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A STUDY OF THE  
THERMAL CONDUCTIVITY AND THERMAL DIFFUSIVITY  
OF SANDSTONE RESERVOIR ROCKS UNDER A VACUUM

1969

BY  
HOMER J. BOYD

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A thesis submitted to the Faculty and the Board of Trustees of the Colorado School of Mines in partial fulfillment of the requirements for the degree of Master of Science in Petroleum Engineering.

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ABSTRACT

Investigations were made to determine the thermal conductivity (k) and thermal diffusivity (H) of porous arkosic sandstones. These thermal properties were determined and illustrated as functions of temperature, porosity, and permeability. Correlations were obtained between the thermal properties and porosity.

The tests were conducted under a pressure of less than 10 microns and a temperature range of 70° F. to 600° F. Time and temperature at pre-selected stations were the parameters measured during an experimental run.

Two standard materials, Coors' porcelain AD-85 and a satin surface quartz rod, were subjected to the same experimental procedure as were sandstone specimens to establish a check on the results obtained.

This experimental work was performed to determine k and H of a porous rock in the absence of a saturating fluid, to establish an effective and relatively easy procedure for determining the thermal properties (k, H) of a porous medium under a vacuum, and to determine if a correlation between thermal and physical properties of the sandstone existed.

The results of this work proved that the experimental procedure was an effective way for determining the above mentioned thermal properties. A correlation between the thermal properties and porosity has been shown to exist. The correlation of thermal properties with permeability is, however, dependent upon a correlation between permeability and porosity (Figure 53).

Included in the appendix are two IBM 360 computer programs written in Fortran IV language, a derivation of the equations used for the experimental procedure, geological description of specimens, sample heating and cooling curves, equations for steady-state surface temperature profiles, and a tabulation of porosity, permeability, bulk density, thermal conductivity and thermal diffusivity.

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INTRODUCTION

During the past decade the possibility of oil recovery through thermal stimulation became an economic reality. Prior to 1950 two insitu combustion test projects were initiated in the Childress and Southern Oklahoma fields near Oklahoma City. In 1956 a major oil company began field experiments with steam injection in the Coalinga area of California. At about the same time, another major company started an insitu combustion project in Illinois.

Since 1956 the interest in thermal stimulation as an oil recovery technique has continued to increase. This increased interest is confirmed by the initiation of thermal recovery projects in the San Juaquin Valley, Ventura Basin, and Los Angeles Basin of California, in addition to areas of Texas, Illinois, Indiana, and South America.

The initiation of any thermal stimulation process raises several problems. For steam injection projects, the heat loss from surface to the hydrocarbon-bearing formation must be evaluated, as must the heat loss to adjacent formations. Insitu combustion processes are concerned primarily with the heat loss to adjacent formations. Regardless of the thermal stimulation process used, an estimate of the effectiveness of the heat within the hydrocarbon-bearing formation



must be made, i. e., for a given amount of heat input, how much rock and fluid will be heated and to what temperature?

It is obvious at this point that thermal properties of the solid and fluid within the porous medium must be known to make the necessary calculations. A survey of the literature revealed no information on the thermal properties of the solid part of the porous medium, thus the primary purpose of this investigation.

Thermal conductivity has been the subject of many research projects, the majority of which have little application to the evaluation of thermal oil recovery processes. A large portion of the published work consists of the determination of thermal conductivity of materials with different chemical composition and geometrical shapes (5, 8, 9, 17).

Birch and Clark (3) were two of the earlier investigators of the thermal properties of rocks. These writers illustrated the dependency of thermal conductivity upon temperature and chemical composition. Several rock samples (igneous and sedimentary) were subjected to temperatures ranging from 0° C. to 500° C. at atmospheric pressure.

Zierfuss and van der Vliet (26) made a significant contribution with their work on thermal conductivity in sedimentary rocks. Fifty samples of different composition, permeability, porosity, and formation resistivity (FRF) were subjected

to temperatures of 0° C. to 300° C. at atmospheric pressure in the presence of a saturating fluid. The results of their studies showed the effect of using various saturating fluids upon the thermal conductivity of the rock-fluid system. The effects of chemical composition of the rock upon thermal conductivity was illustrated. The product of FRF and porosity suggested a correlation with thermal conductivity.

Somerton (21) was one of the first to conduct research on the thermal properties of sedimentary rocks with the purpose of obtaining data used in thermal oil recovery calculations. In this work, ten samples of sandstone, silty sand, siltstone and/or shale were subjected to temperatures ranging from 90° F. to 980° F. These experiments were conducted with the specimens saturated with fluids of known thermal properties at pressures of 14.7, 500, 1500, or 3000 psia. The results of Somerton's work showed the dependency of heat capacity (cp), thermal conductivity (k), and diffusivity (H) upon temperature and pressure.

The determination of thermal diffusivity was the main objective of Somerton and Boozer's (22) research. Samples of sandstone were subjected to temperatures ranging from 200° F. to 1800° F. in the presence of a saturating fluid at atmospheric pressure. The results of this research showed the dependency of thermal diffusivity, conductivity, and heat capacity upon temperature.

The previously discussed literature dealt with the thermal properties of rocks filled with a saturating fluid. Schotte (19) showed the influence of the presence of a saturating fluid (gas) on the laboratory measured thermal properties. Kunii and Smith (15) also studied the thermal conductivity of sandstone samples saturated with a fluid. This latter work was the basis for another publication by Kunii and Smith (16) in which the thermal conductivity of the saturated sample was measured. This research was performed at pressures of 0.039 to 4.00 psia. The samples were saturated with either n-heptane, methyl alcohol, water, or a water-glycerol solution. Measurements were taken with the saturated specimen in a gas environment of helium, nitrogen, air, or carbon dioxide. The permeability of the samples ranged from 18 to 590 md.

The results of this work were used to evaluate a theoretical equation developed earlier (15) to predict the thermal conductivity of a rock filled with a fluid of known thermal properties.

The equation developed in (15) and evaluated in (16) is

$$\frac{K_e}{K_s} = \phi' \left( \frac{K_g}{K_s} \right) + \frac{(1-\phi') \left( 1 + \frac{\phi'}{\phi} a \right)}{1 + \frac{\phi'/\phi}{\frac{1}{a} \left( \frac{K_g}{K_s} \right) + \frac{D_p H_p}{K_s}} .$$

in which  $k_e^o$ ,  $k_s$ , and  $k_g$  are the thermal conductivity of the saturated specimen, solid portion of the rock only, and the gas environment, respectively.  $\phi$  is the porosity of the unconsolidated packed beds, and  $\phi'$  the porosity of sandstone.

Prior to evaluating the above equation,  $k_s$  must be determined. As far as the writer has been able to determine, no direct determinations of thermal conductivity and diffusivity of a porous rock in the absence of a saturating fluid have been made. A portion of the literature surveyed illustrates the effect of a saturating fluid upon the heat conduction through a porous medium. The equations developed for the determinations of  $k$  and  $H$  in a saturated porous medium require the thermal conductivity of the solid portion of the rock.

The following data were presented (16) to show the effect of a saturating fluid upon thermal conductivity and to illustrate the importance of being able to determine the thermal conductivity of the solid portion of the rock.

$k_e^o$	$k_s$
4.40 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R	9.40 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R
3.45 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R	18.10 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R
4.05 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R	23.00 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R
6.10 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R	20.00 BTU/ft <sup>2</sup> /In/Min/ <sup>o</sup> R

where  $k_e^o$  and  $k_s$  are as previously defined.

EXPERIMENTAL PROCEDURE

Twenty-six sandstone specimens, one inch in diameter, from the Stevens, Refugain, and Eocene sands of California were selected for this research on the basis that the specimens represent an acceptable range of porosity (5-30%) and permeability (0-300 md). Prior to experimentation each specimen was indexed at lengths of 1, 1-1½, or 2 centimeters. The specimens were then placed into an oven, heated to above 400° F. for a minimum for two days, and placed in a desiccator for storage.

Thermocouples were placed on the exterior and the heat source attached to one end of the prepared specimen prior to insertion into the vacuum chamber (see Figure 1).

The heat source was generated by applying a constant current of one, two, or three amps to sixteen gage nichrome wire which was wound around, but electrically insulated from, the copper heating element.

Two separate but similar set-ups were used to obtain the reported data. All sandstone specimen were run using a sixteen point recorder while the standard materials were run utilizing a twenty-four point recorder. All other components of the experimental equipment were identical.

Table I gives heating element wire data and energy inputs for the three heating elements used during this research. Also presented are pertinent data on the two standard materials used.

An experimental run was accomplished according to the following steps:

1. Determination of heat capacity as a function of temperature;
2. Positioning of the specimen inside the plexiglas vacuum chamber;
3. Evacuation of the vacuum chamber to less than 10 microns pressure;
4. Application of the current to the heat source and monitoring of the surface temperature with a 16 or 24 point recorder (galvanometer) using Chromel - alumel thermocouples;
5. Maintaining of heat source until the temperature along the specimen has reached a steady-state, i.e., the temperature at any point ceases to increase;
6. Shutting off of power to heating element and monitoring temperature profile for several minutes;
7. Evaluation of the equation

$$D/K = f(T_{aa}) = \frac{A \left( \frac{dt}{dx} \Big|_{x_2} - \frac{dt}{dx} \Big|_{x_1} \right)}{\sigma A_s (T_{aa}^4 - T_s^4)}$$

8. Plotting  $D/k$  as a function of temperature ( $T_{aa}$ ) is used to exclude that portion of the curve that exhibits end effects;

9. Using  $f(T_{aa})$  as determined in (7) to evaluate

$$K = \frac{Mc_p(T_{ae2} - T_{ae1})/\Delta\theta}{A \left( \frac{dT}{dx} \Big|_{x2} - \frac{dT}{dx} \Big|_{x1} \right) - \sigma A_s f(T_{aac})(T_{aac}^4 - T_s^4)}$$

by using program "Thermal Conductivity Coeff Calculation";

10. Calculation of thermal diffusivity by the following equation:

$$H = k/c_p \rho$$

A derivation of the equations in Step 7 and 9 are included in the appendix. Also presented is an alternate method for determining  $H$  from the transient data.

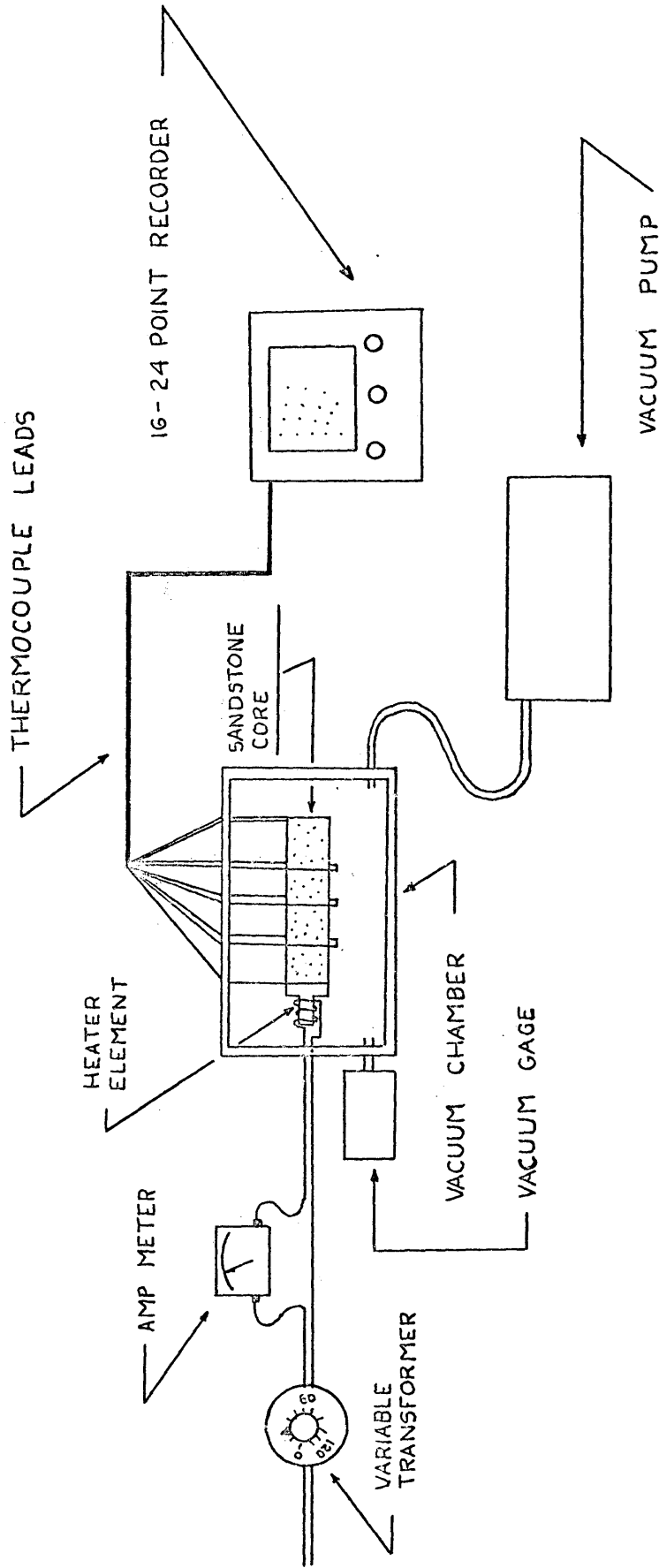


FIGURE 1 LABORATORY EQUIPMENT SET-UP



TABLE 1

## HEATING ELEMENT DATA

<u>HEATER</u>	<u>WIRE LENGTH</u>	<u>TYPE WIRE</u>	<u>RESISTANCE</u>
1	37"	Nichrome 16 gage	1.8 ohms/ft.
2	33"	Nichrome 16 gage	1.8 ohms/ft.
3	29"	Chromel 16 gage	1.04 ohms/ft.

## ENERGY INPUT TO HEATER ELEMENT

<u>HEATER</u>	<u>INPUT CURRENT</u>	<u>WATTS</u>	<u>BTU/MIN</u>
1	1 amp.	5.40	0.307
	2 "	21.60	1.229
	3 "	48.60	2.764
2	1 "	4.95	0.282
	2 "	19.80	1.126
	3 "	44.55	2.534
3	1 "	2.51	0.143

TABLE 1 - - ENERGY INPUT TO HEATER ELEMENT (Continued)

<u>Heater</u>	<u>Input Current</u>	<u>Watts</u>	<u>BTU/MIN</u>
3	2 amp	10.04	0.571
	3 "	22.59	1.285

THERMAL PROPERTIES OF STANDARD MATERIALS  
(Coors Porcelain AD-85)

$C_p$ @ 672°R	Diameter	$k$
0.22 BTU/lbm-°R	1.06 inches	1.691 BTU/in/ft <sup>2</sup> /min/°R @ 528°R
		1.403 " " " @ 672°R
		0.774 " " " @1212°R

(Satin Surface Quartz Rod)

$C_p$ @ 672°R	Diameter	$k$
0.219 BTU/lbm-°R	1.00 inches	1.596 BTU/in/ft <sup>2</sup> /min/°R @ 672°R

### DISCUSSION OF RESULTS

Steady-state surface temperature profiles for the test specimens from which the radiation-conduction ratio,  $D/k$ , was determined are plotted on Figures 7 through 36. Evaluation of any of the equations (Table 2) of the measured steady-state temperature profiles indicates that the temperature profiles can be represented by a second order equation, thereby satisfying one of the restrictions placed on the use of equation (5a) of the appendix.

$D/k$  was presented graphically as a function of temperature in Figure 37. The graph shows that as the temperature increases, the  $D/k$  factor decreases. The inverse proportionality between  $D/k$  and temperature was expected due to the change in the denominator with respect to  $T_{aa}$  of equation (1b) of the appendix.

Figures 39 through 48 illustrate the dependency of thermal conductivity and diffusivity upon temperature. This phenomenon was expected and has been published by several authors (21). It is interesting to note that above about 200° F. the thermal properties cease to change significantly with additional increases in temperature.

A correlation between thermal conductivity and diffusivity exists with porosity as is illustrated by Figures 49 and 50.

The correlation of these two thermal properties with porosity is explained by examining tortuosity as a function of porosity.

In this research the thermal conductivity has been defined as

$$K = A \Delta T / \Delta X \quad (1)$$

in which  $A = \pi d^2/4$  and  $\Delta X$  is the straight line distance over which  $\Delta T$  occurs, however, the thermal conductivity of the solid portion of the rock,  $k_p$ , is

$$K_r = \frac{A(1-\phi)\Delta T}{\Delta X(\gamma)} \quad (2)$$

in which  $1-\phi$  compensates for the reduction of cross-sectional area due to porosity and  $\gamma$ , the tortuosity, accounts for the additional length in the flow path due to porosity.

Now,

$$\frac{K_r}{K} = \frac{\frac{A(1-\phi)\Delta T}{\Delta X(\gamma)}}{\frac{A\Delta T}{\Delta X}} = \frac{1-\phi}{(\gamma)} \quad (3)$$

then 
$$K = \frac{K_r(\gamma)}{(1-\phi)} \quad (3a)$$

Figure 2 illustrates how the measured thermal conductivity will vary as a function of tortuosity, thus as a function of

porosity. Using boundary conditions of  $\phi = 0, \tau = 1 - \phi, \tau = 1,$  or  $\tau =$  non-linear function of porosity will establish the following curves.

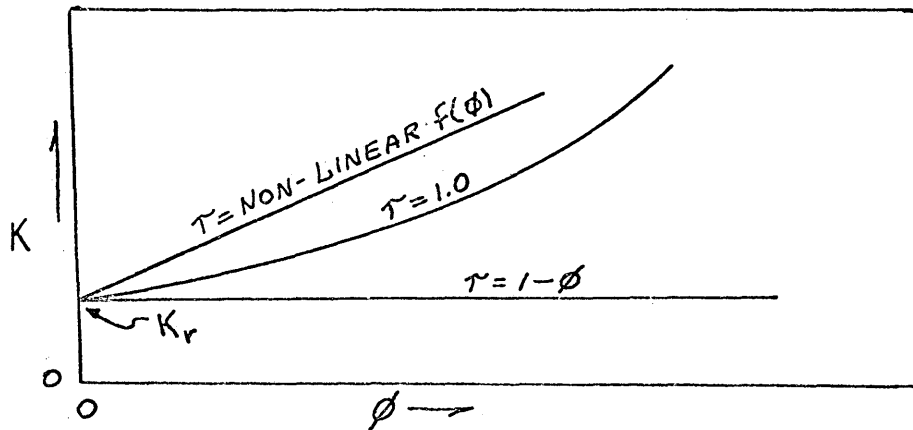


Figure 2. Measured  $k$  as a function of porosity.

From equation (3a) it is evident that when porosity ( $\phi$ ) is zero  $k = k_r \tau$ . It is further noted that when  $\tau = 1 - \phi$ , the thermal conductivity ( $k$ ) of the porous medium is independent of porosity. If the tortuosity ( $\tau$ ) is unity then  $k$  would approach infinity as the porosity approaches unity. Finally if  $\tau$  is some non-linear function of porosity,  $k$  will increase at some constant rate with respect to porosity or

$$k = k_r + m\phi \quad (4)$$

in which  $m$  is the slope of the straight line.

Substituting equation (3a) in equation (4) gives

$$K_r \tau / (1 - \phi) = K_r + m\phi \quad (5)$$

Solving for  $\tau$  yields

$$\tau = (1 + m\phi/K_r)(1 - \phi) = 1 + (m/K_r - 1)\phi - m\phi^2/K_r \quad (6)$$

Using equation (6) in (3a) gives

$$\begin{aligned}
 K &= K_r \gamma / (1 - \phi) = K_r [1 + (m/K_r - 1)\phi - (m/K_r)\phi^2] / (1 - \phi) \quad (7) \\
 &= [K_r + (m - K_r)\phi - m\phi^2] / (1 - \phi)
 \end{aligned}$$

which establishes the correlation of thermal conductivity with porosity.

Specimens S-3, S-4, and S-7 display a slightly higher value of  $k$  than does S-1A, S-2A, S-8, and S-9. The difference in  $k$  of these two sets of specimens is natural owing to the cementing material of the specimens. S-3, S-4, and S-7 are cemented with calcite while the other specimens are cemented primarily with montmorillonite. The effect of chemical composition on thermal conductivity and diffusivity has been previously illustrated (21, 22).

Figure 53 shows the relationship between permeability and porosity for the specimens used in this work.

The thermal conductivity measured for the standard materials was plotted as a function of temperature and is presented on Figures 39 and 40. Comparison of the measured values of thermal conductivity with the published data indicates a discrepancy of about 40% at 200° F. The same comparison at 100° F. indicates better agreement between the two values, or about 30%

disagreement. Better agreement at lower temperatures was expected because the measured temperature more closely approached the true surface temperature of the specimen.

The discrepancy exists for three reasons:

1. The center-line temperature was not used in the determination of  $k$ ;
2. Thermocouples were not uniformly attached to the specimens;
3. Bare thermocouple wire was permitted near the thermocouple junction, allowing heat loss from the wire by radiation.

Figure 3 shows variations in temperature measurements obtained due to different methods used to attach thermocouples to the surface of an iron pipe.

The following example illustrates the effect of using surface temperature measurement as opposed to center-line temperature.

$$\text{Let } T = 189.0 - 35.4X - 3.02X^2$$

$$\text{then } dt/dx = - 35.4 - 6.04X$$

The surface temperature at  $X = 1$  and  $X = 2$  is  $156^\circ$  F. and  $130^\circ$  F. for a  $\Delta T$  of 26 degrees. Using  $dt/dx$  at  $X = 1$  and  $X = 2$  gives center-line temperatures of 185.8 and 153.7 for a  $\Delta T$  of 32.1. The discrepancy between the measured thermal conductivity  $k$  and actual thermal conductivity  $k_a$  due to using surface temperatures will be  $K = (k_a)(26/32)$

Of the data in the literature, none are appropriate for comparison with the results obtained in this research. Kunii and Smith (16) reported thermal conductivity for the solid material of a packed bed, however, the temperature at which  $k$  was measured was not given.



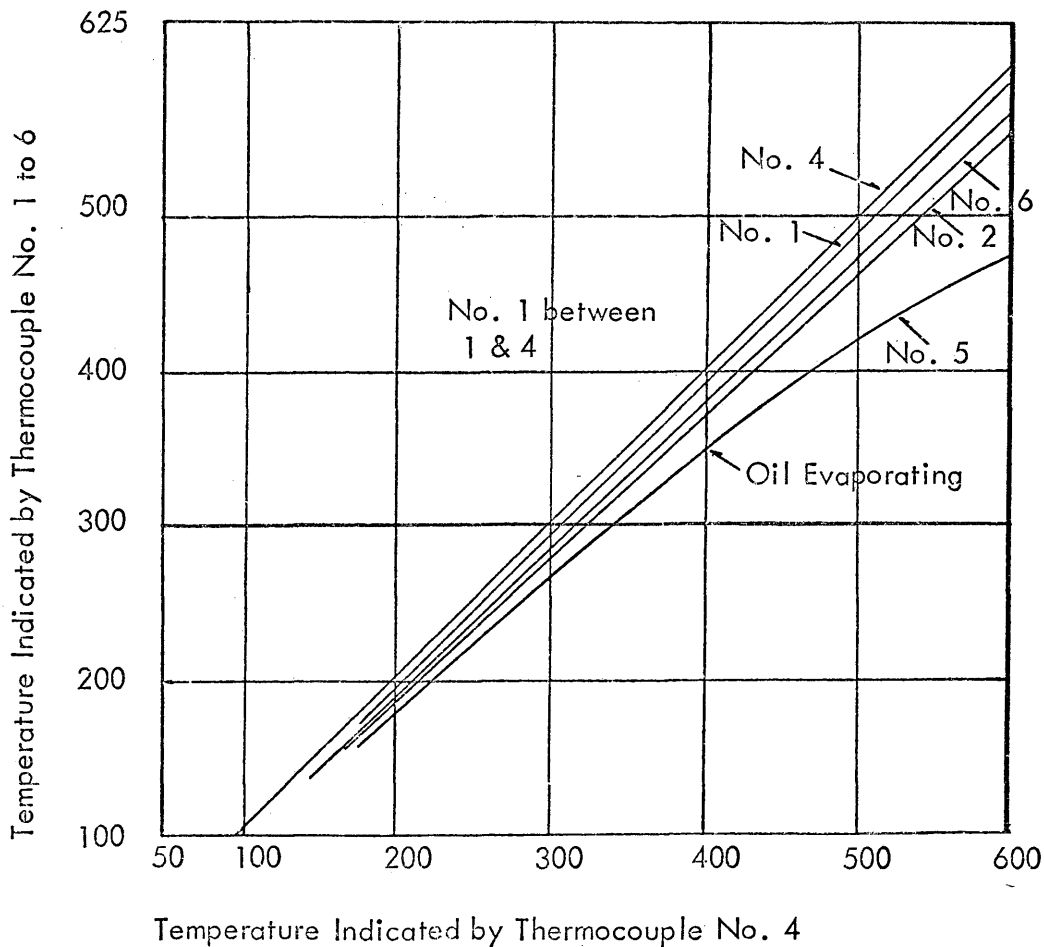
FIGURE 3

Comparison of Methods of Attaching Thermocouples to the Surface of an Iron Pipe.

1. Peened into hole in rust covered surface.
2. Peened into hole in brazed-over surface.
3. Brazed to surface.
4. Brazed to surface - thermocouple wire wrapped around pipe once.
5. In an oil filled hole in rust covered surface.
6. Peened into hole in rust covered surface.

All thermocouple brazed junction, copper-constantan #24

wire.



CONCLUSIONS

1. The experimental procedure used in this study is an effective way to determine thermal conductivity and thermal diffusivity.
2. A correlation between thermal conductivity and porosity does exist and has been established.
3. Thermal conductivity and thermal diffusivity of arkosic sandstones become relatively small at temperatures in excess of 200° F.
4. Additional work should be done in this area on other sedimentary rocks.

RECOMMENDATIONS

1. Obtain a pressure of 10 microns or less inside the vacuum chamber to render heat loss by convection negligible.
2. Calibrate the temperature recorder over the expected operating temperature range.
3. Insure good thermal contact between heater element and specimen.
4. Attach thermocouples uniformly to specimen and cement into position.
5. Wrap thermocouples around the specimens about two times, following an isothermal plane.

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DERIVATION OF THE ONE DIMENSIONAL STEADY-STATE  
AND TRANSIENT HEAT CONDUCTION EQUATION

Before the specific problem at hand is discussed the differential equation governing the temperature distribution in a body will first be derived; then by making certain simplifying assumptions the equation will be reduced into a form applicable for analytical, graphical, or numerical solutions.

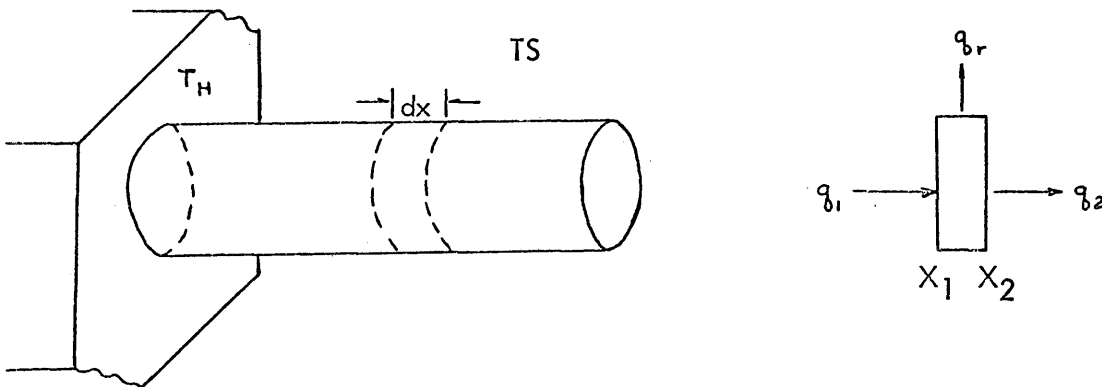


Figure 4. Sketch and nomenclature for a rod protruding from a constant heat source.

Assume a rod to have constant cross-sectional area, surface area, specific heat, and assume that the conductivity  $k$ , diffusivity  $H$  and the emissivity  $e$ , are constant over small differences in temperature. Suppose the rod lies along

the X axis and consider the element of volume bounded by  $X_1$  and  $X_2$  (Figure 4a).

Under steady-state conditions the rate of heat flow into the element must equal the rate of heat flow out of the element, or:

Rate of heat flow by conduction into = the element at $X_1$	Rate of heat flow by conduction out of the + element at $X_2$	Rate of heat flow by radia- tion from the surface between $X_1$ and $X_2$
---	---	---

Symbolically the equation becomes:

$$-KA \left. \frac{dt}{dx} \right|_{X_1} = -KA \left. \frac{dt}{dx} \right|_{X_2} + \sigma Fe A_s (T_{aa}^4 - T_s^4) \quad (1)$$

Rearranging and solving for unknown quantities gives:

$$KA \left( \left. \frac{dt}{dx} \right|_{X_2} - \left. \frac{dt}{dx} \right|_{X_1} \right) = \sigma Fe A_s (T_{aa}^4 - T_s^4) \quad (1a)$$

thus

$$Fe/K = \frac{A \left( \left. \frac{dt}{dx} \right|_{X_2} - \left. \frac{dt}{dx} \right|_{X_1} \right)}{\sigma A_s (T_{aa}^4 - T_s^4)} \quad (1b)$$

where  $F$  = shape factor  
 $e$  = emissivity coefficient  
 $\sigma$  =  $2.868 \times 10^{-11}$  BTU/min - ft<sup>2</sup>°R (Stefan-Boltzmann/Constant.)  
 $A$  = cross-sectional area of rod  
 $A_s$  = surface area of rod between  $X_1$  and  $X_2$   
 $T_{aa}$  = effective radiating surface temperature between  $X_1$  and  $X_2$  and defined mathematically as

$$T_{aa} = \left( \frac{T_{X_1}^4 + T_{X_2}^4}{2.0} \right)^{0.25}$$

(From Steady-State Temperature)

$T_s$  = temperature of the medium into which the heat is radiating (plexiglas vacuum chamber).

This, however, leaves three unknowns in one equation. It would be possible to set  $Fe = D$  and solve the equations simultaneously; however, since  $k$  is in fact a function of temperature, uniqueness of the solution would not be guaranteed. A solution can be obtained, however, in the following manner if cooling curves are available.

Consider the following temperature distribution curves for steady-state and cooling environments.

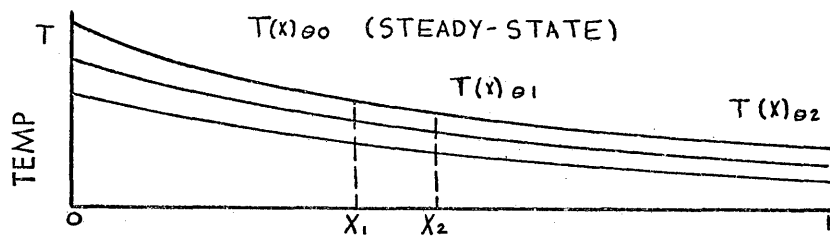


Figure 5. Temperature profiles along a rod.

Again considering the elemental volume bounded by  $X_1$  and  $X_2$ , the change in internal energy can be expressed as:

$$\Delta U = U_{\theta_2} - U_{\theta_1} = mc_p(T_{a\theta_2} - T_{a\theta_1}) \quad (2)$$

Which must also equal the heat flow into the element less the heat loss from the element, or:

$$\Delta U = q_1 - q_2 - q_3 \quad (3)$$

Substitution of equation 1 into 3 gives:

$$\Delta U = \left[ -KA \frac{dt}{dx} \Big|_{x_1} + KA \frac{dt}{dx} \Big|_{x_2} - \sigma DA_s (T_{ad}^4 - T_s^4) \right] \Delta \theta \quad (3a)$$

Now substituting equation 3a into equation 2 gives:

$$m c_p (T_{a\theta_2} - T_{a\theta_1}) / \Delta\theta = K A \left( \frac{dt}{dx} \Big|_{x_2} - \frac{dt}{dx} \Big|_{x_1} \right) - \sigma D A_s (T_{aac}^4 - T_s^4) \quad (4)$$

now from equation 1b let  $f(T_{aa})$  equal

$$\frac{A \left( \frac{dt}{dx} \Big|_{x_2} - \frac{dt}{dx} \Big|_{x_1} \right)}{\sigma A_s (T_{aa}^4 - T_s^4)} \quad (1c)$$

then

$$D = K f(T_{aa}) \quad (1d)$$

substituting equation 1d into equation 4 gives:

$$m c_p (T_{a\theta_2} - T_{a\theta_1}) / \Delta\theta = K \left[ A \left( \frac{dt}{dx} \Big|_{x_2} - \frac{dt}{dx} \Big|_{x_1} \right) - \sigma A_s f(T_{aa}) (T_{aac}^4 - T_s^4) \right] \quad (5)$$

Rearranging equation 5 and solving for k yields:

$$K = \frac{m c_p (T_{a\theta_2} - T_{a\theta_1}) / \Delta\theta}{A \left( \frac{dt}{dx} \Big|_{x_2} - \frac{dt}{dx} \Big|_{x_1} \right) - \sigma A_s f(T_{aa}) (T_{aac}^4 - T_s^4)} \quad (5a)$$

Thus, thermal conductivity is expressed as a function of the temperature  $T_{aac}$ . Particular caution must be applied in obtaining the various numerical values for substitution into the above equation; thus the following explanations:

- $m$  = mass of elemental volume (lbm)
- $C_p$  = specific heat (BTU/lbm-°R)
- $T_{a\theta_2}$  = average temperature between  $X_1$  and  $X_2$  at time two or

$$T_{a\theta_2} = (T_{x_1, \theta_2} + T_{x_2, \theta_2}) / 2.0$$

- $T_{a\theta_1}$  = same as  $T_{a\theta_2}$  except for time one (°R)



$\Delta \theta$  = time two less time one (min)  
 $A$  = cross-sectional area of rod (ft<sup>2</sup>)  
 $A_s$  = surface area between  $X_1$  and  $X_2$  (ft<sup>2</sup>)  
 $\sigma$  = as previously defined

$\frac{dt}{dx}|_{x_n}$  = average rate of change in temperature

with respect to length and is obtained algebraically as:

$$\frac{dt}{dx}|_{x_n} = \left( \frac{dt}{dx}|_{x_n, \theta_i} - \frac{dt}{dx}|_{x_n, \theta_{i+1}} \right) / 2.0$$

$T_{aac}$  = average effective radiating temperature while specimen is cooling

$$= \frac{\left( \frac{T_{x_n, \theta_i}^4 + T_{x_{n+1}, \theta_i}^4}{2.0} \right)^{0.25} + \left( \frac{T_{x_n, \theta_{i+1}}^4 + T_{x_{n+1}, \theta_{i+1}}^4}{2.0} \right)^{0.25}}{2.0}$$

$f(T_{aac})$  = as defined previously but evaluated at  $T_{aac}$   
 °R/in

DERIVATION OF THE UNSTEADY-STATE  
THREE DIMENSIONAL HEAT CONDUCTION EQUATION

In order to obtain an expression for thermal diffusivity,  $H$ , it will be necessary to expand the one dimensional steady-state heat flow equation into a three dimensional unsteady-state condition, thus resulting in an equation for the temperature distribution within the rod.

Consider a small rectangular elemental volume of material in a solid body as shown in Figure 6.

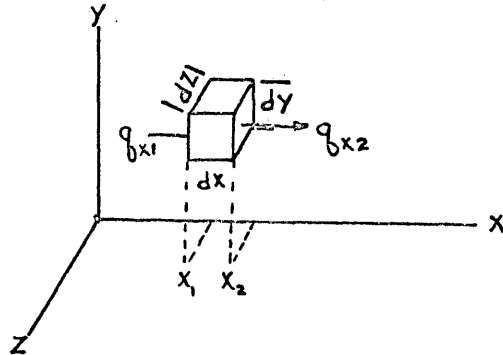


Figure 6. Sketch and nomenclature for three-dimensional heat flow.

The energy balance for the elemental volume during time  $d\theta$  may be expressed as:

$$\begin{array}{l} \text{Heat flow into} \\ \text{element during} \\ d\theta \end{array} + \begin{array}{l} \text{Heat generated} \\ \text{within element} \\ \text{by internal source} \\ \text{during } d\theta \end{array} = \begin{array}{l} \text{Heat flow out} \\ \text{of element} \\ \text{during } d\theta \end{array} + \begin{array}{l} \text{Change in in-} \\ \text{ternal energy of} \\ \text{element during} \\ d\theta \end{array}$$

or

$$(q_{x_1} + q_{y_1} + q_{z_1})d\theta + q_g(dx dy dz)d\theta = (q_{x_2} + q_{y_2} + q_{z_2})d\theta + \rho c_p dx dy dz dT \quad (6)$$

where  $dT$  is the change in temperature of the element during  $d\theta$ , or  $dT = dt/d\theta$ . Thus the temperature distribution curve will be a function of the three dimensions and time, i.e.,  $T = F(X, Y, Z, \theta)$ .

The heat  $q_x$  conducted into the element across the face  $X_1$  during  $d\theta$  is expressed in equation (1) as:

$$q_{x_1} = -K dz dy \left. \frac{\partial T}{\partial x} \right|_{x_1} \quad (7)$$

It is necessary to express the temperature gradient as a partial derivative since  $T$  is a function not only of  $X$  but also of  $Y$  and  $Z$ . It is easily shown that the rate of heat flow across the face  $X_2$  can be expressed as:

$$q_{x_2} = \left[ -K \left. \frac{\partial T}{\partial x} \right|_{x_1} + \frac{\partial}{\partial x} \left( -K \frac{\partial T}{\partial x} \right) dx \right] dz dy \quad (8)$$

Subtracting equation 8 from 7 gives:

$$q_{x_1} - q_{x_2} = \left[ \frac{\partial}{\partial x} \left( K \left. \frac{\partial T}{\partial x} \right) \right] dx dy dz \quad (9)$$

Similar expressions can be written for the Y and Z direction accordingly.

$$q_{z1} - q_{z2} = \left[ \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) \right] dx dz dy \quad (10)$$

$$q_{y1} - q_{y2} = \left[ \frac{\partial}{\partial y} \left( k \frac{\partial T}{\partial y} \right) \right] dx dz dy \quad (11)$$

Substituting equation 9, 10 and 11 into the energy balance equation and disregarding any internal heat source and dividing each term by  $dx dz dy$  yields:

$$\frac{\partial}{\partial x} \left( k \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + \frac{\partial}{\partial y} \left( k \frac{\partial T}{\partial y} \right) = c_p \rho \frac{\partial T}{\partial \theta} \quad (12)$$

If the temperature difference between points 1 and 2, i.e.,  $X_1, X_2, Z_1, Z_2,$  and  $Y_1, Y_2$  is not too great then  $C_p,$  and  $k$  can be considered constant over the temperature range  $T_1$  to  $T_2$ . Making the assumption of constancy and dividing equation (2) by  $k$  gives:

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial z^2} + \frac{\partial^2 T}{\partial y^2} = \frac{c_p \rho}{k} \frac{\partial T}{\partial \theta} \quad (13)$$

Setting  $k/c_p \rho = H$ , (thermal diffusivity) and substituting into equation (13) gives the Fourier heat transfer equation

$$\frac{\partial^2 T}{\partial X^2} + \frac{\partial^2 T}{\partial Z^2} + \frac{\partial^2 T}{\partial Y^2} = \frac{1}{H} \frac{\partial T}{\partial \theta}. \quad (14)$$

If the rod is carried to a steady state, i.e.,  $\frac{\partial T}{\partial \theta} = 0$  then the result is the Laplace heat transfer equation

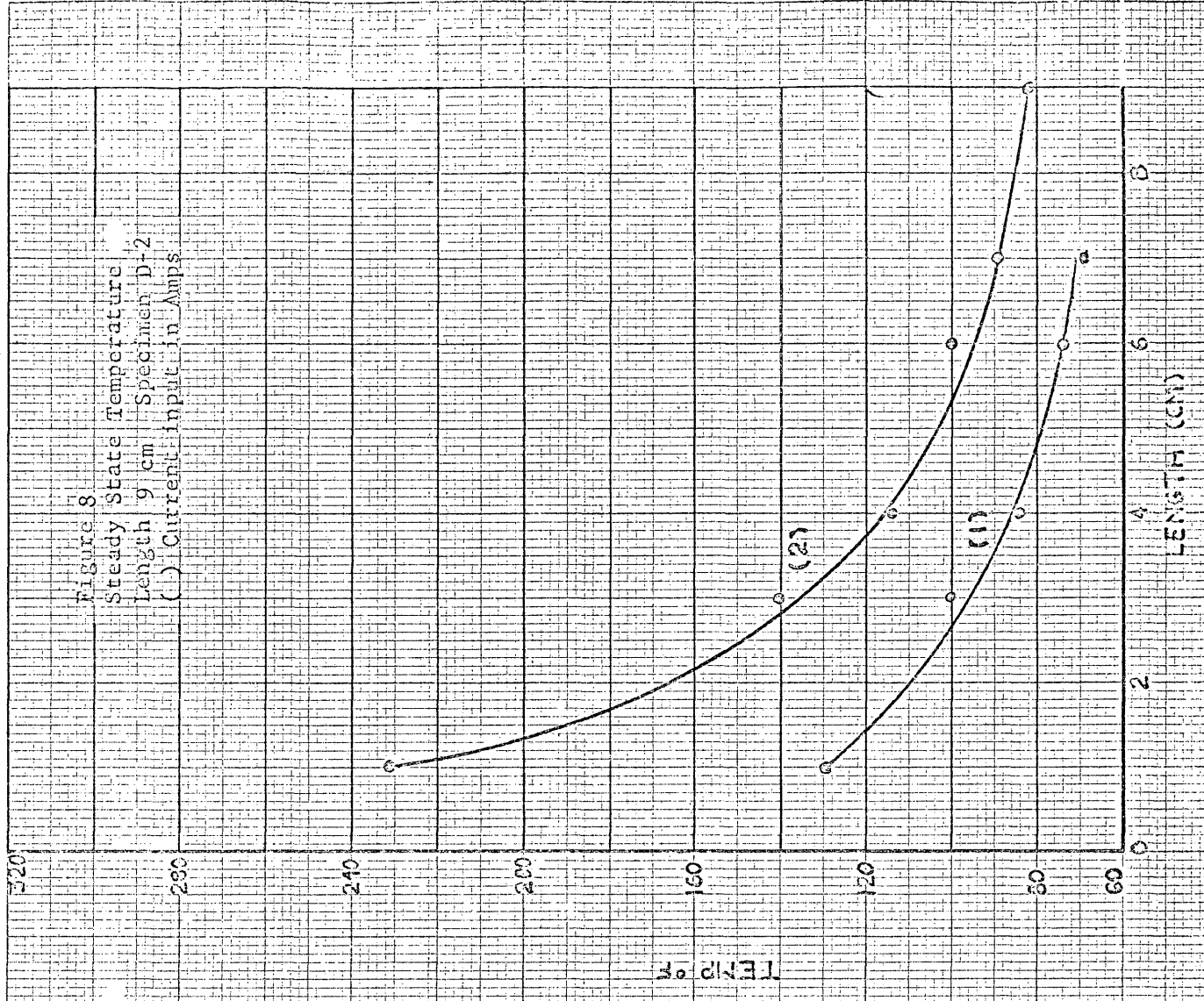
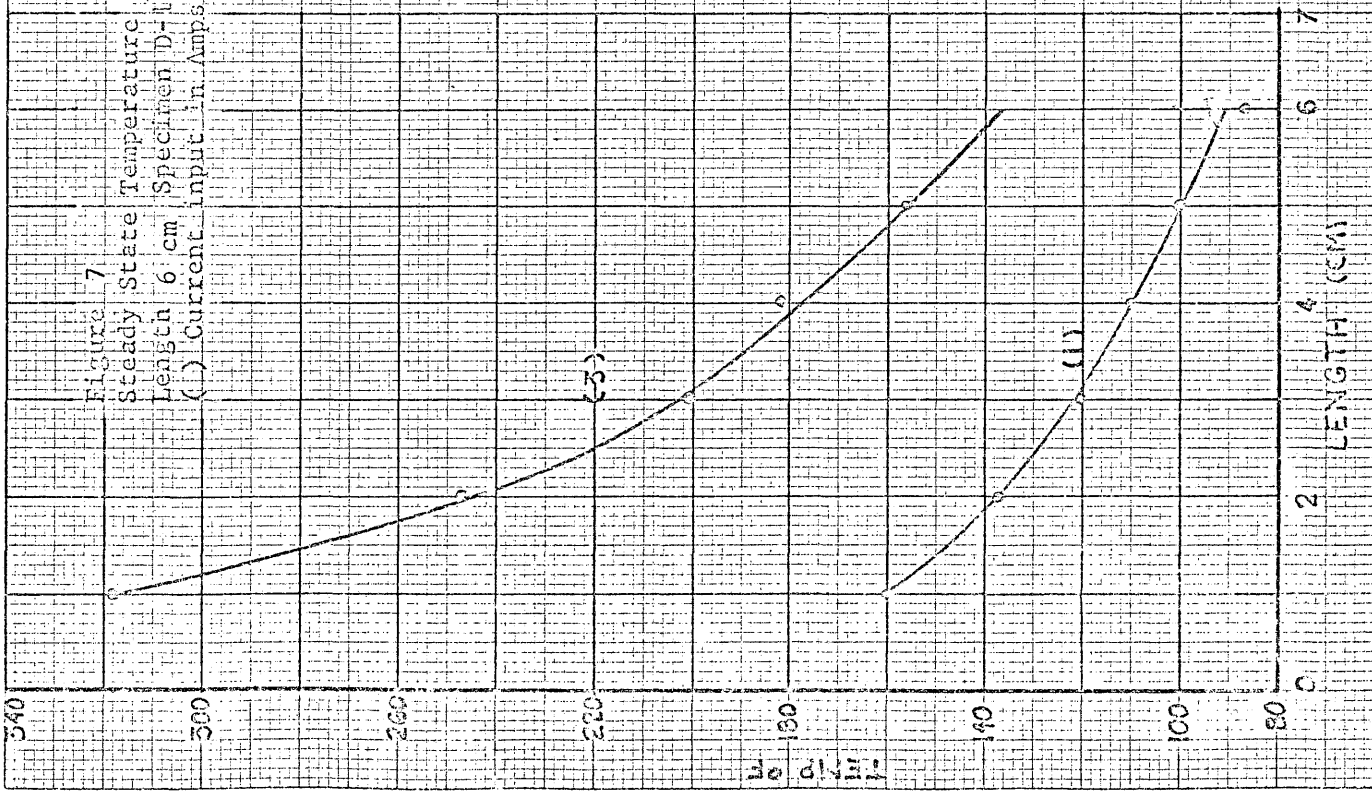
$$\frac{\partial^2 T}{\partial X^2} + \frac{\partial^2 T}{\partial Z^2} + \frac{\partial^2 T}{\partial Y^2} = 0. \quad (15)$$

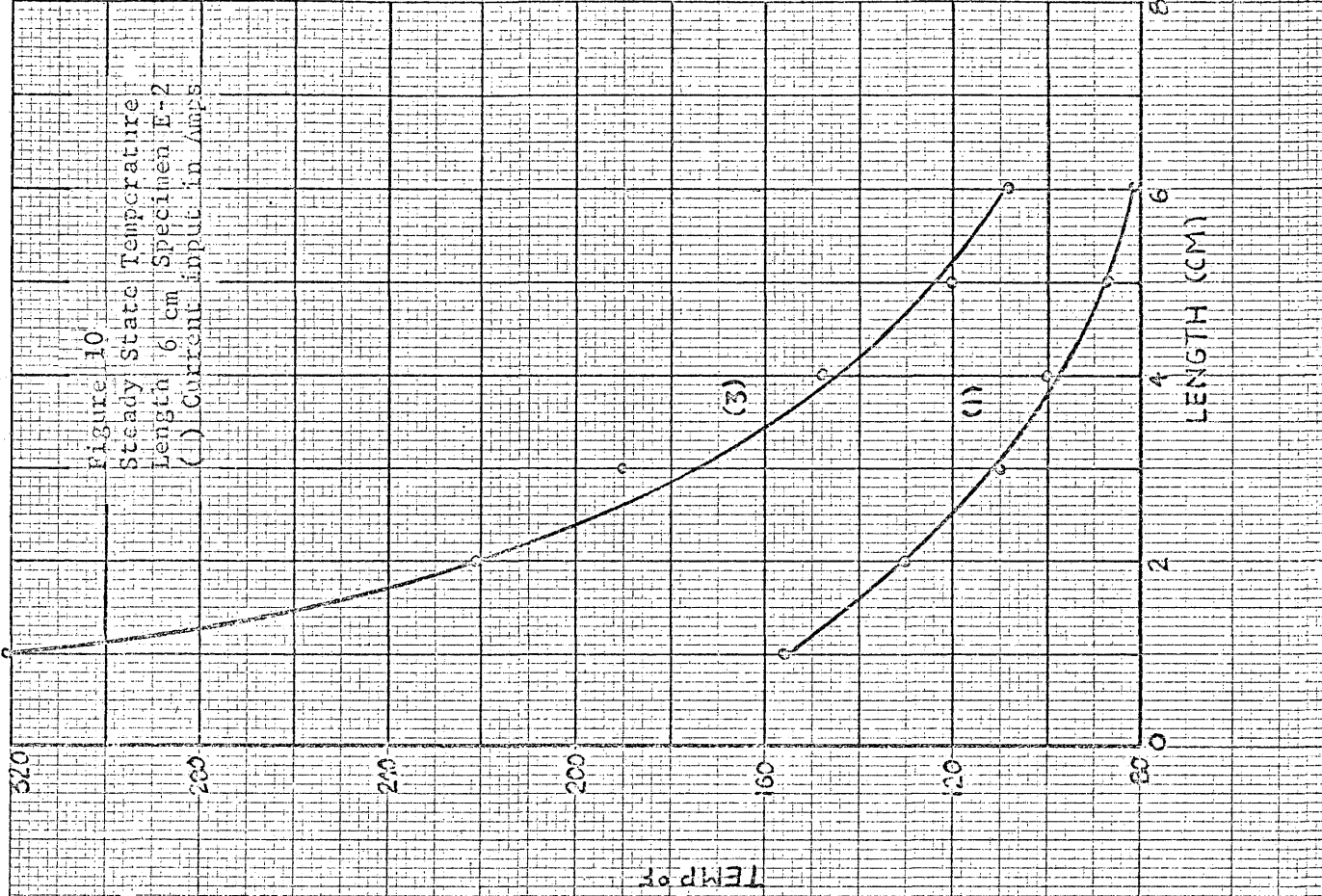
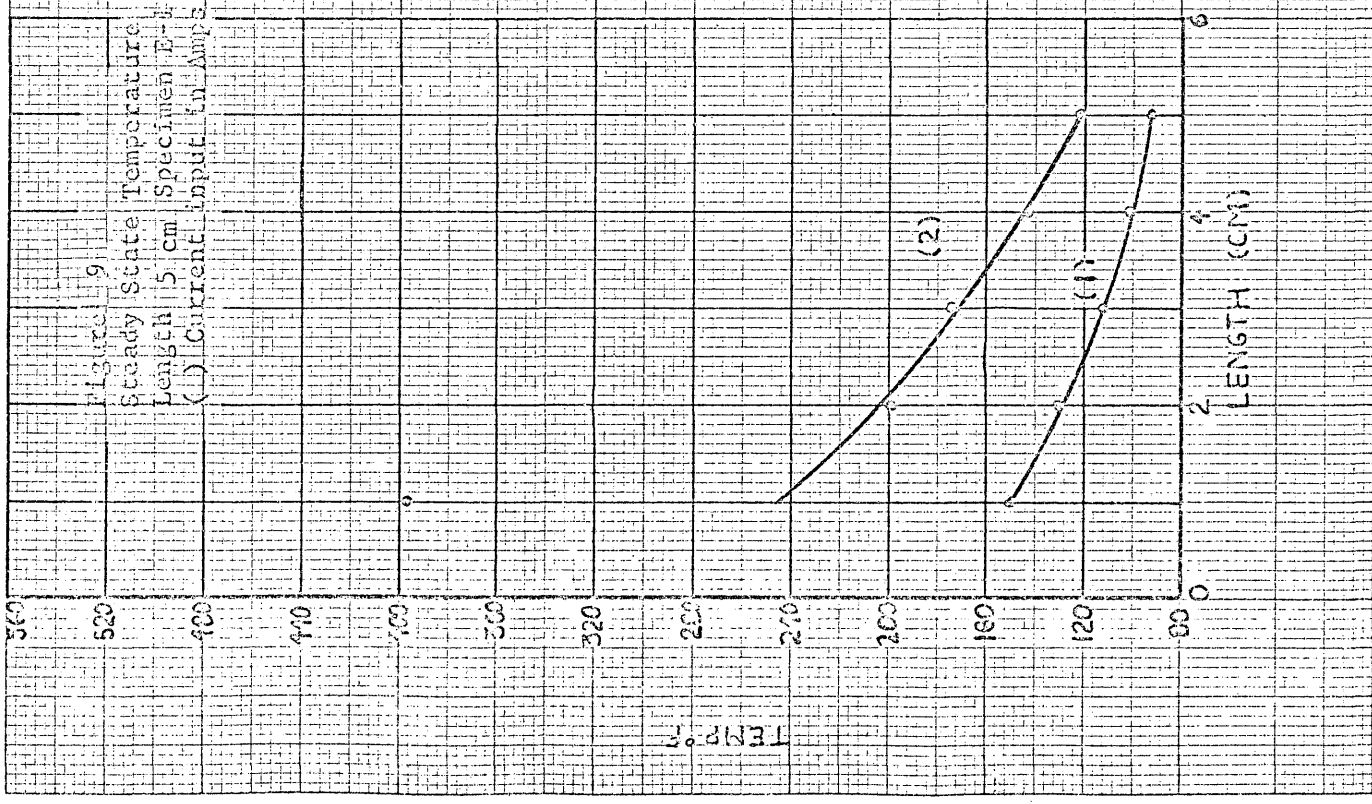
Since the heat flow in the rods used for this research was in one direction, equation (14) reduces to

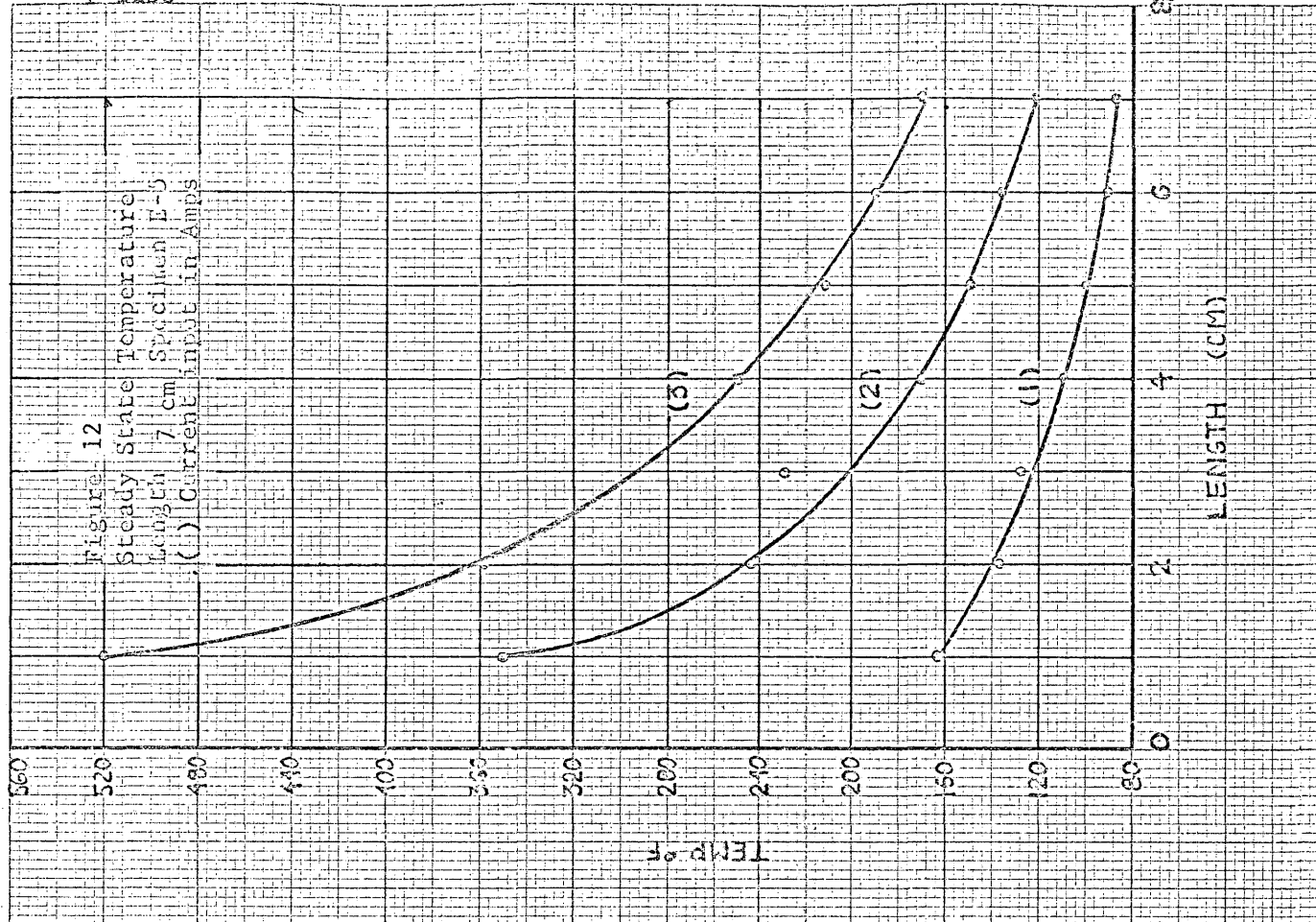
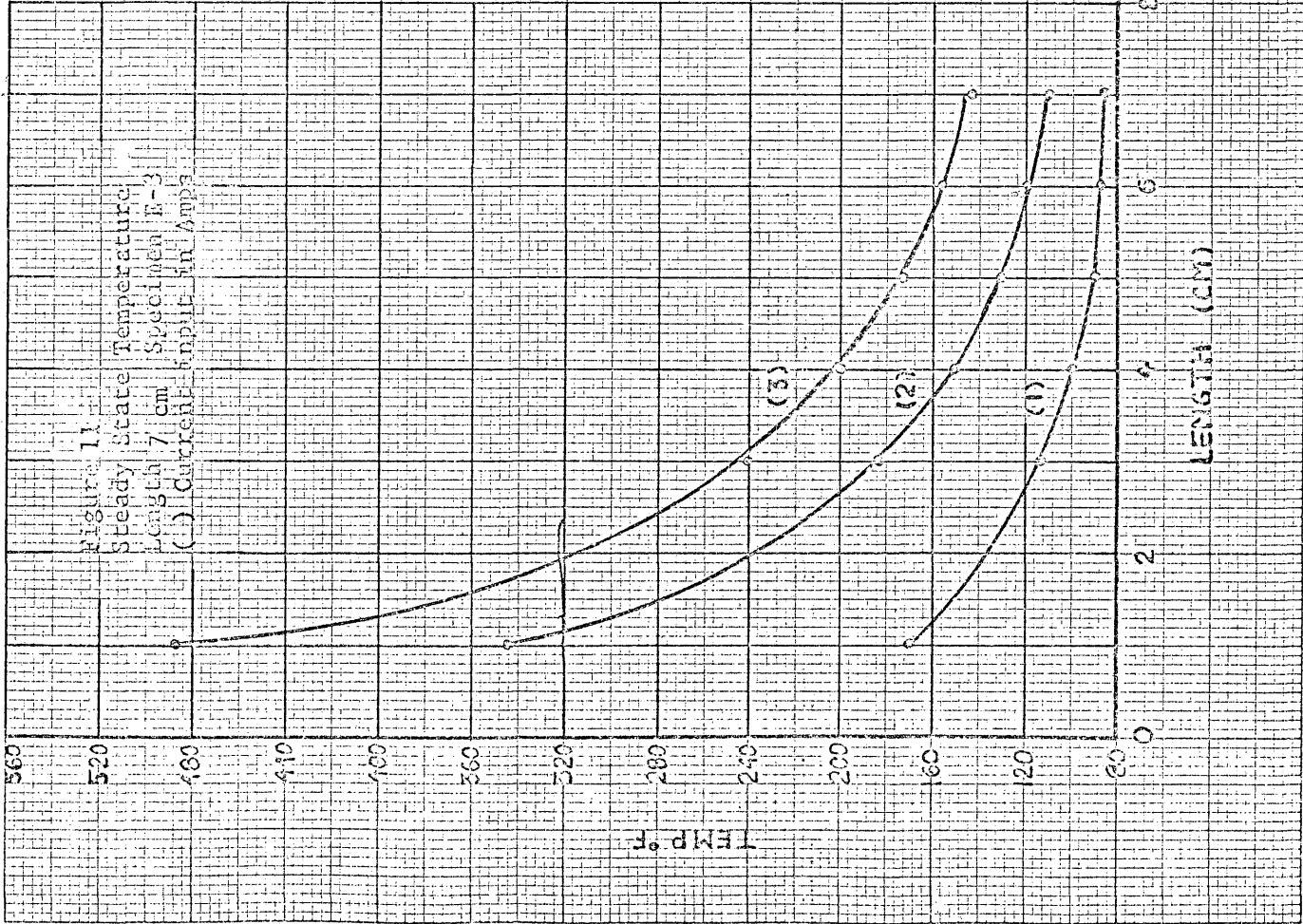
$$\frac{\partial^2 T}{\partial X^2} = \frac{1}{H} \frac{\partial T}{\partial \theta}. \quad (16)$$

Unsteady state temperature profiles are available on the specimen used in this research; however, H was calculated as:

$$H = \frac{K}{C_p \rho}. \quad (17)$$









560

520

480

440

400

360

320

280

240

200

160

120

80

0

TEMP °F

Figure 13

Steady State Temperature

Length 7 cm Specimens E-6, E-8

( ) Current Input in Amps



560

520

480

440

400

360

320

280

240

200

160

120

80

0

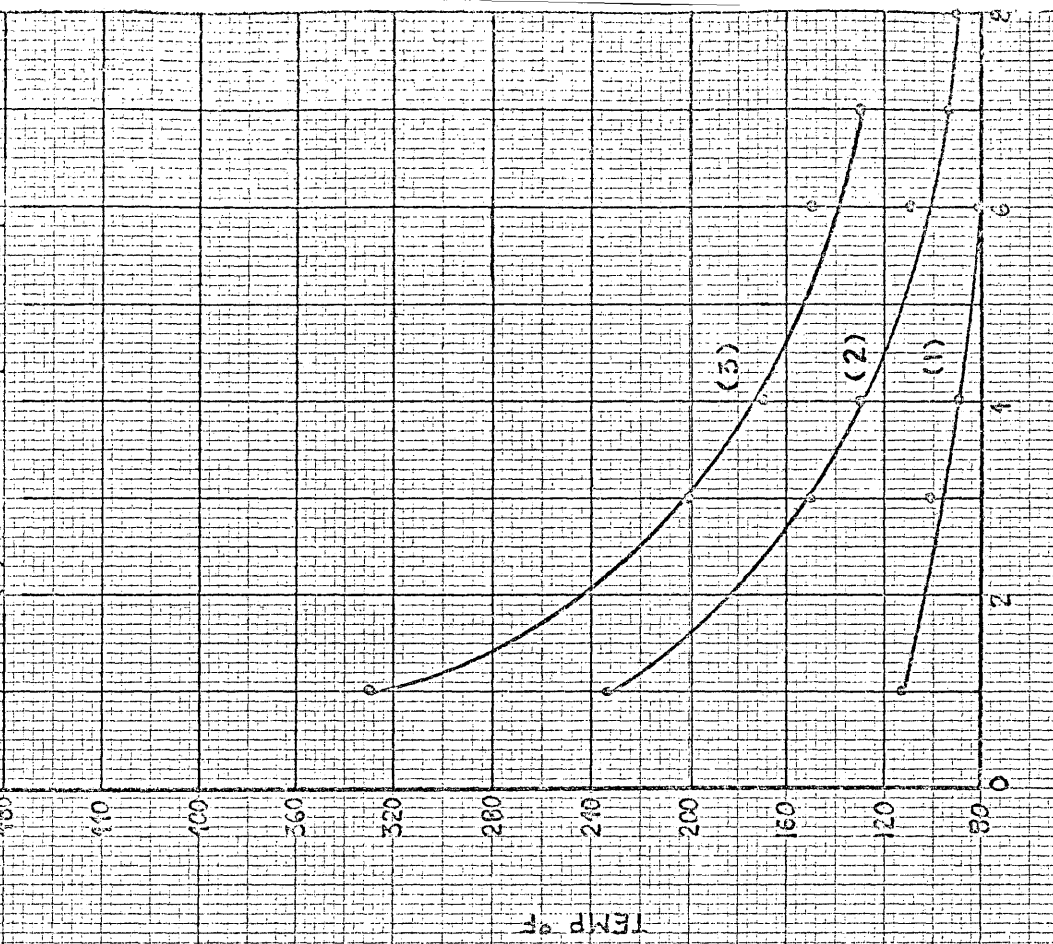
TEMP °F

Figure 14

Steady State Temperature

Length 8 cm Specimen O-1

( ) Current Input in Amps



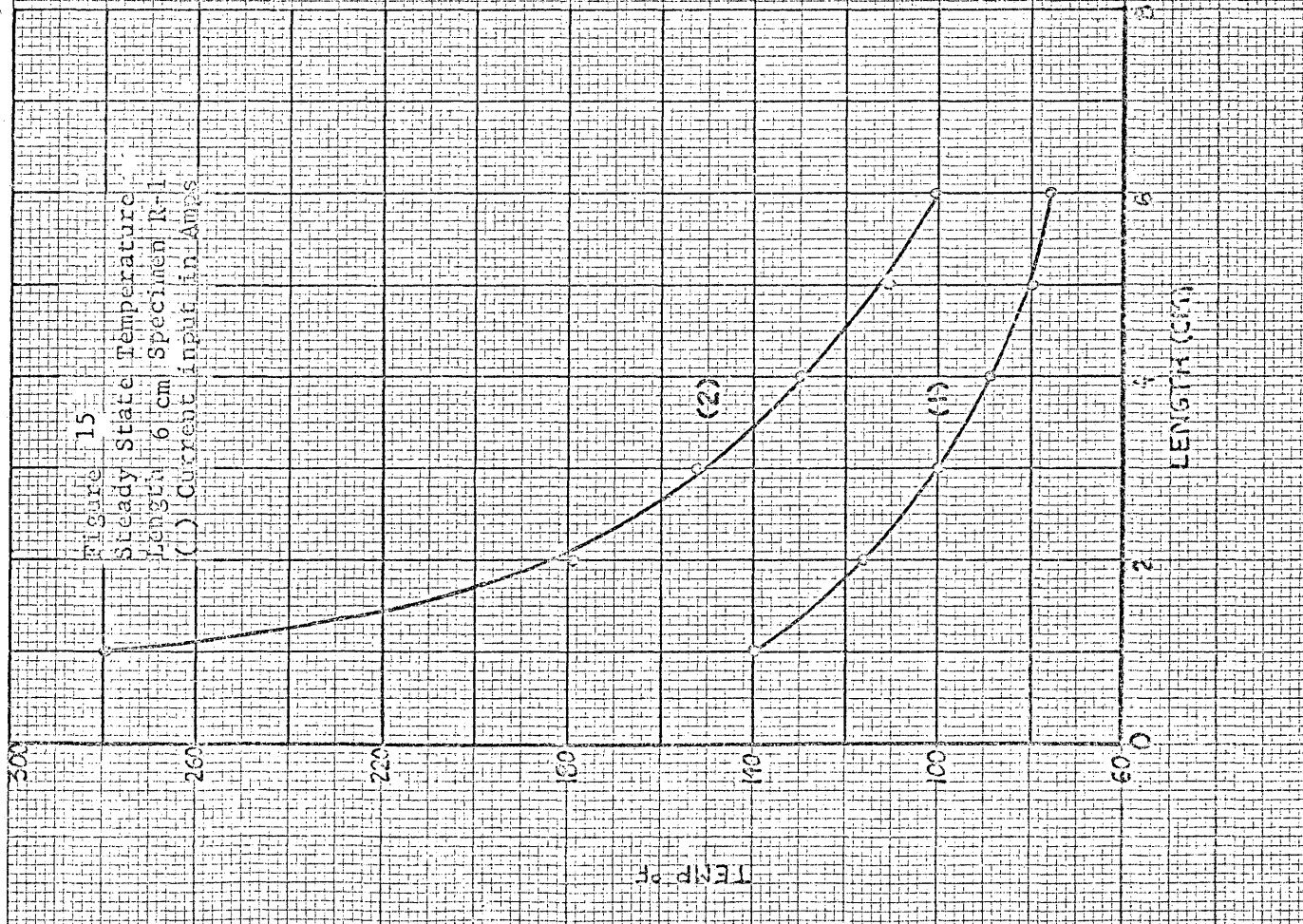
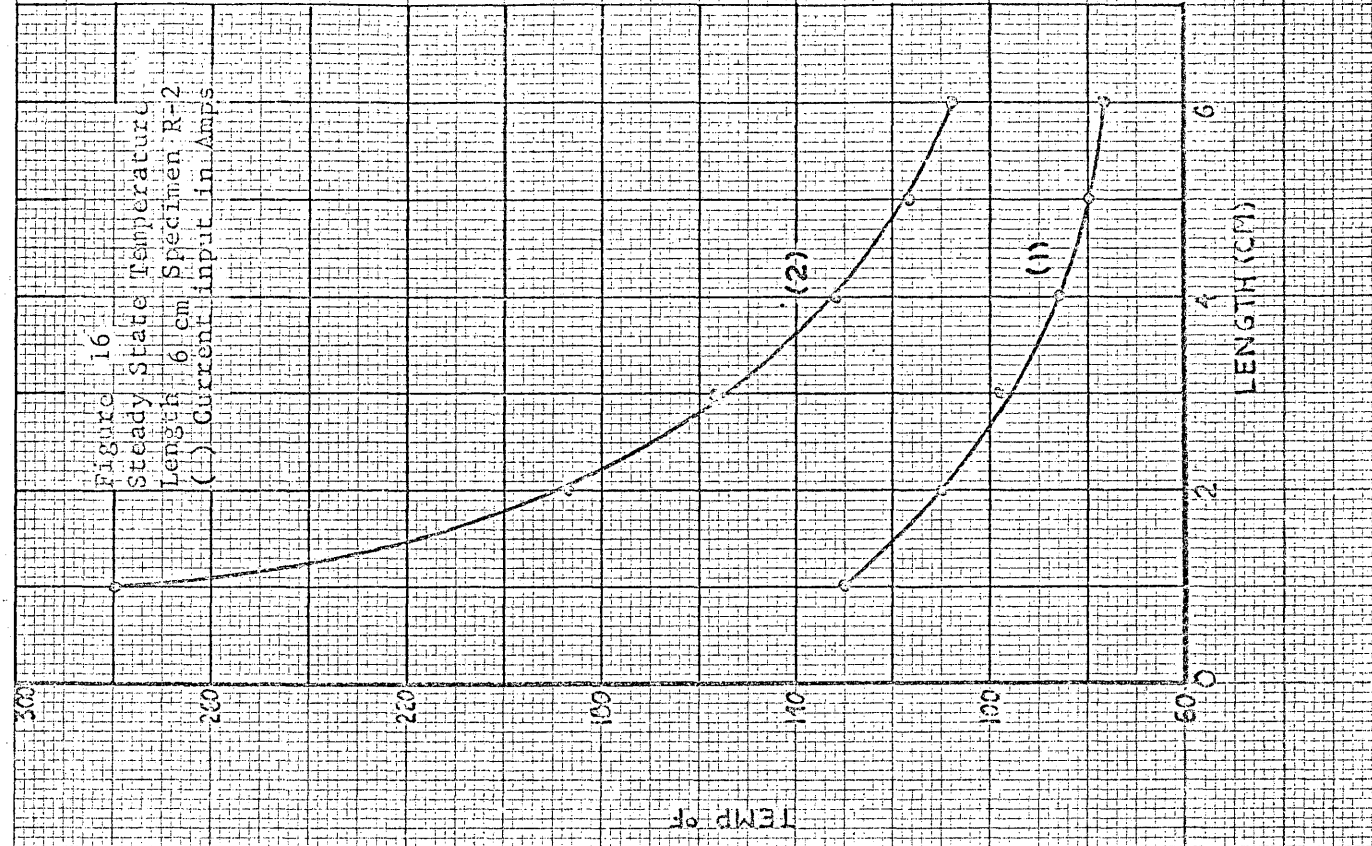


Figure 17

Steady State Temperature  
Length 6 cm Specimen S-1  
( ) Current Input in Amps

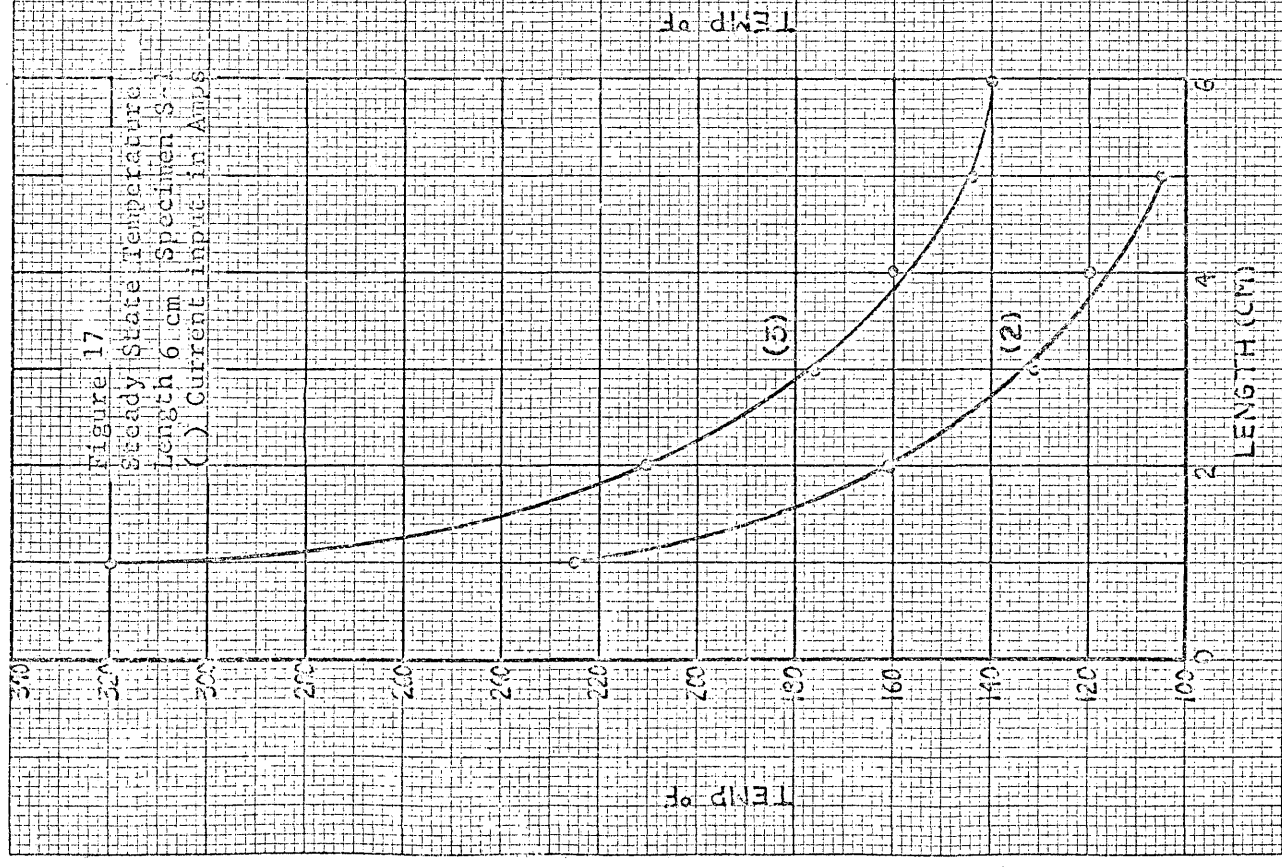
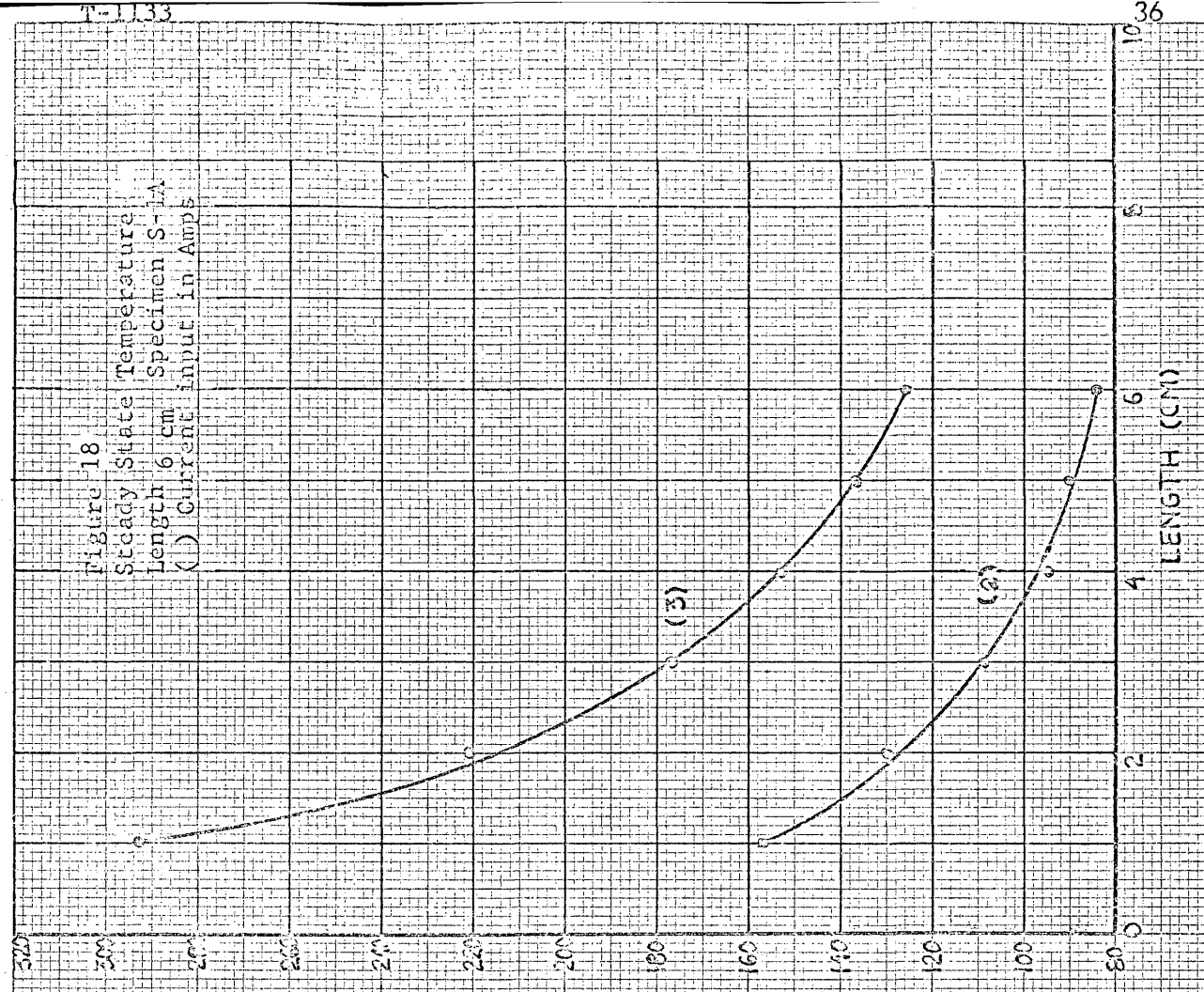


Figure 18

Steady State Temperature  
Length 6 cm Specimen S-1A  
( ) Current Input in Amps



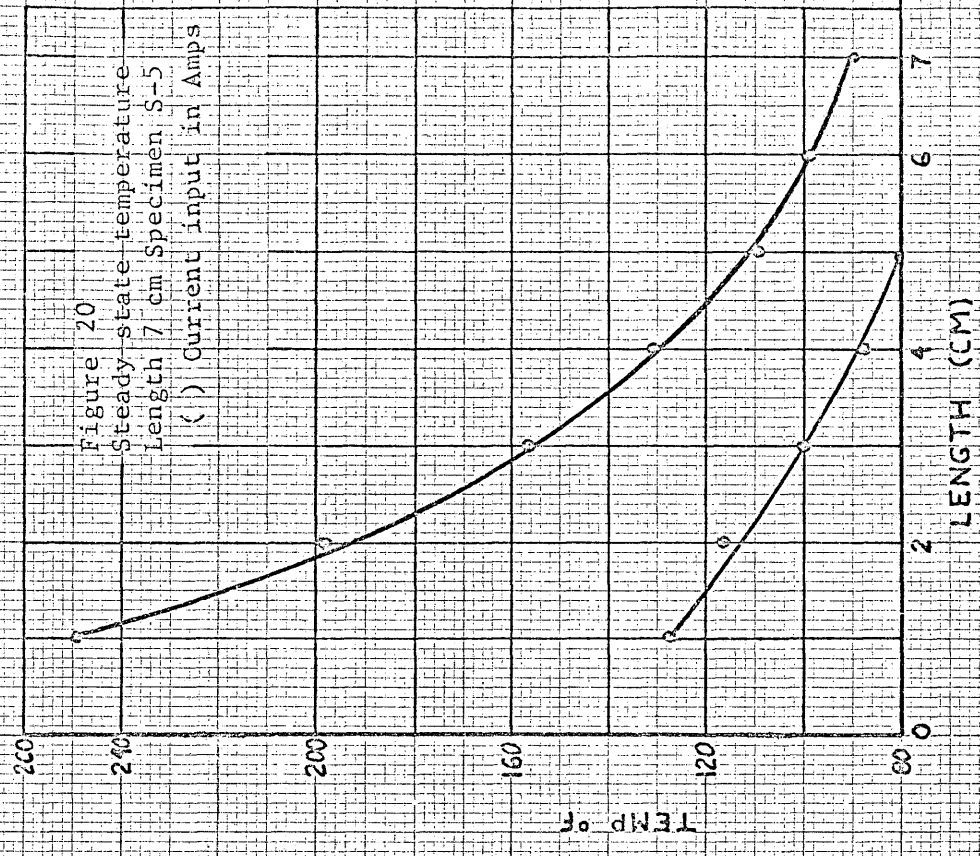
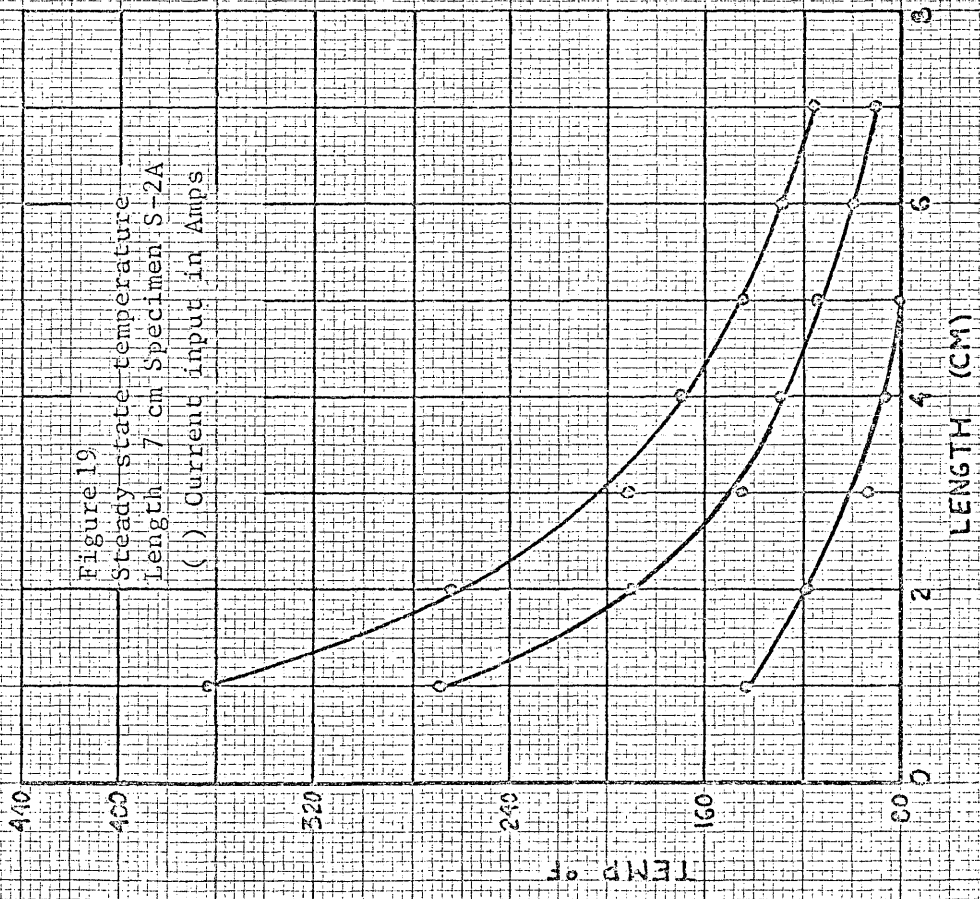


Figure 21  
Steady state temperature  
Length 9 cm Specimen S-3  
( ) Current input in Amps

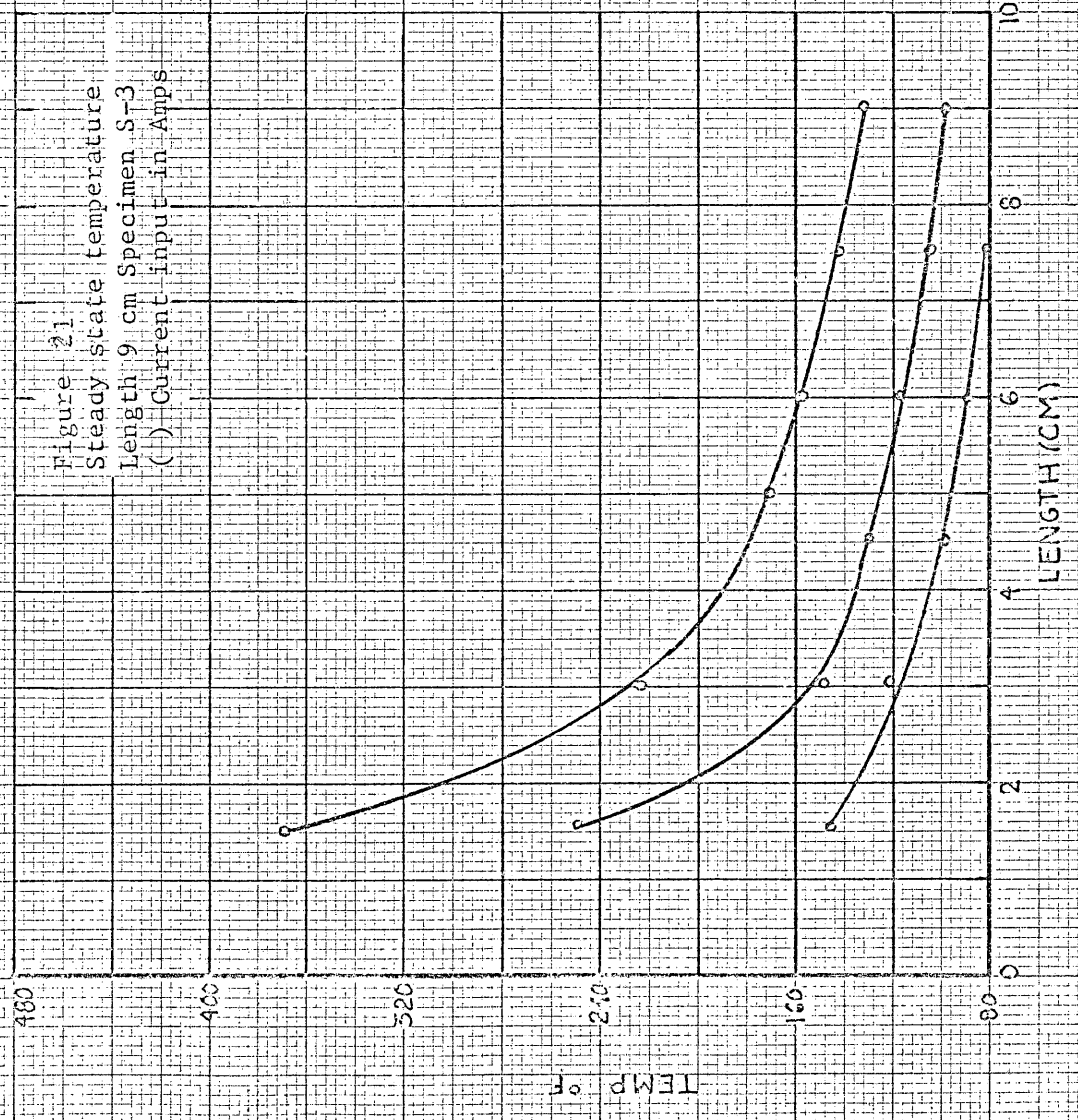
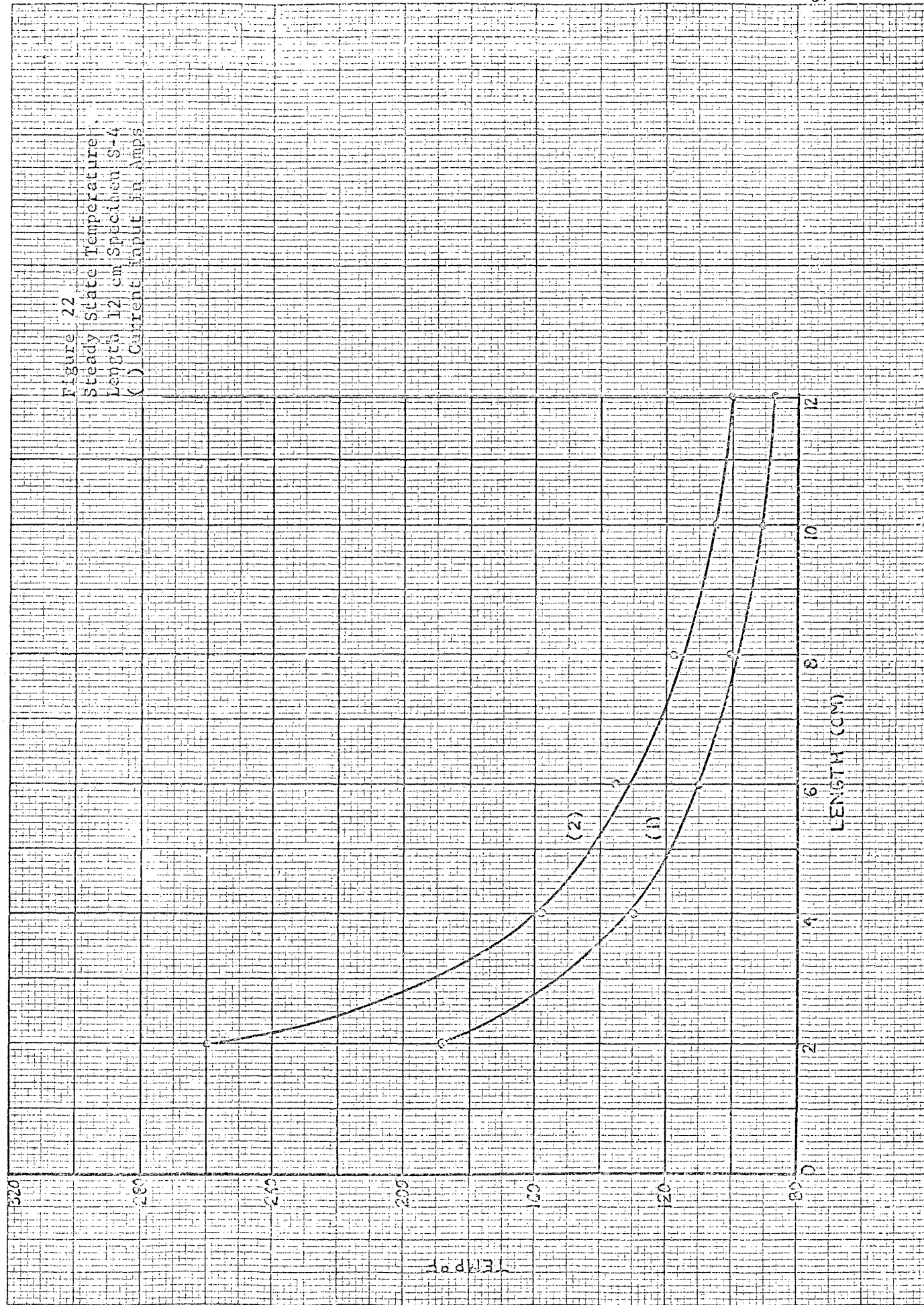


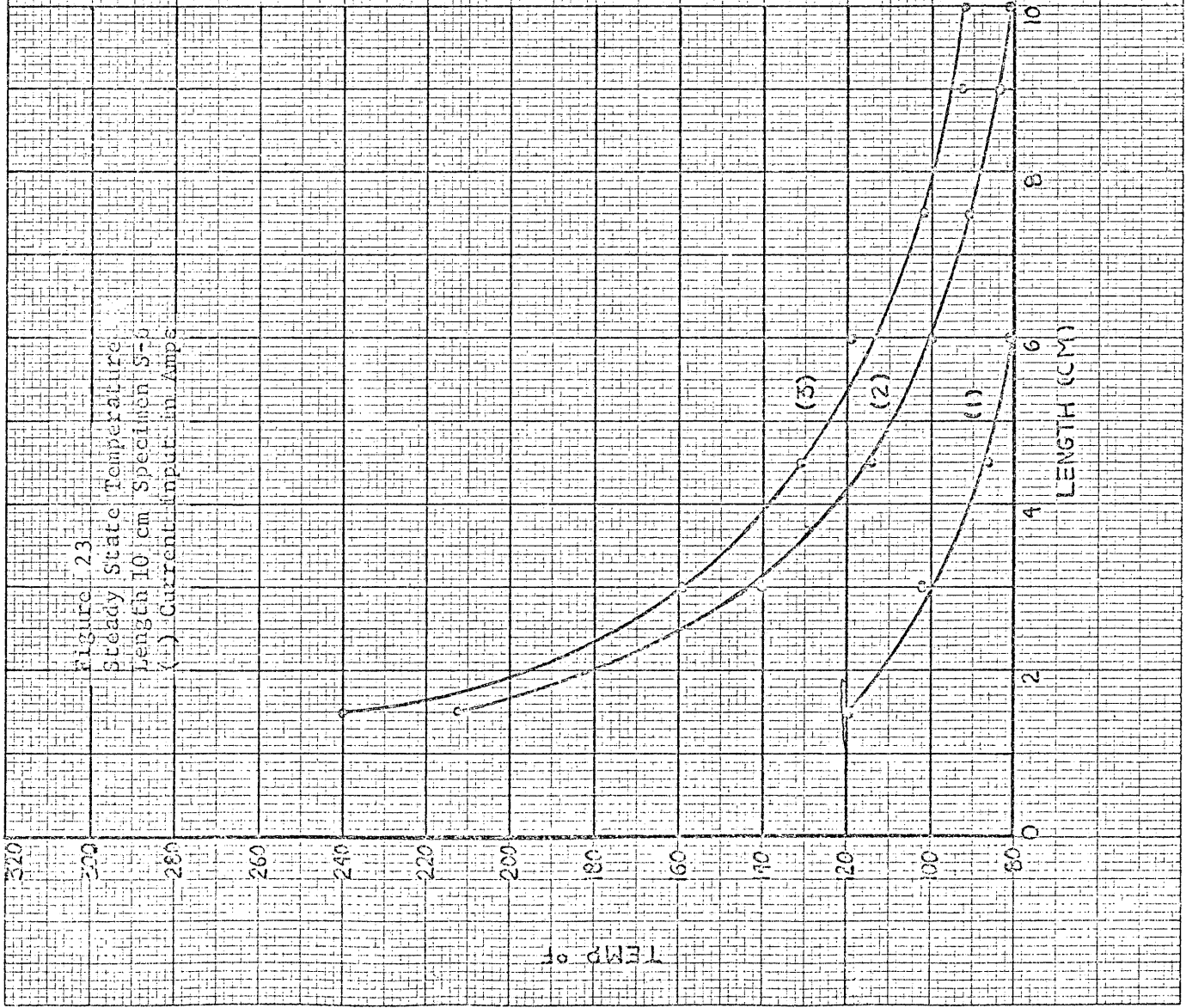
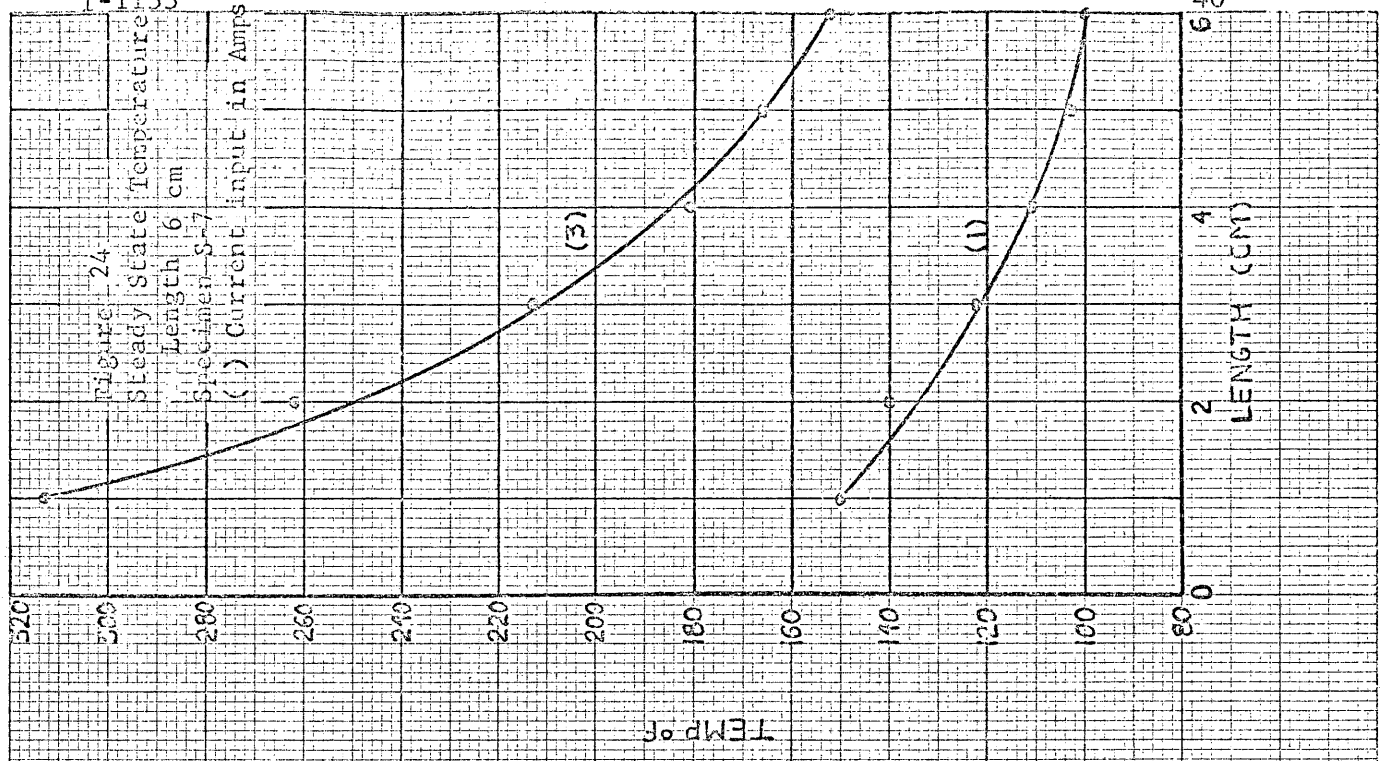
Figure 22

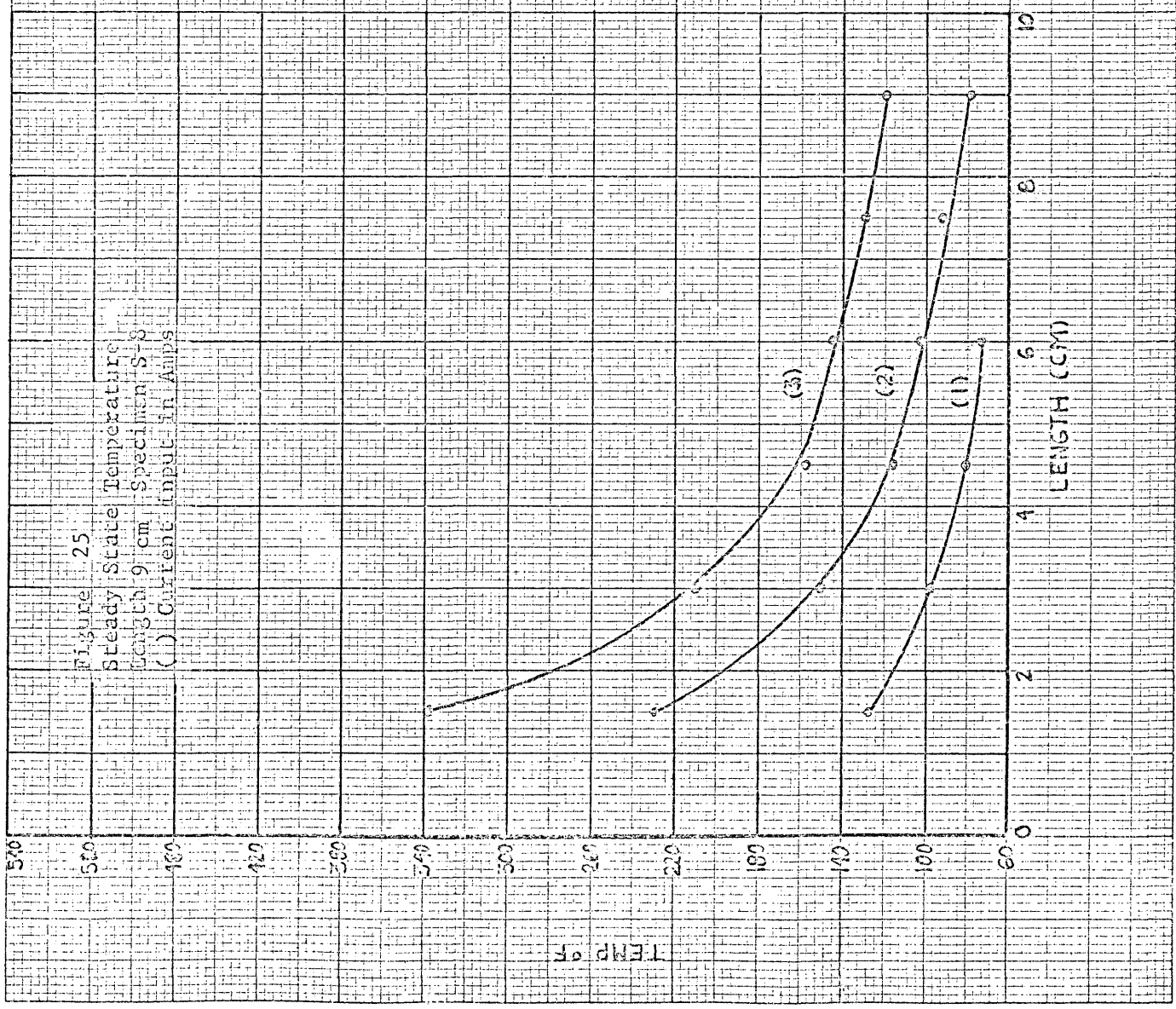
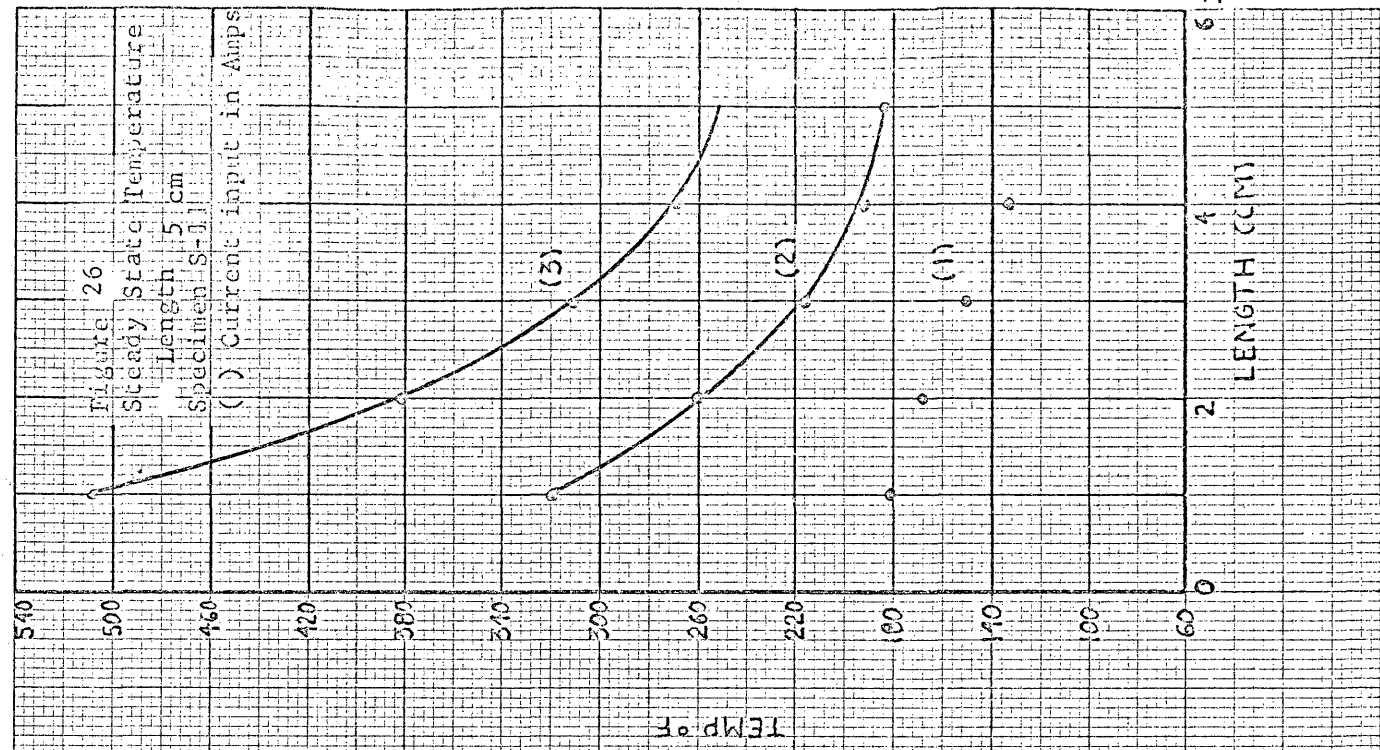
Steady State Temperature

Length 12 cm Specimen S-4

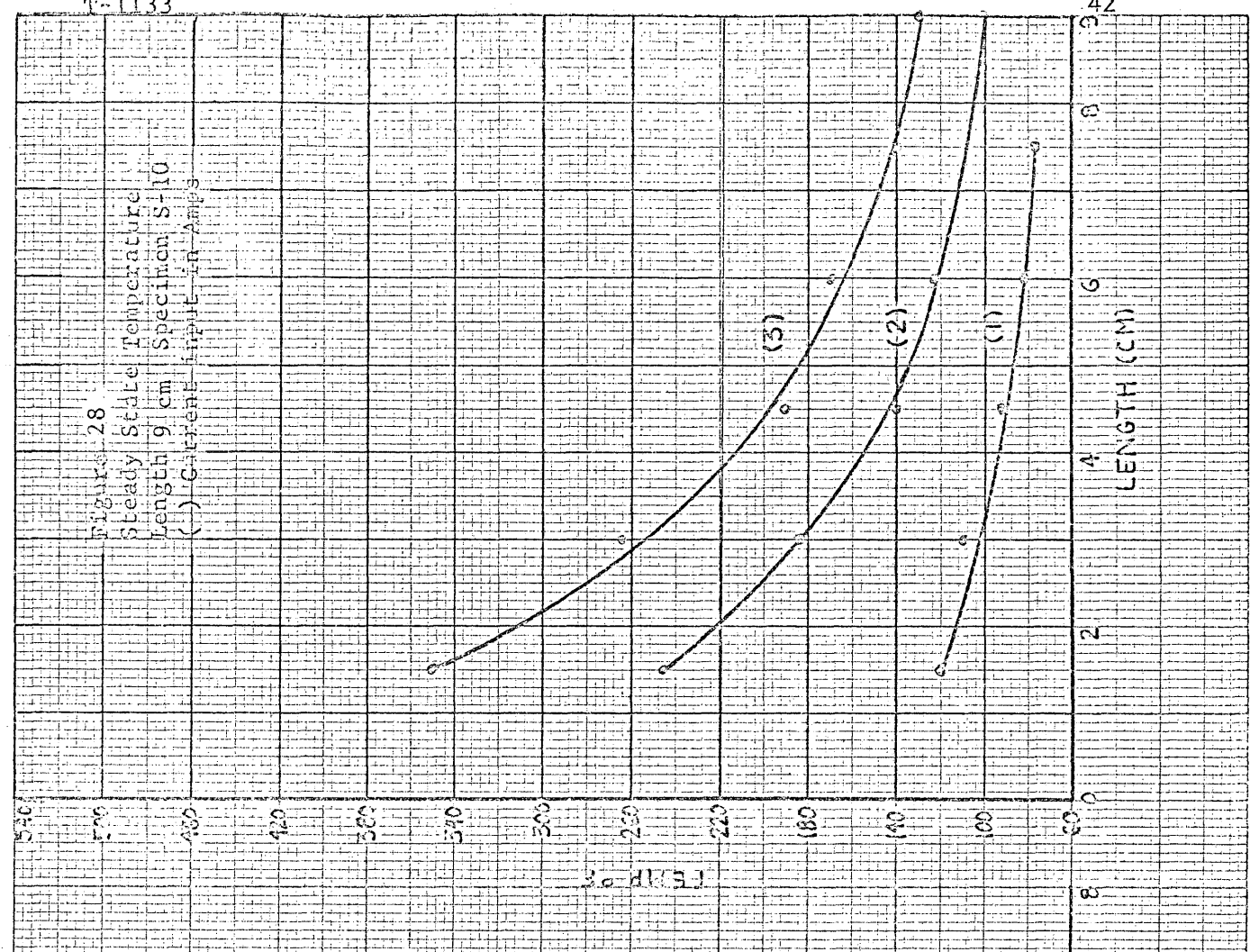
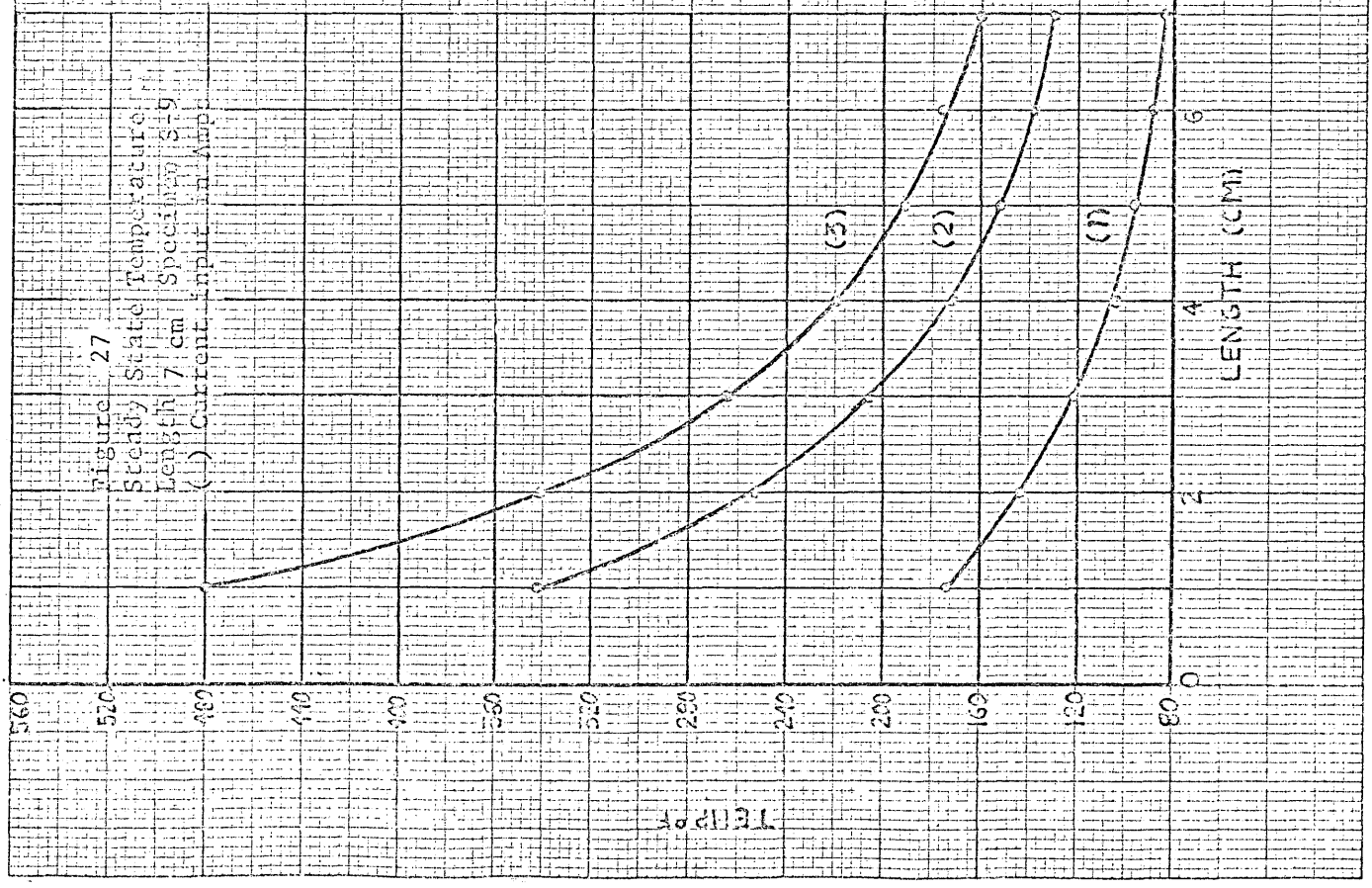
(C) Current (Amps) 10

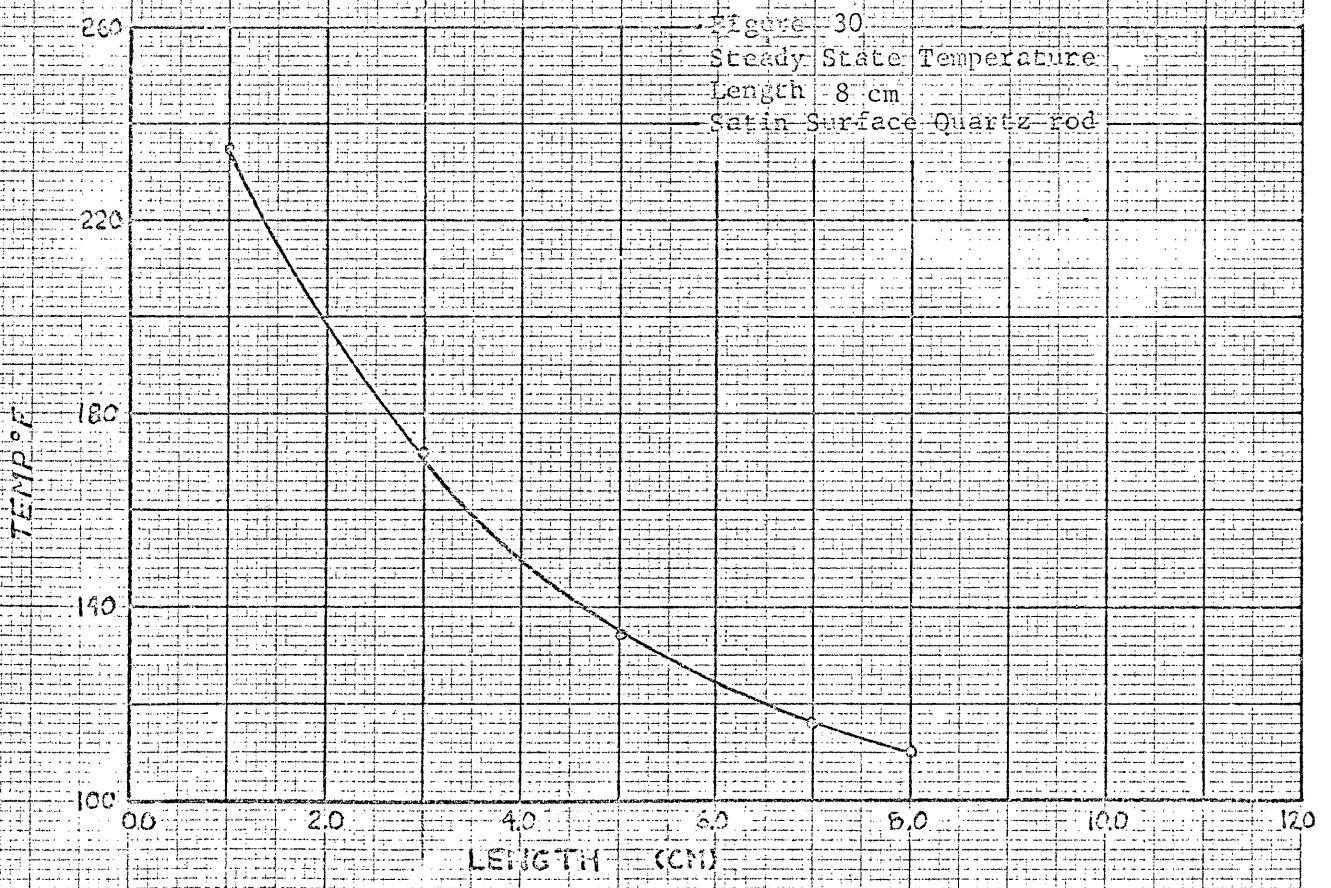
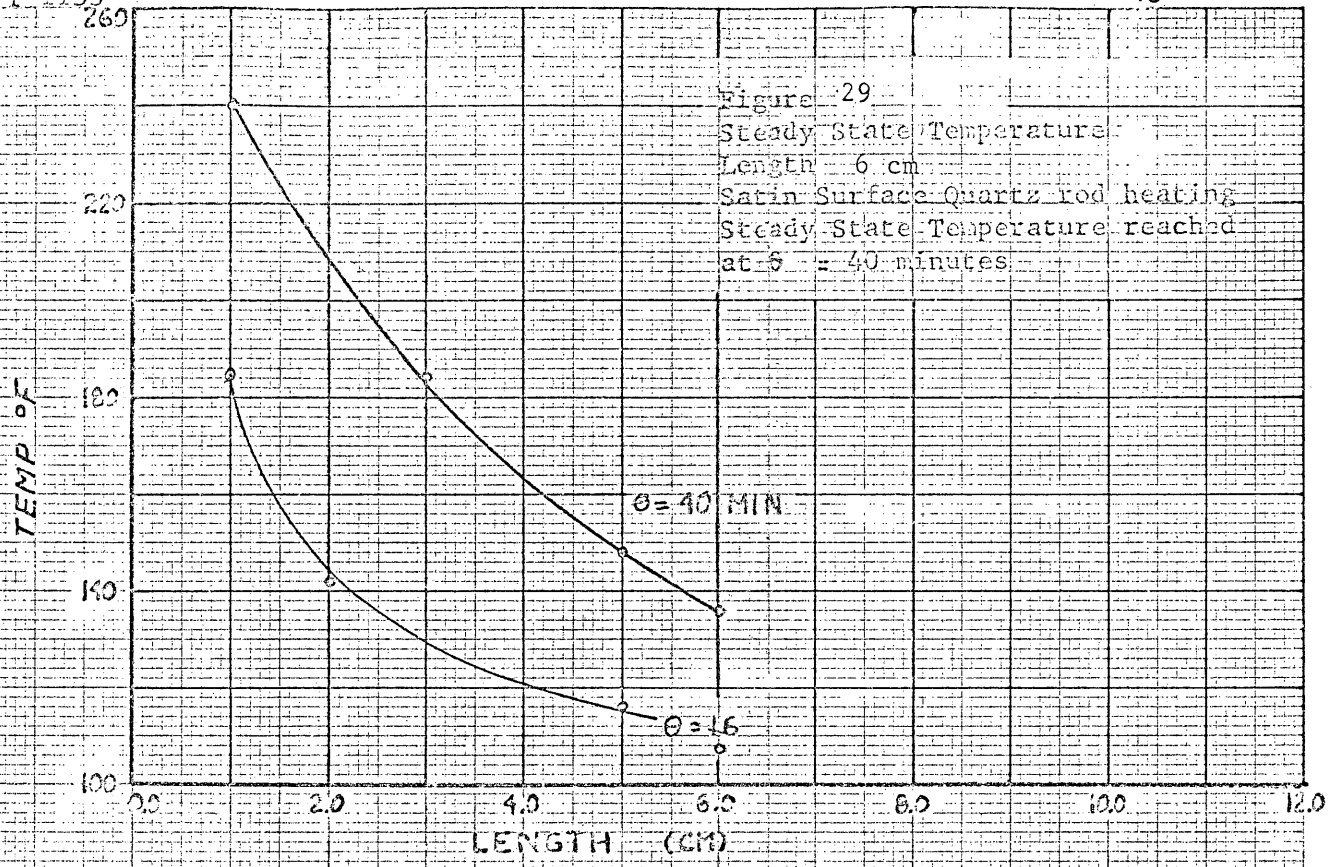












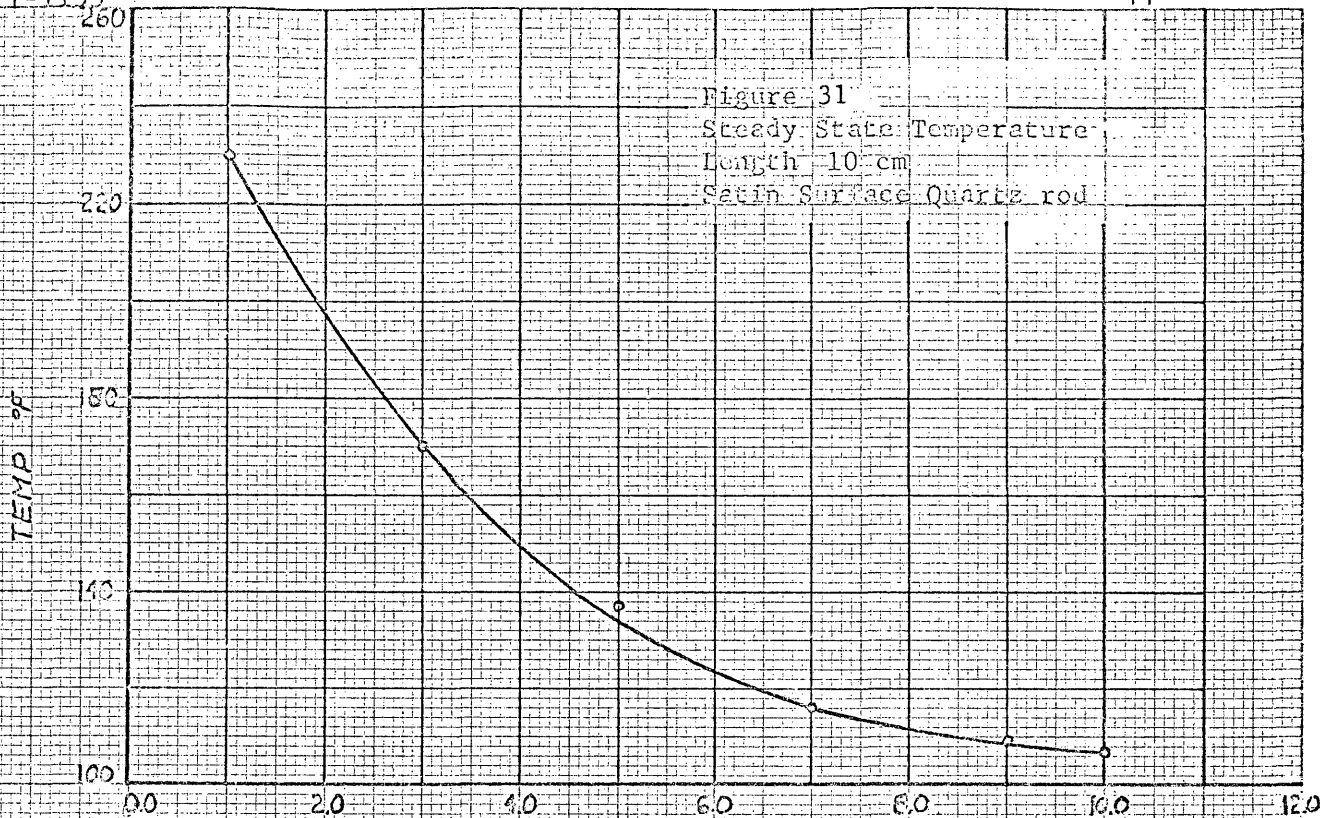
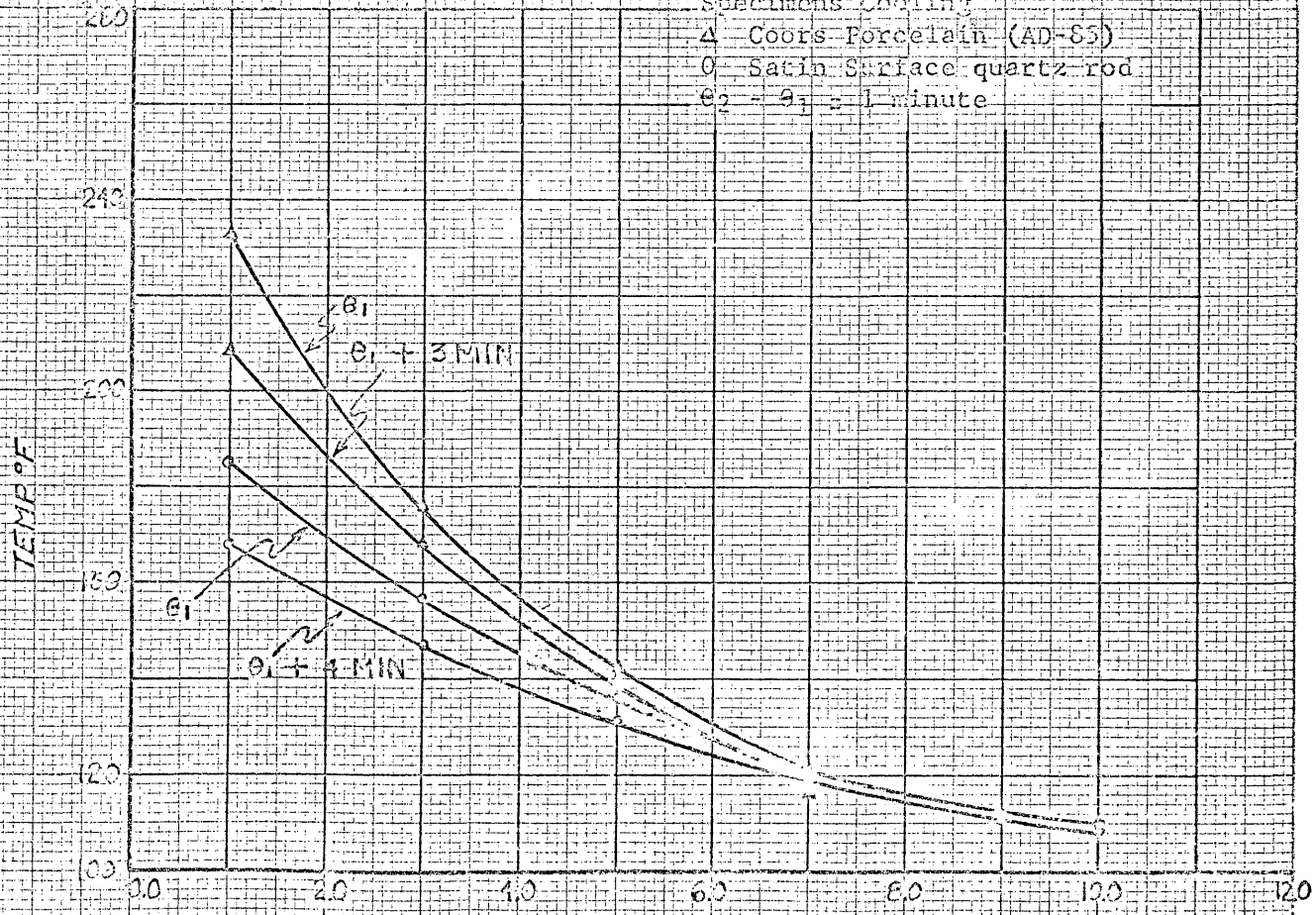
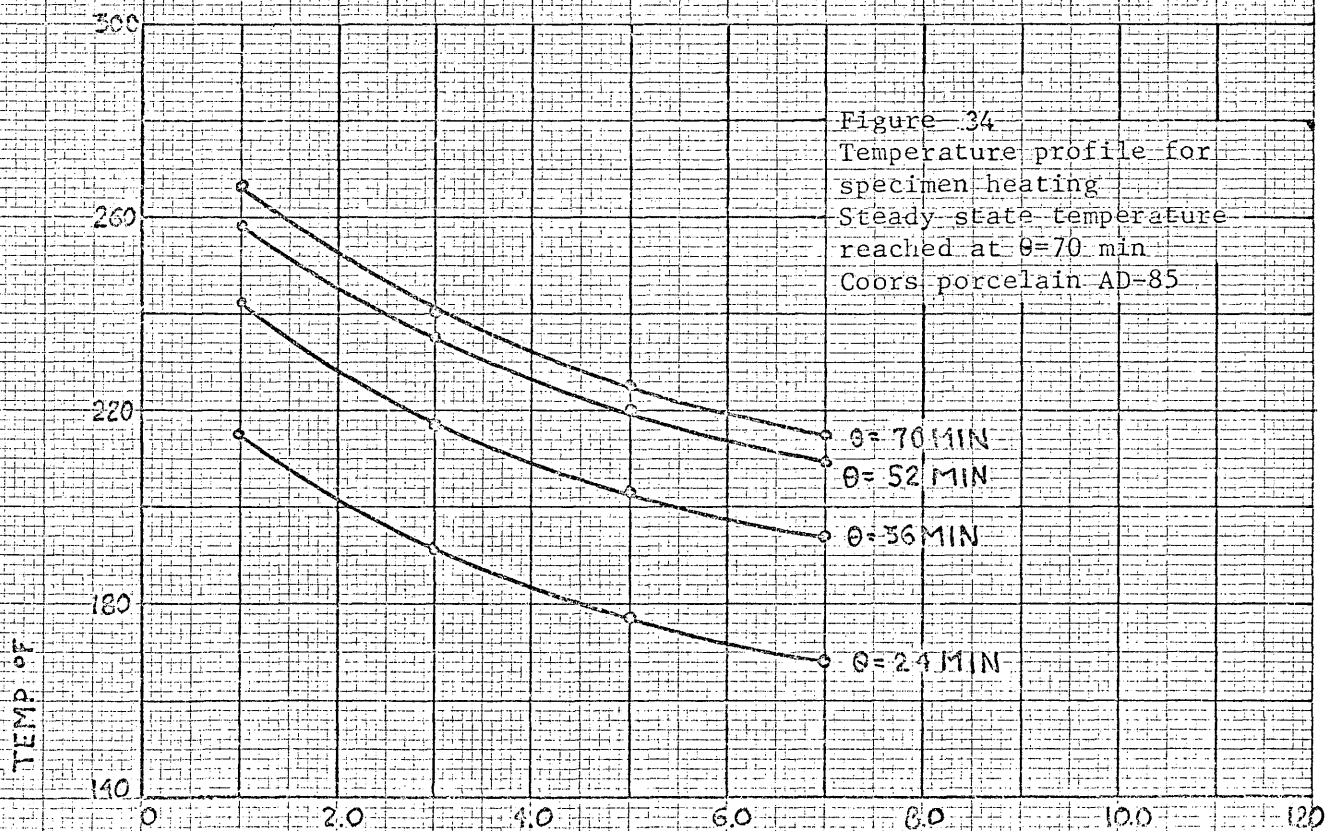
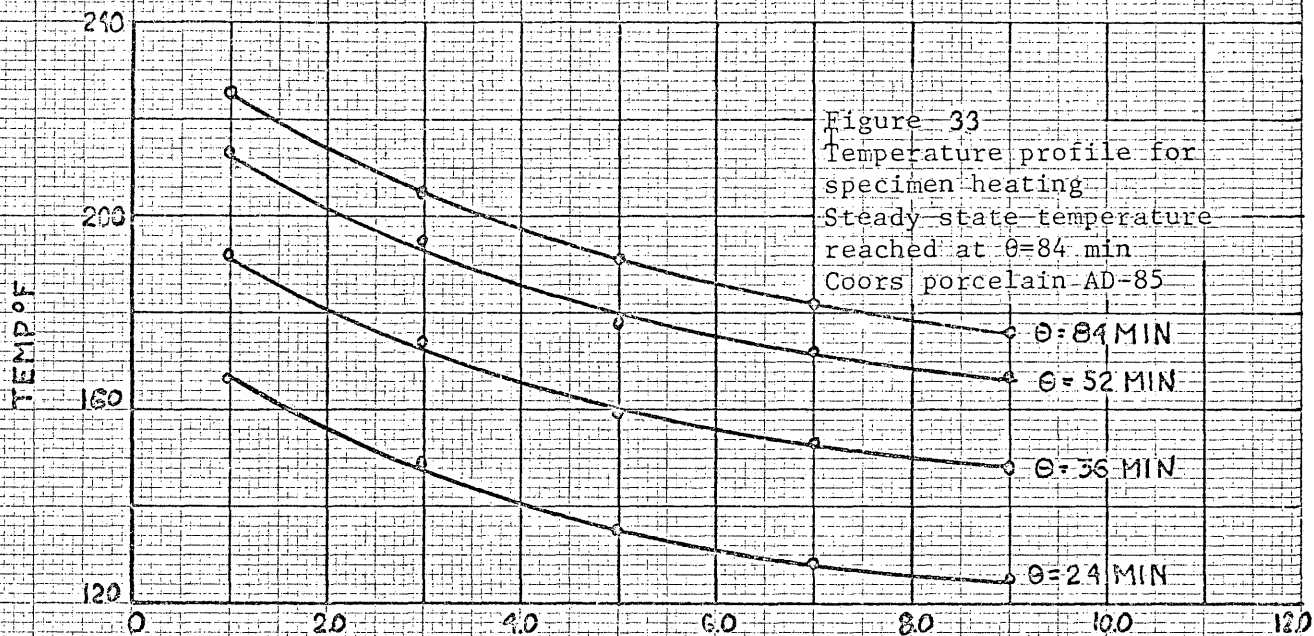


Figure 32

Temperature Length 10 cm  
Specimens Cooling  
Δ Coors Porcelain (AD-83)  
○ Satin Surface quartz rod  
 $t_2 - t_1 = 1$  minute





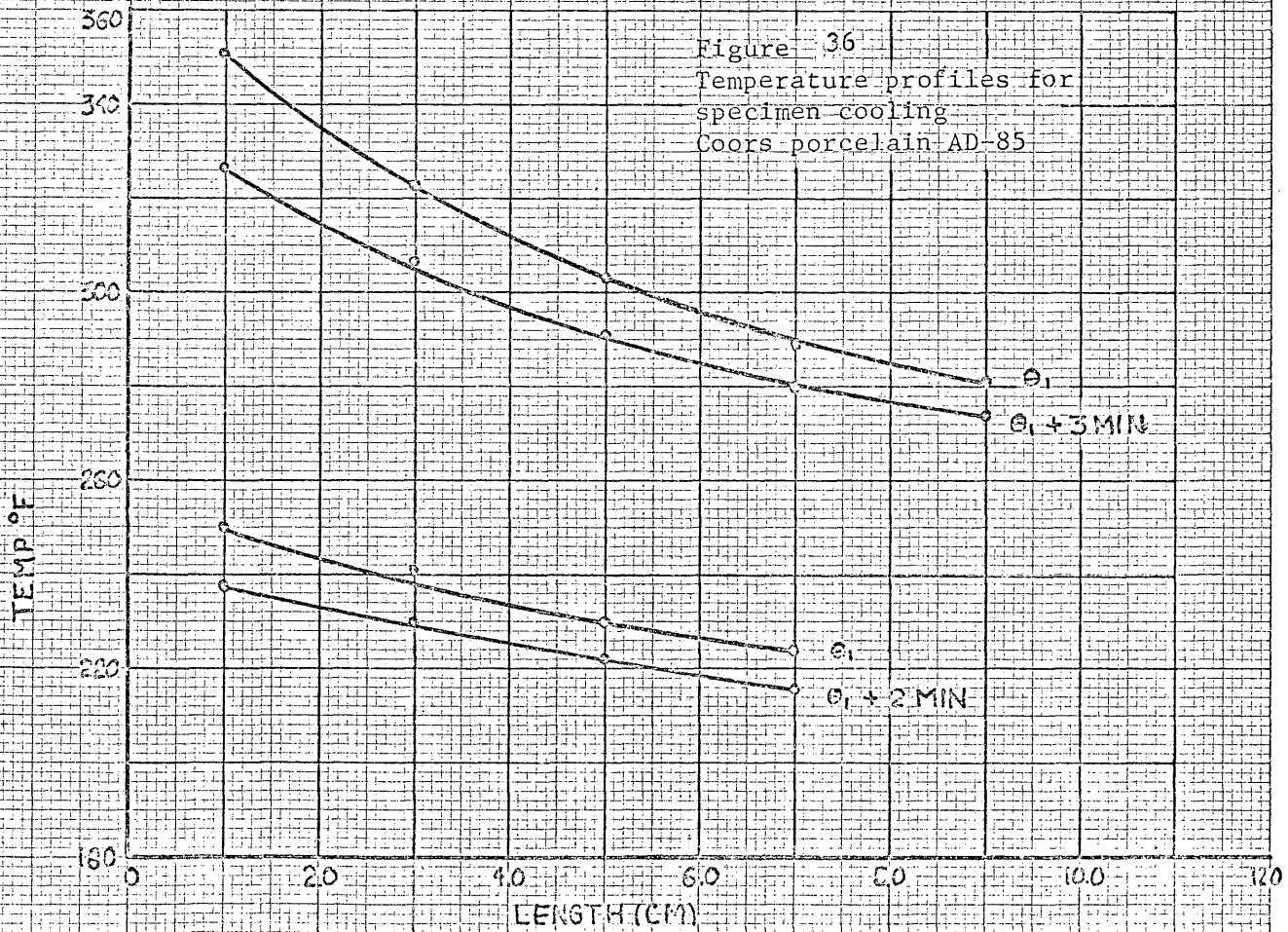
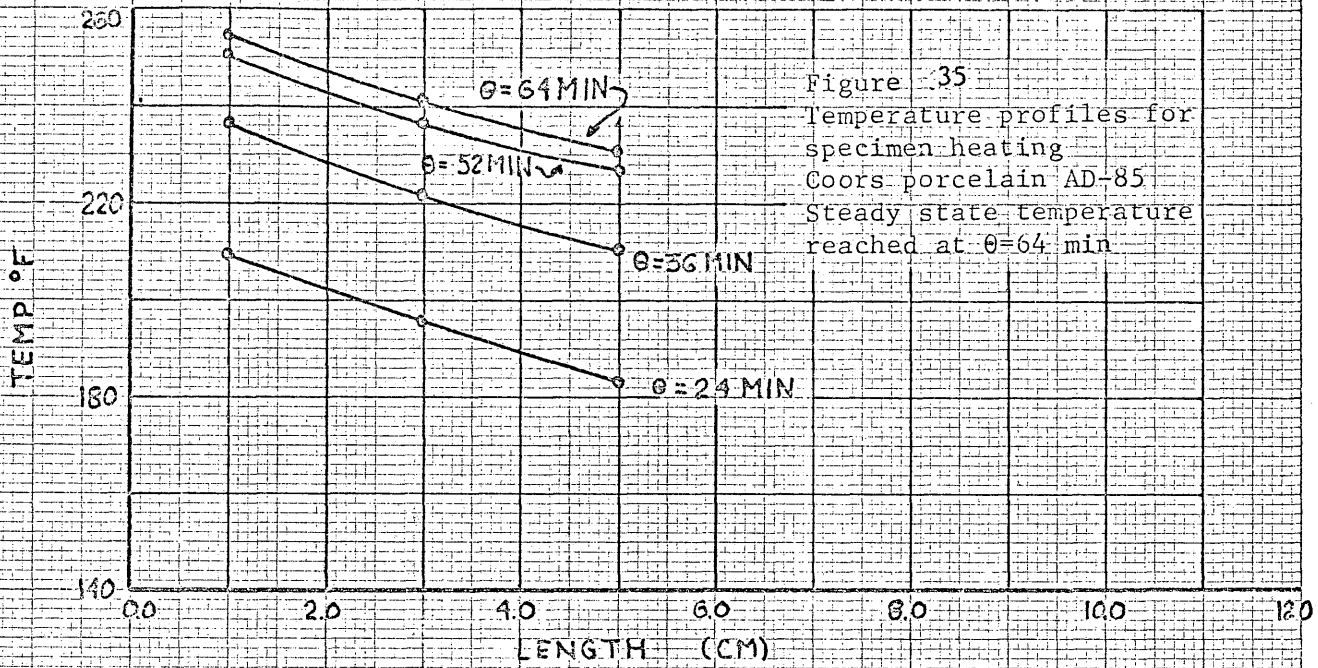
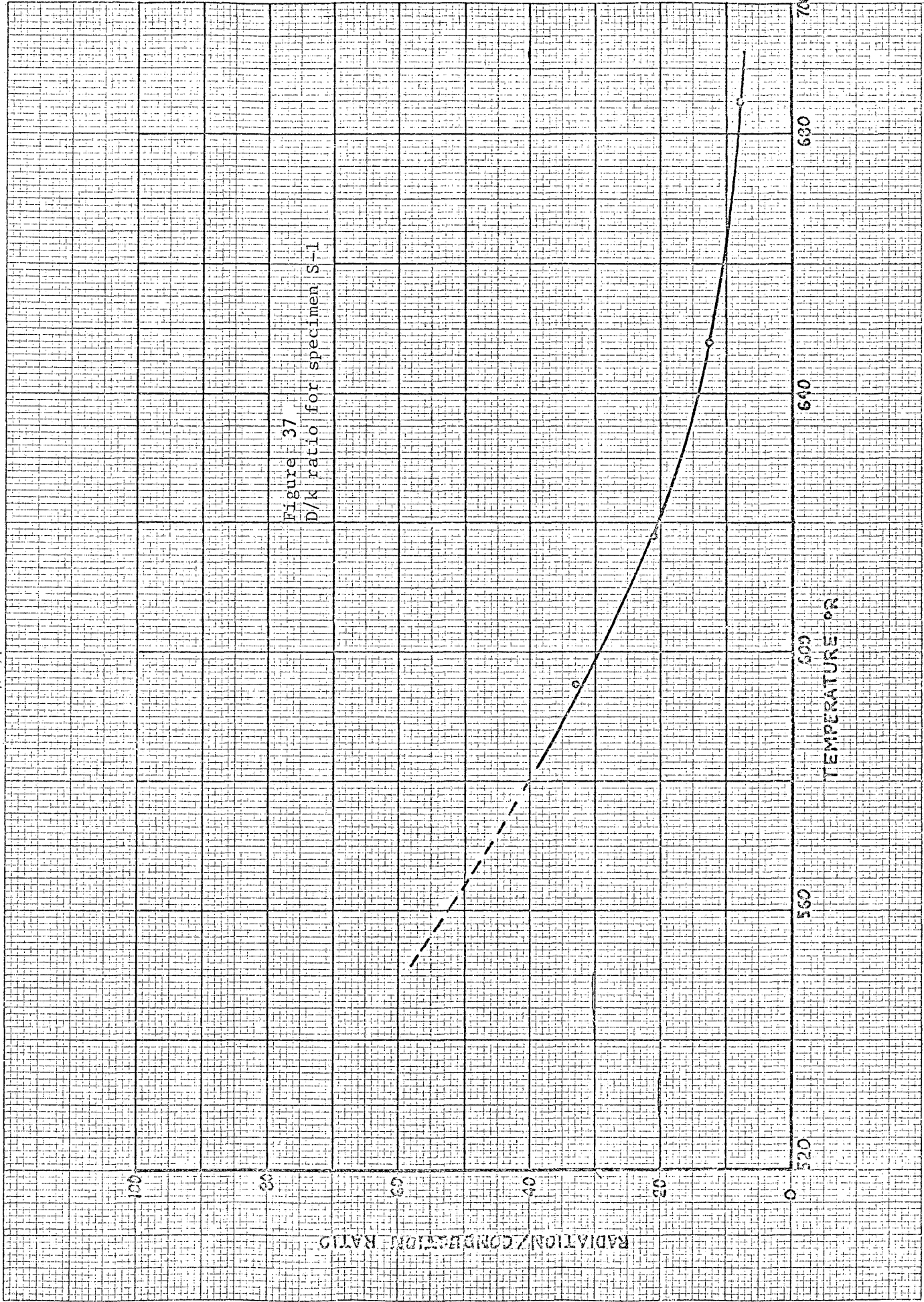
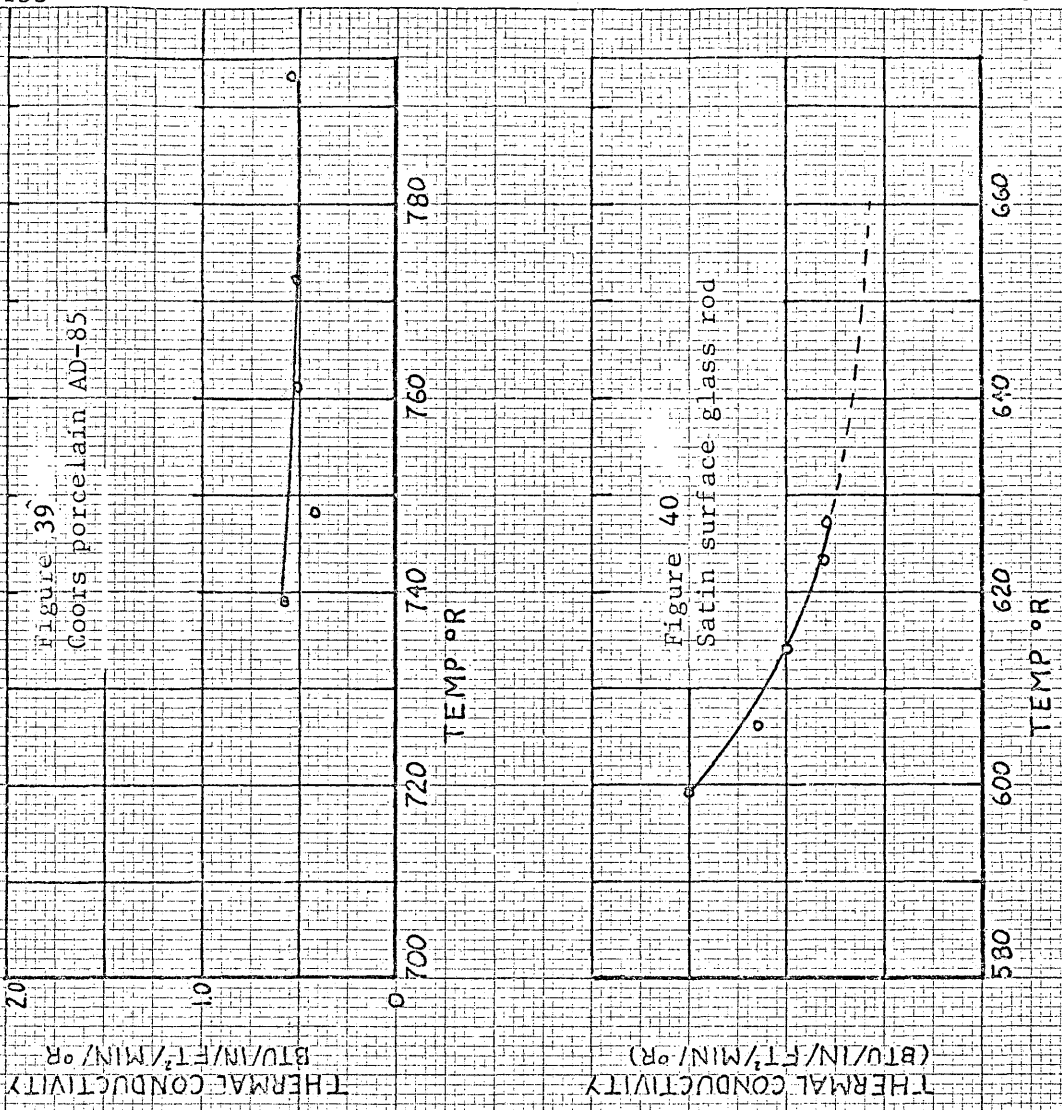
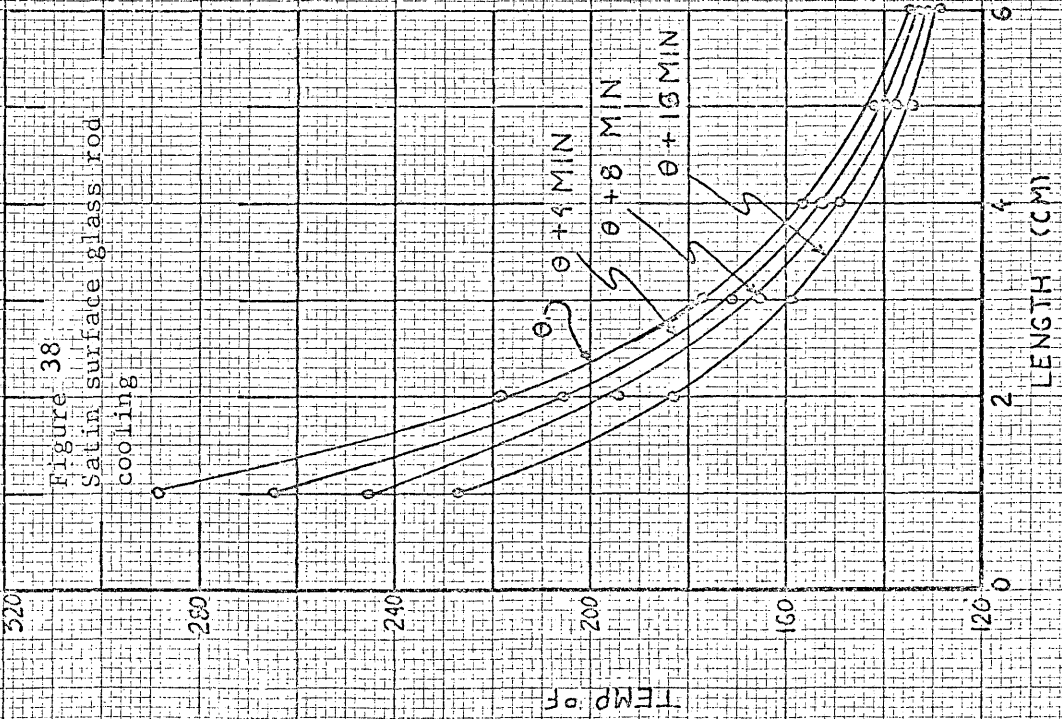
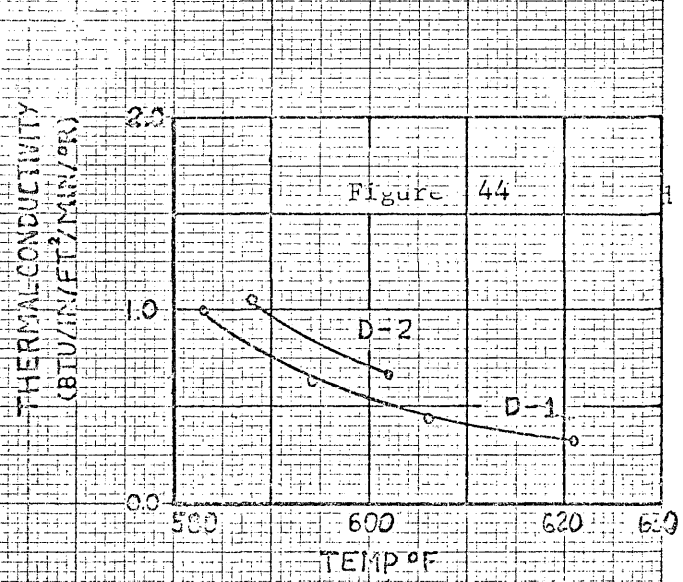
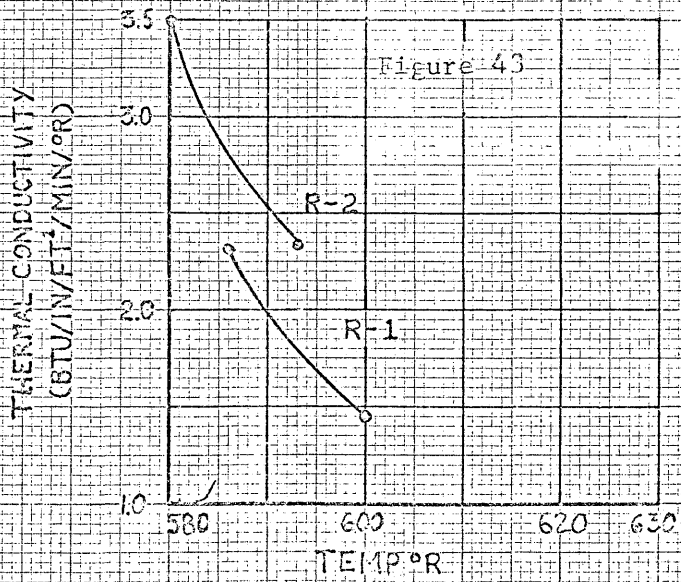
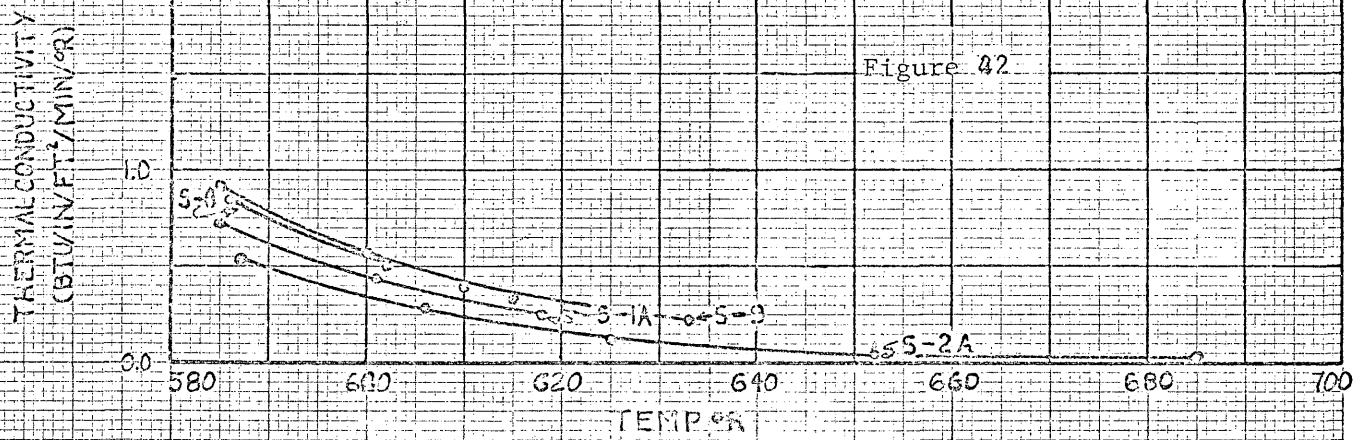
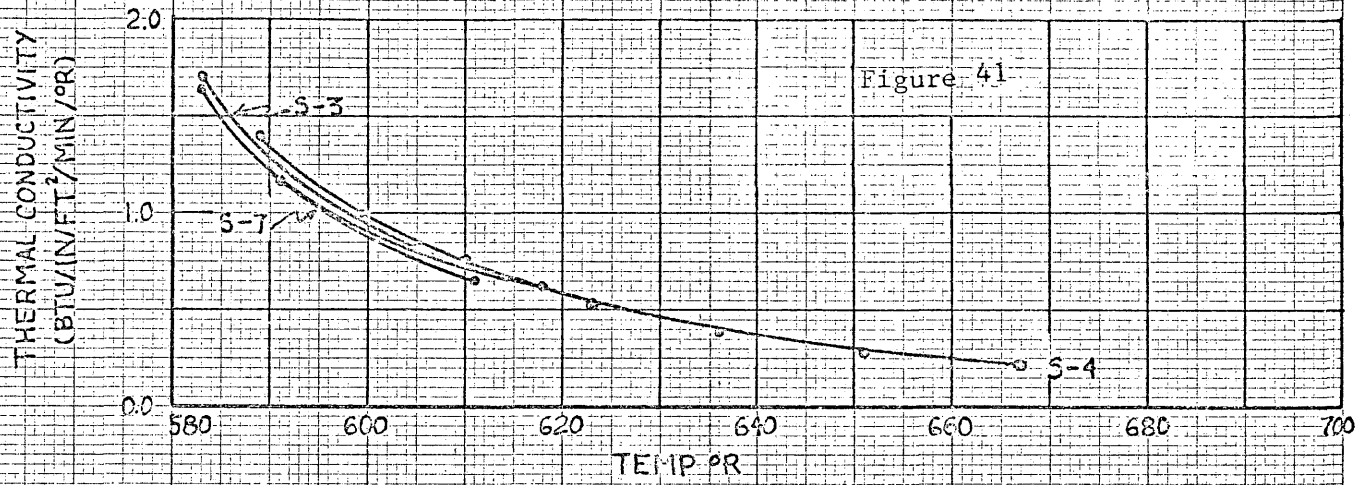


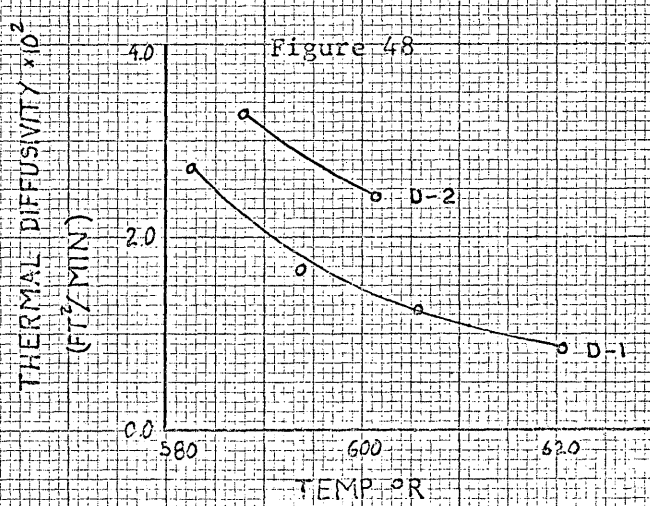
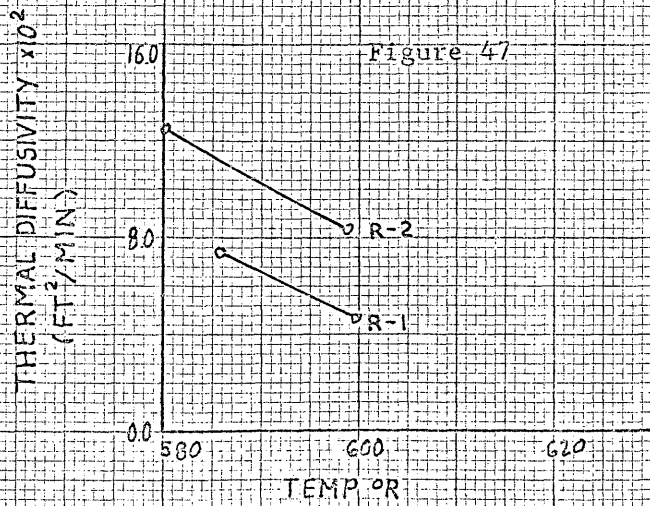
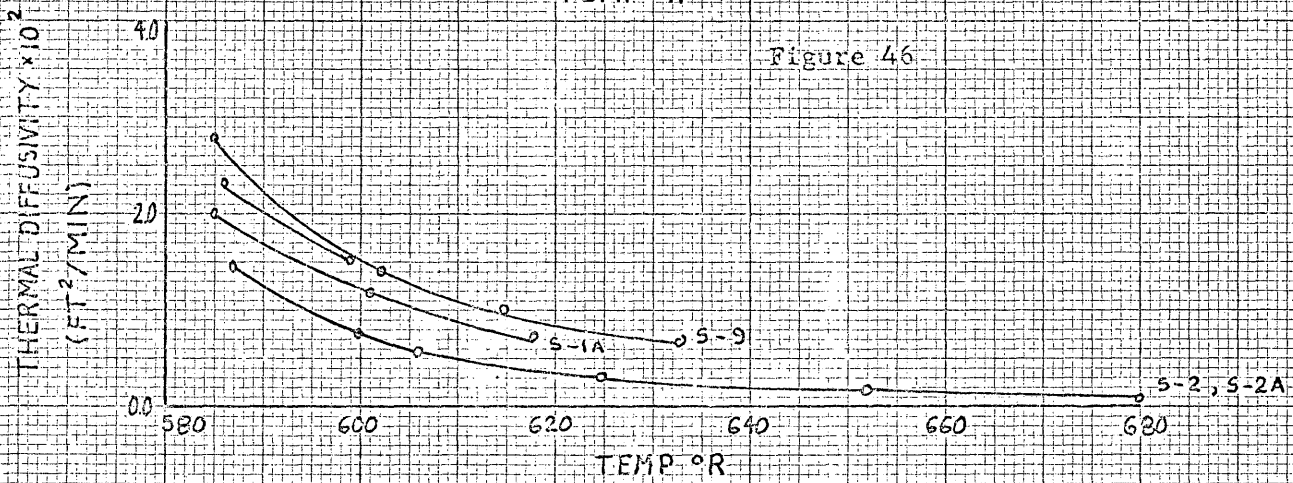
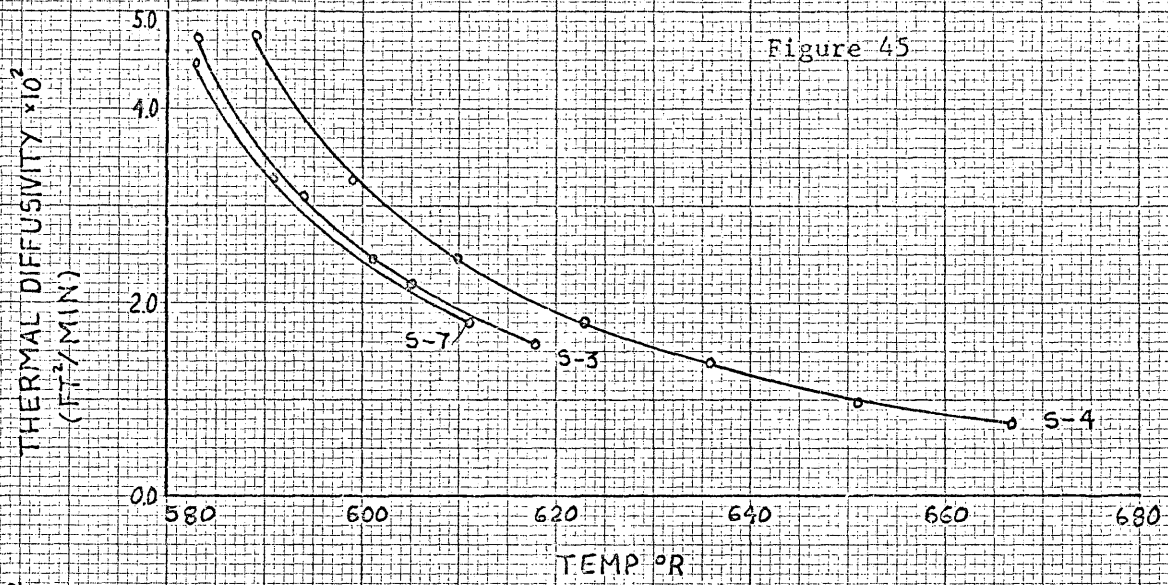
Figure 37  
D/K ratio for specimen S-1

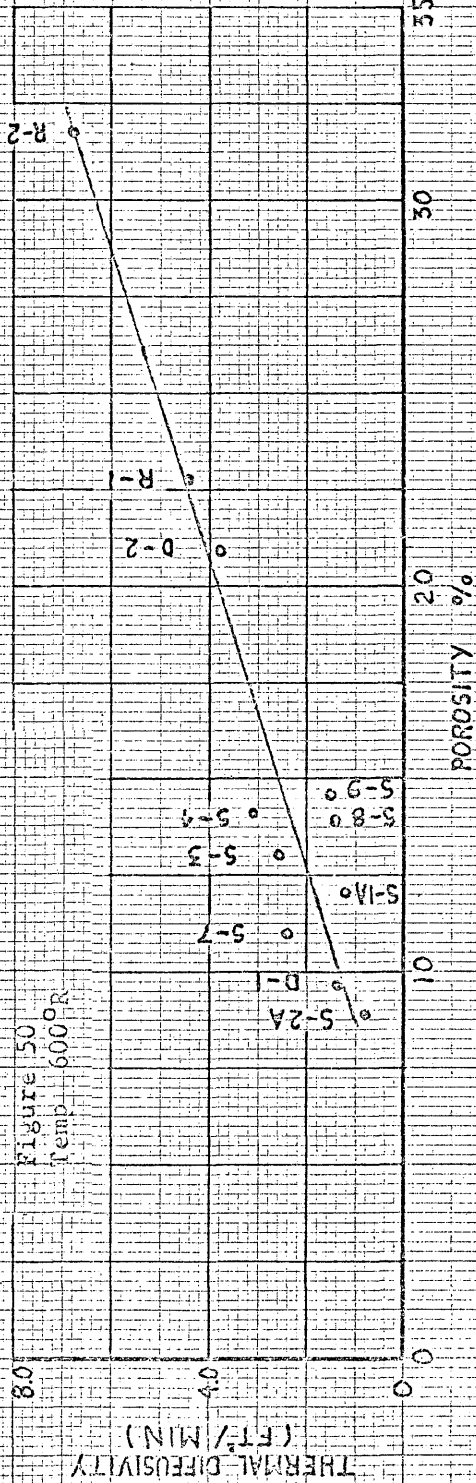
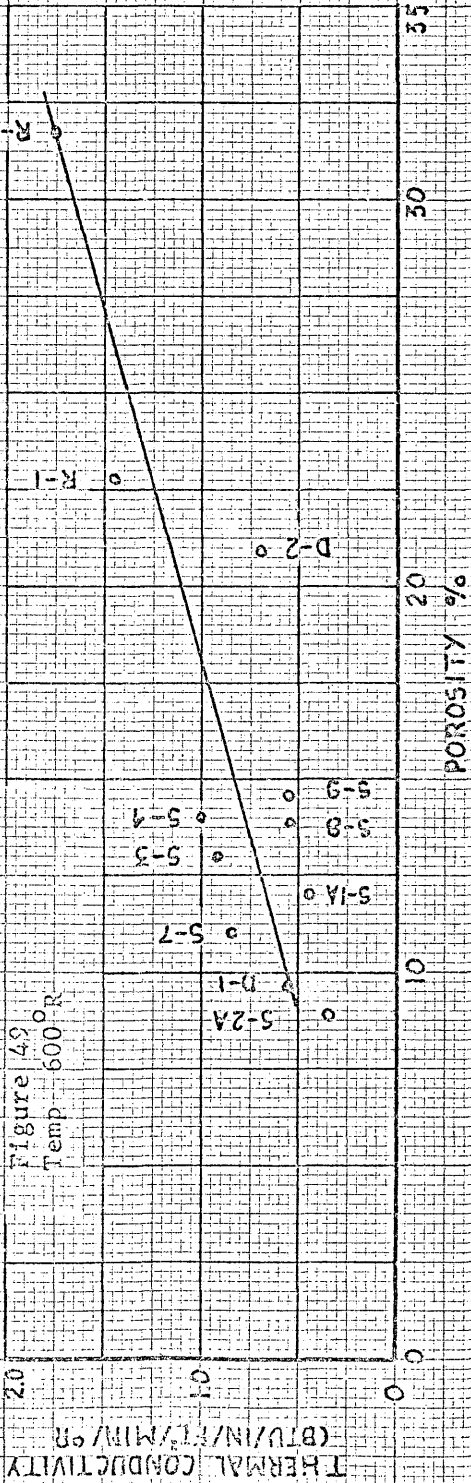


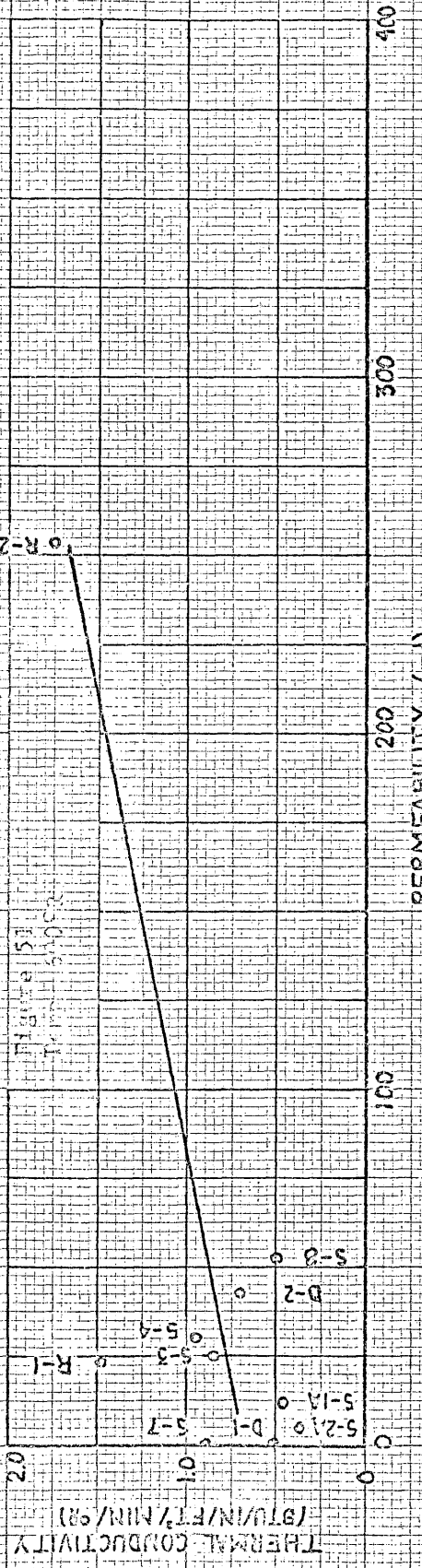












PERMEABILITY (md)

250  
200  
150  
100  
50  
0

FIGURE 53

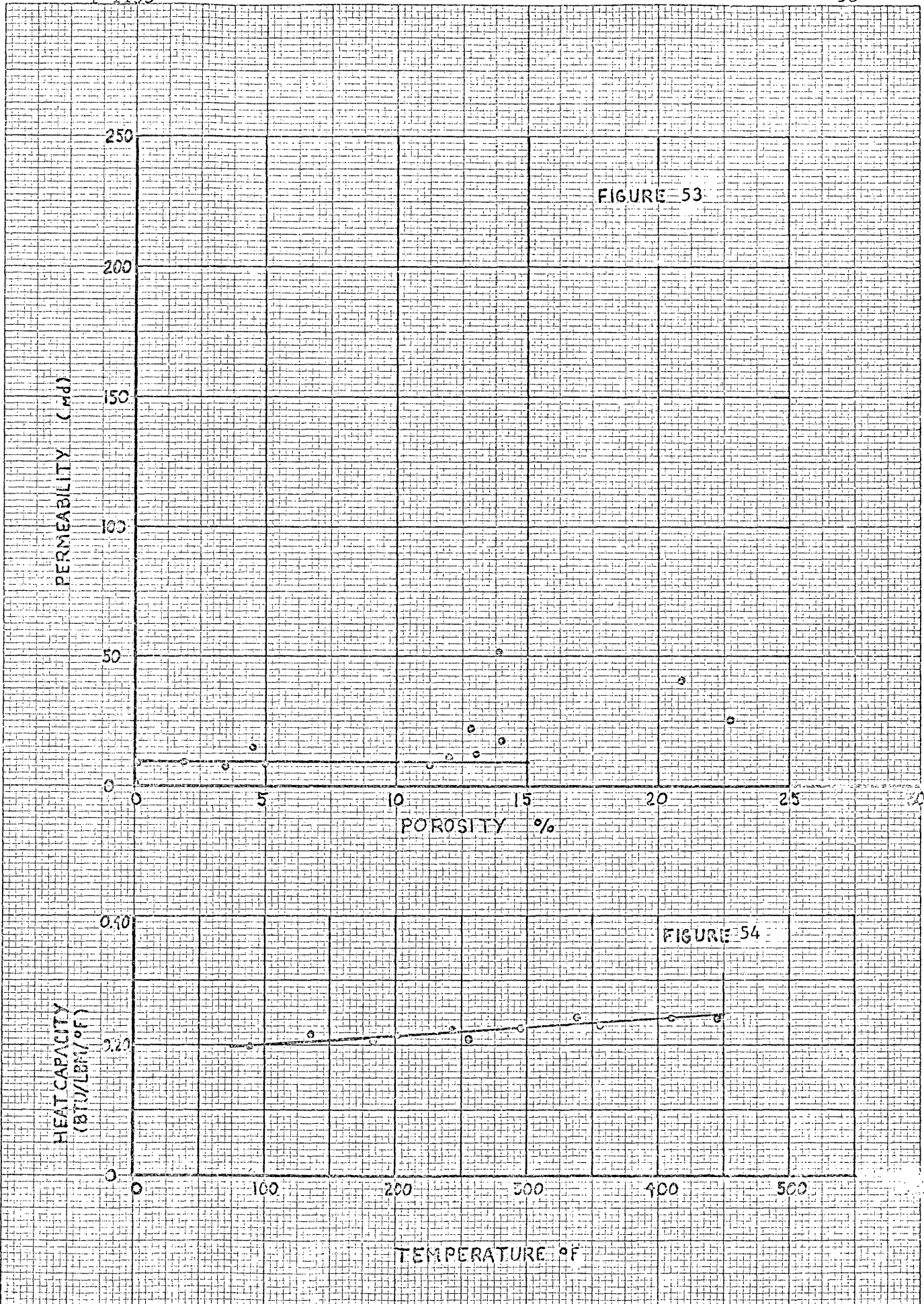
POROSITY %

HEAT CAPACITY (BTU/LBM/°F)

0.40  
0.20  
0

FIGURE 54

TEMPERATURE °F



GEOLOGICAL DESCRIPTION OF SPECIMENS

<u>Specimen</u>	<u>Permeability (to air)</u>	<u>Porosity (effective)</u>	<u>Description</u>
D-1	0.2 md	9.59%	Sandstone; fine grain; well consolidated; well sorted, subangular to rounded quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica (biotite and muscovite).
D-2	41.5 md	20.8%	Sandstone; fine to medium grain; slightly friable; well sorted, subangular to rounded quartz grains cemented with montmorillonite and calcite; medium gray with hornblende and mica.
E-1	7.57 md	7.57%	Sandstone; medium grain; slightly friable; well sorted, subangular quartz grains cemented with montmorillonite and calcite; medium gray with hornblende and mica.
E-2	3.9 md	9.82%	Same as E-1
E-3	1.9 md	7.65%	Sandstone; medium to fine grain; well consolidated; well sorted, subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
E-5	1.7 md	11.3%	Sandstone; fine to medium grain; well consolidated; well sorted, rounded to subangular quartz grains cemented with calcite and montmorillonite. Medium gray with hornblende and mica.

<u>Specimen</u>	<u>Permeability (to air)</u>	<u>Porosity (effective)</u>	<u>Description</u>
E-6	22.2 md	12.8%	Sandstone; medium to large grain; well consolidated; well sorted, subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
E-8	0.5 md	9.49%	Sandstone; fine grain; well consolidated; well sorted, subangular to rounded quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
O-1	96.1 md	18.5%	Sandstone; medium to large grain; friable; well sorted, subangular quartz grains cemented with montmorillonite and calcite. Some secondary cementation with calcite and dolomite; medium gray with hornblende and mica.
R-1	25.8 md	22.7%	Sandstone; medium to fine grain; unconsolidated; well sorted, subangular quartz grains cemented with montmorillonite and calcite. Slightly laminated; medium gray with hornblende and mica. Laminating material is primarily hornblende and partially altered hornblende.
R-2	253.0 md	31.1%	Same as R-1
S-1	11.3 md	12.0%	Sandstone; medium grain; slightly friable; well sorted, subangular quartz grains cemented with montmorillonite and clacite; medium gray with horn blende and mica.

<u>Specimen</u>	<u>Permeability (to air)</u>	<u>Porosity (effective)</u>	<u>Description</u>
S-1A	11.3 md	12.0%	Same as S-1
S-2	5.0 md	8.9%	Sandstone; medium grain, friable; well sorted, subangular quartz grains cemented with montmorillonite and calcite; medium gray with hornblende and mica.
S-2A	5.0 md	8.9%	Same as S-2
S-3	12.5 md	13.0%	Sandstone; medium grain; well consolidated well sorted, subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
S-4	15.0 md	14.1%	Sandstone; medium grain; well consolidated; well sorted, subangular to rounded quartz grains cemented with montmorillonite and calcite; medium gray with hornblende and mica.
S-5A	7.0 md	10.8%	Same as S-5
S-6	82.6 md	14.5%	Sandstone; medium to fine grain; slightly friable; well sorted, subangular quartz grains cemented with montmorillonite and calcite; medium gray with hornblende and mica.
S-7	0.1 md	11%	Sandstone; fine to medium grain; well consolidated; well sorted, subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.

<u>Specimen</u>	<u>Permeability (to air)</u>	<u>Porosity (effective)</u>	<u>Description</u>
S-8	52.6 md	13.9%	Sandstone; medium grain; slightly consolidated; well sorted, subangular quartz grains cemented with montmorillonite; medium gray with hornblende and mica.
S-9	4.5 md	14.6%	Sandstone; fine grain; well consolidated; well sorted, rounded to subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
S-10	2.7 md	12.8%	Sandstone; medium to fine grain well consolidated; well sorted, subangular quartz grains cemented with calcite and montmorillonite; medium gray with hornblende and mica.
S-11	0.2 md	5.5%	Sandstone; fine grain; well consolidated; well sorted, subangular to rounded quartz grains cemented with calcite and clay; medium gray with hornblende and mica.



TABLE 2  
 STEADY-STATE TEMPERATURE EQUATIONS AND  
 DERIVATIVE AS A FUNCTION OF LENGTH

<u>Specimen</u>	<u>Current Input (Amps)</u>	<u>Equation and Derivative</u>
D-1	1.0	$T = 186.0 - 28.84X + 2.36X^2$ $dt/dx = - 28.84 + 4.72X$
	2.0	$T = 395.4 - 88.39X + 8.28X^2$ $dt/dx = - 88.39 + 16.56X$
D-2	1.0	$T = 165.4 - 27.01X + 1.9X^2$ $dt/dx = - 27.01 + 3.8X$
	2.0	$T = 297.7 - 57.7X + 3.85X^2$ $dt/dx = - 57.7 + 7.7X$
E-1	1.0	$T = 175.0 - 27.26X + 2.14X^2$ $dt/dx = - 27.26 + 4.28X$
	3.0	$T = 563.4 - 210.3X + 24.9X^2$ $dt/dx = - 210.3 + 49.8X$
E-2	1.0	$T = 183.4 - 30.69X + 2.29X^2$ $dt/dx = - 30.69 + 4.58X$
	2.0	$T = 401.0 - 96.78X + 8.8X^2$ $dt/dx = - 97.78 + 16.2X$
E-3	1.0	$T = 205.0 - 39.29X + 3.23X^2$ $dt/dx = - 39.29 + 6.46X$
	2.0	$T = 437.2 - 106.6X + 8.7X^2$ $dt/dx = - 106.6 + 17.4X$
	3.0	$T = 624.5 - 159.4X + 13.2X^2$ $dt/dx = - 159.4 + 26.4X$

TABLE 2 (Continued)

<u>Specimen</u>	<u>Current Input (Amps)</u>	<u>Equations and Derivative</u>
E-5	1.0	$T = 184.86 - 24.88X + 1.55X^2$ $dt/dx = - 24.88 + 3.1X$
	2.0	$T = 419.0 - 90.92X + 7.08X^2$ $dt/dx = - 90.92 + 14.16X$
	3.0	$T = 624.3 - 136.5X + 10.5X^2$ $dt/dx = - 136.5 + 21.0X$
E-6	1.0	$T = 182.06 - 26.06X + 2.07X^2$ $dt/dx = - 26.06 + 4.14X$
E-8	1.0	$T = 161.6 - 20.17X + 1.28X^2$ $dt/dx = - 20.17 + 2.56X$
	2.0	$T = 365.86 - 68.59X + 4.98X^2$ $dt/dx = - 68.59 + 9.96X$
	3.0	$T = 530.7 - 114.4X + 8.9X^2$ $dt/dx = - 114.4 + 17.8X$
O-1	1.0	$T = 129.8 - 11.76X + .6X^2$ $dt/dx = - 11.76 + 1.2X$
	2.0	$T = 305.8 - 58.47X + 3.9X^2$ $dt/dx = - 58.47 + 7.8X$
	3.0	$T = 460.7 - 104.36X + 8.1X^2$ $dt/dx = - 104.36 + 16.2X$
R-1	1.0	$T = 165.4 - 28.45X + 2.29X^2$ $dt/dx = - 28.45 + 4.58X$
	2.0	$T = 429.3 - 190.0X + 40.87X^2$ $dt/dx = - 190.0 + 82.7X$
R-2	1.0	$T = 151.7 - 24.11X + 1.95X^2$ $dt/dx = - 24.11 + 3.9X$

TABLE 2 (Continued)

<u>Specimen</u>	<u>Current Input (Amps)</u>	<u>Equations and Derivative</u>
S-1	2.0	$T = 352.1 - 92.9X + 8.7X^2$ $dt/dx = - 92.9 + 17.4X$
	1.0	$T = 291.2 - 78.7X + 8.36X^2$ $dt/dx = - 78.7 + 16.72X$
S-1A	2.0	$T = 403.4 - 107.1X + 10.75X^2$ $dt/dx = - 107.1 + 21.5X$
	1.0	$T = 188.9 - 35.38X + 3.02X^2$ $dt/dx = - 35.38 + 6.04X$
S-2A	2.0	$T = 365.1 - 83.9X + 7.45X^2$ $dt/dx = - 83.9 + 14.9X$
	1.0	$T = 178.6 - 38.3X + 3.7X^2$ $dt/dx = - 38.3 + 7.4X$
S-3	2.0	$T = 327.6 - 74.8X + 6.04X^2$ $dt/dx = - 74.8 + 6.04X$
	3.0	$T = 450.0 - 105.8X + 8.5X^2$ $dt/dx = - 105.8 + 17.0X$
	1.0	$T = 179.2 - 23.6X + 1.43X^2$ $dt/dx = - 23.6 + 2.76X$
S-4	2.0	$T = 315.0 - 58.8X + 3.97X^2$ $dt/dx = - 58.8 + 7.94X$
	3.0	$T = 479.7 - 94.1X + 6.4X^2$ $dt/dx = - 94.1 + 12.8X$
S-5	1.0	$T = 232.6 - 28.4X + 1.4X^2$ $dt/dx = - 28.4 + 2.8X$
S-5	1.0	$T = 146.5 - 19.5X + 1.3X^2$ $dt/dx = - 19.5 + 2.6X$

TABLE 2 (Continued)

<u>Specimen</u>	<u>Current Input (Amps)</u>	<u>Equations and Derivative</u>
	2.0	$T = 305.6 - 61.6X + 4.5X^2$ $dt/dx = - 61.6 + 9.0X$
S-6	1.0	$T = 147.7 - 19.7X + 1.4X^2$ $dt/dx = - 19.7 + 2.8X$
S-	2.0	$T = 260.6 - 43.1X + 2.6X^2$ $dt/dx = - 43.1 + 5.2X$
	3.0	$T = 299.7 - 51.1X + 3.1X^2$ $dt/dx = - 51.1 + 6.2X$
S-7	1.0	$T = 169.0 - 20.2X + 1.41X^2$ $dt/dx = - 20.2 + 2.82X$
	3.0	$T = 379.2 - 72.4X + 5.8X^2$ $dt/dx = - 72.4 + 11.6X$
S-8	1.0	$T = 168.8 - 31.1X + 2.6X^2$ $dt/dx = - 31.1 + 5.2X$
	2.0	$T = 298.4 - 56.6X + 3.8X^2$ $dt/dx = - 56.6 + 7.6X$
	3.0	$T = 440.8 - 87.1X + 5.9X^2$ $dt/dx = - 87.1 + 11.8X$
S-9	1.0	$T = 204.0 - 35.1X + 2.6X^2$ $dt/dx = - 35.1 + 5.2X$
	2.0	$T = 416.0 - 90.4X + 7.2X^2$ $dt/dx = - 90.4 + 14.4X$
	3.0	$T = 592.3 - 139.1X + 11.3X^2$ $dt/dx = - 139.1 + 22.6X$
S-10	1.0	$T = 138.8 - 13.6X + .7X^2$ $dt/dx = - 13.6 + 1.4X$
	2.0	$T = 312.0 - 50.7X + 3.1X^2$ $dt/dx = - 50.7 + 6.2X$

TABLE 2 (Continued)

<u>Specimen</u>	<u>Current Input (Amps)</u>	<u>Equation and Derivative</u>
	3.0	$T = 449.5 - 74.2X + 4.4X^2$ $dt/dx = - 74.2 + 8.8X$
S-11	1.0	$T = 196.0 - 12.7X + .75X^2$ $dt/dx = - 12.7 + 1.5X$
	2.0	$T = 399.0 - 87.3X + 8.8X^2$ $dt/dx = - 87.3 + 17.6X$
	3.0	$T = 670.7 - 185.3X + 21.2X^2$ $dt/dx = - 185.3 + 42.4X$

TABLE 2A

STEADY-STATE AND COOLING EQUATIONS  
 PERTAINING TO STANDARD MATERIALS  
 Coors Porcelain AD-85

<u>Length</u>	<u>Equation</u>	<u>Remarks</u>
10.0 cm	$T = 234.2 - 10.6X + .43X^2$ $dt/dx = - 10.6 + .86X$	Steady State
"	$T = 366.9 - 16.9X + .8X^2$ $dt/dx = - 16.9 + 1.6X$	Cooling, $\theta_1$
"	$T = 339.1 - 12.5X + .6X^2$ $dt/dx = - 12.5 + 1.2X$	Cooling, $\theta_1 + 2$ min
8.0 cm	$T = 281.1 - 16.4X + 1.X^2$ $dt/dx = - 16.4 + 2.X$	Steady State
"	$T = 253.0 - 4.6X + .1X^2$ $dt/dx = - 4.6 + .2X$	Cooling, $\theta_1$
"	$T = 241.0 - 2.75X + .1X^2$ $dt/dx = - 2.75 + .2X$	Cooling, $\theta_1 + 1.5$ min
6.0 cm	$T = 216.6 - 6.5X + .12 X^2$ $dt/dx = - 6.5 + .24X$	Heating, $\theta = 24$ min
"	$T = 244.5 - 7.5X$	Heating, $\theta = 36$ min
"	$T = 256.9 - 5.5X + .4X^2$	Heating, $\theta = 52$ min

TABLE 2A (Continued)  
SATIN SURFACE QUARTZ ROD

<u>Length</u>	<u>Equation</u>	<u>Remarks</u>
10 cm	$T = 444.6 - 70.4X + 3.8X^2$ $dt/dx = - 70.4 + 7.6X$	Steady State
"	$T = 229.4 - 24.23X + 1.24X^2$ $dt/dx = - 24.23 + 2.48X$	Cooling, $\theta_1$
"	$T = 201.4 - 17.2X + .8X^2$ $dt/dx = - 17.2 + 1.6X$	Cooling, $\theta_1 + 5$ min
"	$T = 180.25 - 12.8X + .6X^2$	Cooling, $\theta_1 + 9$ min
8 cm	$T = 311.6 - 87.2X + 9.4X^2$ $dt/dx = - 87.2 + 18.8X$	Steady State
6 cm	$T = 277.0 - 39.5X + 2.7X^2$ $dt/dx = - 39.5 + 5.4X$	Steady State

COOLING CURVE EQUATIONS FOR SPECIMEN S-1

<u>Time</u>	<u>Equation</u>
$\theta_1$	$T = 355.5 - 79.7X + 7.3X^2$ $dt/dx = - 79.7 + 14.6X$
$\theta_1 + 4$ min	$T = 320.7 - 65.9X + 5.8X^2$ $dt/dx = - 65.9 + 11.6X$
$\theta_1 + 8$ min	$T = 294.0 - 57.9X + 5.3X^2$ $dt/dx = - 57.9 + 10.6X$
$\theta_1 + 12$ min	$T = 265.6 - 46.5X + 4.0X^2$ $dt/dx = - 46.5 + 8.0X$

TABLE 3

## PHYSICAL AND THERMAL PROPERTIES OF SPECIMENS

<u>Specimen</u>	<u>Porosity</u> %	<u>Permeability</u> md	<u>Density</u> lb/ft <sup>3</sup>	<u>Temperature</u> OR	<u>Thermal</u> <u>Conductivity</u>	<u>Thermal</u> <u>Diffusivity</u>
D-1	9.6	0.2	150.07	621	.316	.85x10 <sup>-2</sup>
				606	.460	1.23x10 <sup>-2</sup>
				594	.617	1.65x10 <sup>-2</sup>
				583	1.010	1.70x10 <sup>-2</sup>
D-2	20.9	41.5	131.5	602	.670	2.4 x10 <sup>-2</sup>
				588	1.06	3.22x10 <sup>-2</sup>
E-1	7.57	3.5	153.4	611	.384	1.04x10 <sup>-2</sup>
				597	.672	1.76x10 <sup>-2</sup>
				585	1.01	2.63x10 <sup>-2</sup>
E-2	9.82	1.9	149.8	617	.366	.98x10 <sup>-2</sup>
				601	.568	1.50x10 <sup>-2</sup>
				587	.920	1.46x10 <sup>-2</sup>
E-5	11.3	1.7	147.2	623	.465	1.27x10 <sup>-2</sup>



TABLE 3 (Continued)

<u>Specimen</u>	<u>Porosity %</u>	<u>Permeability md</u>	<u>Density lb/ft<sup>3</sup></u>	<u>Temperature °R</u>	<u>Thermal Conductivity</u>	<u>Thermal Diffusivity</u>
E-5 (Continued)				609	.650	1.178x10 <sup>-2</sup>
				597	.920	1.58 x10 <sup>-2</sup>
				586	1.36	3.714x10 <sup>-2</sup>
E-6	12.8	22.2	144.7	619	.375	1.0 x10 <sup>-2</sup>
				606	.532	1.47 x10 <sup>-2</sup>
				595	.757	2.10 x10 <sup>-2</sup>
R-1	22.7	25.8	128.3	600	1.45	4.55 x10 <sup>-2</sup>
				586	2.33	7.29 x10 <sup>-2</sup>
R-2	31.7	253.0	114.4	591	2.34	8.20 x10 <sup>-2</sup>
				578	3.70	12.90 x10 <sup>-2</sup>
S-1A	12.0	11.3	146.0	618	.257	.71 x10 <sup>-2</sup>
				601	.432	1.18 x10 <sup>-2</sup>
				585	.720	1.98 x10 <sup>-2</sup>
S-2	8.9	5.0	151.2	723	0.031	0.82 x10 <sup>-3</sup>
				685	0.036	0.96 x10 <sup>-3</sup>

TABLE 3 (Continued)

<u>Specimen</u>	<u>Porosity</u> <u>%</u>	<u>Permeability</u> <u>md</u>	<u>Density</u> <u>lb/ft<sup>3</sup></u>	<u>Temperature</u> <u>°R</u>	<u>Thermal</u> <u>Conductivity</u>	<u>Thermal</u> <u>Diffusivity</u>
				652	0.0615	$1.63 \times 10^{-3}$
S-2A	8.9	5.0	151.2	625	0.113	$3.00 \times 10^{-3}$
				606	0.216	$5.73 \times 10^{-3}$
				600	0.296	$7.86 \times 10^{-3}$
				587	0.542	$14.3 \times 10^{-3}$
S-3	13.0	12.5	144.4	618	0.545	$1.52 \times 10^{-2}$
				605	0.780	$2.17 \times 10^{-2}$
				594	1.100	$3.06 \times 10^{-2}$
				583	1.690	$4.70 \times 10^{-2}$
S-4	14.1	15.0		667	0.220	$.74 \times 10^{-2}$
				651	0.270	
				636	0.382	
				623	0.528	
				610	.72	
				599	.967	

TABLE 3 (Continued)

<u>Specimen</u>	<u>Porosity %</u>	<u>Permeability md</u>	<u>Density lb/ft<sup>3</sup></u>	<u>Temperature OR</u>	<u>Thermal Conductivity</u>	<u>Thermal Diffusivity</u>
S-4 (Continued)				589	1.39	
S-6	14.5	82.6	141.9	590	1.26	3.57x10 <sup>-2</sup>
				580	1.86	5.25x10 <sup>-2</sup>
S-7	11.0	0.1	147.7	611	.66	1.79x10 <sup>-2</sup>
				601	.894	2.43x10 <sup>-2</sup>
				591	1.19	3.24x10 <sup>-2</sup>
				583	1.64	4.464x10 <sup>-2</sup>
S-8	13.9	52.6	146.2	602	.50	1.38x10 <sup>-2</sup>
				586	.83	2.284x10 <sup>-2</sup>
S-9	14.6	4.5	135.8	633	.227	.67x10 <sup>-2</sup>
				615	.338	1.00x10 <sup>-2</sup>
				599	.538	1.50x10 <sup>-2</sup>
				585	.886	2.75x10 <sup>-2</sup>

## IBM OS/360 BASIC FORTRAN IV(E) COMPILATION

```

3  FORMAT (3F10.4)
4  FORMAT (12)
5  FORMAT (215)
10 READ (1, 4) IP
    IF (IP-99) 11, 12, 12
11 READ (1, 5) N, NP
    READ (1, 3) (C1, C2, C3, 1=1, N)
    CALL FIT (NP, C1, C2, C3, N)
    GO TO 10
12 STOP
    END

```

END OF COMPILATION MAIN

## IBM OS/360 BASIC FORTRAN IV(E) COMPILATION

```

C  SUBROUTINE FIT (IP, C1, C2, C3, N)
    THIS SUBROUTINE CALCULATED THE RADIATION-CONDUCTION
    RATIO (D/k=RCTA) DOUBLE PRECISION T1(20), TA (20),
    TAA (20), TSA
    DIMENSION DT1 (20), DT2 (20), RCTA (20)
    WRITE (3, 3)
3  FORMAT ('11')
    D=1.0
    TS=565.0
    DX=0.25
    ACS=(3.14*(D**2.0))/(4.0*144.0)
    AR=3.14*D*DX/144.0
    X1=.25
    DO 1 I=1, IP
    X2=X1/DX
    A=X1*2.54
    B=X2*2.54
    T1(I)=C1/C2*A/C3*A**2.
    T2(I)=C1/C2*B/C3*B**2.
    DT1(I)=(C2/2*C3*A)*2.54
    DT2(I)=(C2/2*C3*B)*2.54

```

```

TA(I)=( (T1(I)/T2(I))/2.)/460
TSA=ts**4.0
TAA(I)=(((T1(I)/460.0)**4.0/(T2(I)/460.0)**4.0)/2. )**0.25
C=ACS*(DT2(I)-DT1(I))
F=2.856E-11*AR*(TAA(I)**4.0-TSA)
RCTA(I)=C/F
TA(I)=TA(I)-460.
X1=X2
WRITE (3, 2) TA(I), RCTA(I), DT1(I), DT2(I), TAA(I)
1 CONTINUE
2 FORMAT (5X, 4HTAVG, F7.2, 2X, 4HRCTA, 2X, 1HK, E20.8,
3F10.5)
RETURN
END

```

END OF COMPILATION FIT

IBM OS/360 BASIC FORTRAN IV(E) COMPILATION

```

PROGRAM II
C THERMAL CONDUCTIVITY COEFE CALCULATIONS
C T1 IS TEMP AT X1 FOR STEADY STATE CONDITIONS
C T2 IS SAME AS T1 BUT FOR X2
C T1D IS THE DERIVATIVE OF T1 AT X1
C T2D IS SAME AS T1D BUT AT X2
C F(TAAC) IS D/k=F(TAA) FROM STEADY STATE CURVE
C A IS TAA BETWEEN X1 AND X2 FOR TIME 1
C B IS SAME AS A BUT FOR TIME 2
C TAAC IS RADIATION SURFACE TEMPERATURE
C TA1 IS AVE TEMP BETWEEN X1 AND X2 AT TIME 1
C TA2 IS AVE TEMP BETWEEN X1 AND X2 AT TIME 2
C DOUBLE PRECISION A, XKD, B, TS, TA1, XK, ADV
C DOUBLE PRECISION TA2, XKN, TAAC
C DIMENSION HEAD (20)
T1(X)=C1/C2*X/C3*X*X/C4*X**3./C5*X**4./460.
T2(X)=D1/D2*X/D3*X*X/D4*X**3./D5*X**4./460.
T1D(X)=(C2/2.*C3*X/3.*C4*X*X/4.*C5*X**3.)*2.54
T2D(X)=(D2/2.*D3*X/3.*D4*X*X/4.*D5*X**3.)*2.54
F(TAAC)=F1/F2*TAAC/F3*TAAC**2./F4*TAAC**3
DX=0.25*2.54
TS=565
READ(1, 2) HEAD

```

```

1  FORMAT (3F5.0,12,F10.0)
   DEN=DEN*2.54
   READ (1, 3) F1, F2, F3, F4
3  FORMAT (3F10.4)
   WRITE (3, 4) HEAD
4  FORMAT (10I, 20A4)
   WRITE (3, 5) XMASS, CP, DZ
5  FORMAT ('MASS=', F5.4, 2X, 'CP=', F4.3, 'DZ-', F4.2)
   N=N-1
   READ (1, 7) TH1, C1, C2, C3, C4, C5
   DO 30 I=1, N
   WRITE (3,200)
200 FORMAT (//)
   READ (1, 7) TH2, D1, D2, D3, D4, D5
   7  FORMAT (6F10.0)
   XL=
100 XL=XL/DX
   IF9DEN-XL) 6, 8, 8
   8  A=((T1(XL)**4/T1(XL/DX)**4)/2. )**0.25
   B=((T2(XL)**4/T2(XL/DX)**4)/2. )**0.25
   TAAC=((A**4./B**4.)/2. )**0.25
   TA1=(T1(XL)/T1(XL/DX))/2.
   TA2=(T2(XL)/T2(XL/DX))/2.
   XKN=XMASS*CP*(TA2-TA1)/(TH2-TH1)
   XKD=((T1D(XL/DX)/T2D(XL/DX))/2.-(T1D(XL)/T2D(XL))/2.*5.45E-
1-F(TAAC)*2.856E-11*.00545*(TAAC**4-TS**4
   XK=XKD
   ADV=(TA1/TA2)/2.
   WRITE(93, 20) XK, ADV, XL
20  FORMAT (3E20.8)
   GO TO 100
   6  C1=D1
   C2=D2
   C3=D3
   C4=D4
   C5=D5
   TH1=TH2
30  CONTINUE
   STOP
   END

```

END OF COMPILATION MAIN

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