-4*SNS(I)+0.770
           Y5=+0.000E-12*SNS(I)**3+00.189E-8*SNS(I)**2-0.693E-4*SNS(I)+0.764
           Y6=+0.000E-12*SNS(I)**3+00.407E-8*SNS(I)**2-1.251E-4*SNS(I)+0.783
19106      J=3
           K=3
           GO TO 19200
C
C          B/A2 TURB Q/A   F 12
C
19110      IF(NSNS(I))19115,19115,19111
19111      IF(NSNS(I)-7600)19112,19113,19114
C
19112      Y0=-0.67
           Y1=-0.37
           Y2=-0.16
           Y3=-0.03
           Y4= 0.10
           Y5= 0.24

```

T-1566

```

.4I      COEFP
-----
          Y6= 0.45
          GOTO 19116
19113    Y0=-1.50
          Y1=-1.09
          Y2=-0.79
          Y3=-0.63
          Y4=-0.46
          Y5=-0.29
          Y6= 0.00
          GOTO 19116
19114    Y0=-2.33
          Y1=-1.58
          Y2=-1.30
          Y3=-1.10
          Y4=-0.84
          Y5=-0.55
          Y6=-0.15
          GOTO 19116
19115    Y0=+0.000E-12*SNS(I)**3-00.077E-8*SNS(I)**2-1.244E-4*SNS(I)-0.510
          Y1=+0.000E-12*SNS(I)**3+00.252E-8*SNS(I)**2-1.362E-4*SNS(I)-0.200
          Y2=+0.000E-12*SNS(I)**3+00.151E-8*SNS(I)**2-1.200E-4*SNS(I)+0.042
          Y3=+0.000E-12*SNS(I)**3+00.176E-8*SNS(I)**2-1.186E-4*SNS(I)+0.179
          Y4=+0.000E-12*SNS(I)**3+00.286E-8*SNS(I)**2-1.260E-4*SNS(I)+0.338
          Y5=+0.000E-12*SNS(I)**3+00.354E-8*SNS(I)**2-1.305E-4*SNS(I)+0.554
          Y6=+0.000E-12*SNS(I)**3+00.407E-8*SNS(I)**2-1.251E-4*SNS(I)+0.783
19116    J=3
          K=4
C
C
C  USE MATRIX TO CALCULATE COEF
C
C
19200    RS5 =(Y5-Y0)/(X6**6)
          RS5 =(Y5-Y0)/(X5**6)
          RS4 =(Y4-Y0)/(X4**6)
          RS3 =(Y3-Y0)/(X3**6)
          RS2 =(Y2-Y0)/(X2**6)
          RS1 =(Y1-Y0)/(X1**6)
C
          Q1 =1./X1
          Q2 =1./X2
          Q3 =1./X3
          Q4 =1./X4
          Q5 =1./X5
          Q6 =1./X6
C
          R1 =Q1*Q1
          R2 =Q2*Q2
          R3 =Q3*Q3
          R4 =Q4*Q4
          R5 =Q5*Q5
          R6 =Q6*Q6
C
          S1 =Q1*R1
          S2 =Q2*R2

```

T-1566

## 5.41 COEFP

$$S3 = Q3 * R3$$

$$S4 = Q4 * R4$$

$$S5 = Q5 * R5$$

$$S6 = Q6 * R6$$

C

$$T1 = Q1 * S1$$

$$T2 = Q2 * S2$$

$$T3 = Q3 * S3$$

$$T4 = Q4 * S4$$

$$T5 = Q5 * S5$$

$$T6 = Q6 * S6$$

C

$$U1 = Q1 * T1$$

$$U2 = Q2 * T2$$

$$U3 = Q3 * T3$$

$$U4 = Q4 * T4$$

$$U5 = Q5 * T5$$

$$U6 = Q6 * T6$$

C

C

$$RS36 = (RS6 - RS1) / (Q6 - Q1)$$

$$RS35 = (RS5 - RS1) / (Q5 - Q1)$$

$$RS34 = (RS4 - RS1) / (Q4 - Q1)$$

$$RS33 = (RS3 - RS1) / (Q3 - Q1)$$

$$RS32 = (RS2 - RS1) / (Q2 - Q1)$$

C

$$R61 = (R6 - R1) / (Q6 - Q1)$$

$$R51 = (R5 - R1) / (Q5 - Q1)$$

$$R41 = (R4 - R1) / (Q4 - Q1)$$

$$R31 = (R3 - R1) / (Q3 - Q1)$$

$$R21 = (R2 - R1) / (Q2 - Q1)$$

C

$$S61 = (S6 - S1) / (Q6 - Q1)$$

$$S51 = (S5 - S1) / (Q5 - Q1)$$

$$S41 = (S4 - S1) / (Q4 - Q1)$$

$$S31 = (S3 - S1) / (Q3 - Q1)$$

$$S21 = (S2 - S1) / (Q2 - Q1)$$

C

$$T61 = (T6 - T1) / (Q6 - Q1)$$

$$T51 = (T5 - T1) / (Q5 - Q1)$$

$$T41 = (T4 - T1) / (Q4 - Q1)$$

$$T31 = (T3 - T1) / (Q3 - Q1)$$

$$T21 = (T2 - T1) / (Q2 - Q1)$$

C

$$U61 = (U6 - U1) / (Q6 - Q1)$$

$$U51 = (U5 - U1) / (Q5 - Q1)$$

$$U41 = (U4 - U1) / (Q4 - Q1)$$

$$U31 = (U3 - U1) / (Q3 - Q1)$$

$$U21 = (U2 - U1) / (Q2 - Q1)$$

C

C

$$RS46 = (RS36 - RS32) / (R61 - R21)$$

$$RS45 = (RS35 - RS32) / (R51 - R21)$$

$$RS44 = (RS34 - RS32) / (R41 - R21)$$

$$RS43 = (RS33 - RS32) / (R31 - R21)$$

T-1566

5.4I

COEFP

C

$$S52 = (S61 - S21) / (R61 - R21)$$

$$S52 = (S51 - S21) / (R51 - R21)$$

$$S42 = (S41 - S21) / (R41 - R21)$$

$$S32 = (S31 - S21) / (R31 - R21)$$

C

$$T62 = (T61 - T21) / (R61 - R21)$$

$$T52 = (T51 - T21) / (R51 - R21)$$

$$T42 = (T41 - T21) / (R41 - R21)$$

$$T32 = (T31 - T21) / (R31 - R21)$$

C

$$U62 = (U61 - U21) / (R61 - R21)$$

$$U52 = (U51 - U21) / (R51 - R21)$$

$$U42 = (U41 - U21) / (R41 - R21)$$

$$U32 = (U31 - U21) / (R31 - R21)$$

C

C

$$RS56 = (RS45 - RS43) / (S62 - S32)$$

$$RS55 = (RS45 - RS43) / (S52 - S32)$$

$$RS54 = (RS44 - RS43) / (S42 - S32)$$

C

$$T53 = (T62 - T32) / (S62 - S32)$$

$$T53 = (T52 - T32) / (S52 - S32)$$

$$T43 = (T42 - T32) / (S42 - S32)$$

C

$$U63 = (U62 - U32) / (S62 - S32)$$

$$U53 = (U52 - U32) / (S52 - S32)$$

$$U43 = (U42 - U32) / (S42 - S32)$$

C

C

$$RS66 = (RS56 - RS54) / (T63 - T43)$$

$$RS65 = (RS55 - RS54) / (T53 - T43)$$

C

$$U64 = (U63 - U43) / (T63 - T43)$$

$$U54 = (U53 - U43) / (T53 - T43)$$

C

C

C CALC COEF

C

$$G = Y0$$

C

$$F = (RS66 - RS65) / (U64 - U54)$$

C

$$E = RS65 - F * U54$$

C

$$D = RS54 - F * U43 - E * T43$$

C

$$C = RS43 - F * U32 - E * T32 - D * S32$$

C

$$B = RS32 - F * U21 - E * T21 - D * S21 - C * R21$$

C

$$A = RS1 - F * U1 - E * T1 - D * S1 - C * R1 - B * Q1$$

IF (J-1) 19300, 19300, 19330

19300 IF (K-1) 19500, 19500, 19310

19310 IF (K-2) 19300, 19520, 19320

T-1566

## 5.4I COEFP

19320 IF (K-3) 19310, 19540, 19560

C

19330 IF (J-2) 19300, 19340, 19370

19340 IF (K-1) 19500, 19500, 19350

19350 IF (K-2) 19500, 19520, 19360

19360 IF (K-3) 19350, 19540, 19560

C

19370 IF (K-1) 19500, 19500, 19380

19380 IF (K-2) 19370, 19520, 19390

19390 IF (K-3) 19380, 19540, 19560

19500

AHQP (I, J) = A

BHQP (I, J) = B

CHQP (I, J) = C

DHQP (I, J) = D

EHQP (I, J) = E

FHQP (I, J) = F

GHQP (I, J) = G

IF (J-1) 19010, 19010, 19510

19510 IF (J-2) 19010, 19050, 19090

C

19520 AHAP (I, J) = A

BHAP (I, J) = B

CHAP (I, J) = C

DHAP (I, J) = D

EHAP (I, J) = E

FHAP (I, J) = F

GHAP (I, J) = G

IF (J-1) 19020, 19020, 19530

19530 IF (J-2) 19020, 19060, 19100

C

19540 ABQP (I, J) = A

BBQP (I, J) = B

CBQP (I, J) = C

DBQP (I, J) = D

EBQP (I, J) = E

FBQP (I, J) = F

GBQP (I, J) = G

IF (J-1) 19030, 19030, 19550

19550 IF (J-2) 19030, 19070, 19110

C

19560 ABAP (I, J) = A

BBAP (I, J) = B

CBAP (I, J) = C

DBAP (I, J) = D

EBAP (I, J) = E

FBAP (I, J) = F

GBAP (I, J) = G

IF (J-1) 19040, 19040, 19570

19570 IF (J-2) 19040, 19080, 19580

19580 CONTINUE

DO 19589 J=1, 3

WRITE (61, 19582) AHQP (I, J), BHQP (I, J), CHQP (I, J), DHQP (I, J),

1 EHQP (I, J), FHQP (I, J), GHQP (I, J)

WRITE (61, 19583) AHAP (I, J), BHAP (I, J), CHAP (I, J), DHAP (I, J),

1 EHAP (I, J), FHAP (I, J), GHAP (I, J)

T-1566

5.4I COEFP

```
1 WRITE (61,19583) ABQP (I, J), BBQP (I, J), CBQP (I, J), DBQP (I, J),  
EBQP (I, J), FBQP (I, J), GBQP (I, J)
```

```
1 WRITE (61,19583) ABAP (I, J), BBAP (I, J), CBAP (I, J), DBAP (I, J),  
EBAP (I, J), FBAP (I, J), GBAP (I, J)
```

```
19582 FORMAT (/7E12.5)
```

```
19583 FORMAT (7E12.5)
```

```
19589 CONTINUE
```

```
RETURN
```

```
END
```

```
MULTI-BANK COMPILATION.
```